

**Building 147 – Replace Chiller #3
Alexandria VA Medical Center
Pineville, LA**



Bid Documents

Design Analysis

**November 09, 2022
Project Number: 502-21-110**





Bid Documents Submittal

9 September 2022

Building 147 – Replace Chiller #3 at the Alexandria VAMC Pineville, LA

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Building 147 – Replace Chiller #3, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

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I. Executive Summary

I. Executive Summary

A. Goals and Needs:

1. The primary goal is to complete replacement of Chiller #3, located in the Central Chiller Plant at the Alexandria VA Health Care System, Pineville, LA. Chiller #3 is currently decommissioned. It is recommended to replace the chilled water pump due to known issues.
2. The main objectives to replace Chiller #3 with a new chiller that will effectively serve VA priorities, including (in this order):
 - a. Range of capacities required to support VA hospital, pharmacy, residential facilities, employee work stations, and supporting facilities.
 - b. Reliability under extreme conditions. These conditions include Central Louisiana's climate, hurricanes and other disasters, and VA emergency response and disaster relief efforts.
 - c. Simplicity of operation.
 - d. Ease of maintenance.
 - e. Smooth integration into existing central chiller plant.
3. Additional scope is to replace deteriorated areas of chilled water piping insulation as well as widen the door opening to accommodate chiller replacement and reinstall existing door and frame with new louver sidelite.

B. Existing Conditions

The existing 1-story structure consists of a main building with one main space and smaller ancillary rooms. Structure of the building consists of a precast concrete double-T's roof slab supported on load bearing CMU/brick veneer exterior walls. Floor is concrete slab-on-grade. Structural calculations of the existing structure are not available.

C. Construction Cost Analysis

1. [REDACTED]
2. [REDACTED] are not

A/E Chiller #3 Replacement, Project No. 502-21-110
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D. Project Schedule



Building 147 – Replace Chiller #3, Project No. 502-21-110
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II. Environmental

(Not Applicable)

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III. Civil
(Not Applicable)

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IV. Structural

IV. Structural

A. Goals and Needs:

1. Structural work on this project includes the design of a new wider door opening in the exterior existing masonry wall in order to better facilitate installation of new chiller equipment. The new design will comply with all state and VA building code requirements.

B. Existing Conditions

1. The existing structure is comprised of precast double tee roof panels supported by 8in masonry walls founded on conventionally reinforced concrete wall foundations. Exact wall reinforcing is not known and will require verification by the contractor during construction.

C. Scope of Demolition:

1. Partial demolition of existing wall to widen the existing opening.

D. Scope of New Work:

1. Install new HSS steel lintel.
2. Install new reinforcing steel in wall.

E. Potential Owner Furnished and Installed Items:

None Anticipated

F. Items Needed From VA

1. None anticipated drawings

G. Codes and Standards:

As Listed in Attachment 1 Design Criteria

H. Applicable Specifications

1. 05 12 00 Structural Steel
2. 01 45 35 Special Inspections

I. Attachments

- 1.Attachment 1: Design Criteria
- 2.Attachment 2: Structural Calculations

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Attachment No. 1
Design Criteria

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1. Codes & Standards:
 - a. International Building Code (IBC) 2018 Edition as modified by:
 - i. Seismic Design Requirements, H-18-8, Department of Veterans Affairs.
 - b. ANSI/ASCE 7-16, Minimum Design Loads for Buildings and Other Structures
 - c. ACI 318-14 Building Code Requirements for Structural Concrete
 - d. AISC 360-16, Specification for Steel Buildings
2. Design Loads
 - a. Dead Loads
 - i. Existing Roof and Existing Floor: Self weight of building materials
 - b. Collateral Loads
 - i. Existing Roof and Existing Floor: 10 psf
 - c. Live Loads
 - i. Roof: 20 psf unreducible
 - ii. Walls: 5 psf minimum
 - d. Snow Loads: 5 psf
 - e. Wind loads
 - i. Ultimate Wind Speed: $V=120$ mph
 - ii. Risk Category: IV
 - iii. Wind Importance Factor: $I_w=1.0$
 - iv. Wind Exposure: B
 - v. Internal Pressure Coefficient:
 - a) ± 0.18 "Enclosed"
 - f. Seismic Loads
 - i. Risk Category: IV
 - ii. Seismic Importance Factor: $I_E=1.5$
 - iii. Mapped Spectral Response Accelerations:
 - a) $S_S=0.111$
 - b) $S_1=0.063$

- iv. Site Class: D
- v. Spectral Response Coefficients:
 - a) $S_{DS}=0.118$
 - b) $S_{D1}=0.101$
- vi. Seismic Design Category: C
- vii. Basic Seismic-Force-Resisting System
 - a) Ordinary Reinforced Masonry Shearwall (Assumed)
- viii. Response Modification Factor(s): $R = 2$
- ix. Analysis Procedure Used: Equivalent Lateral Force Procedure
- x. Seismic Response Coefficient(s): $C_s = 0.089$

3. Serviceability

- a. Drift & Deflection Criteria: The criteria used shall be compatible with finish materials for the building that are connected to the building structure.
 - i. Vertical Deflection
 - a) New HSS Lintel:
 - 1) Live Load: $L/600$
 - 2) Dead + Live Load: $L/600$

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V. Architectural

V. Architectural

A. Goals and Needs:

1. The primary goal is to complete replacement of Chiller #3, located in the Central Chiller Plant at the Alexandria VA Health Care System, Pineville, LA. Chiller #3 is currently decommissioned. It is recommended to replace the chilled water pump due to known issues.
2. See mechanical narrative for detailed discussion of primary requirements for this project.
3. Additional scope is to replace deteriorated areas of chilled water piping insulation as well as widen the door opening to accommodate chiller replacement and reinstall existing door and frame with new louver sidelite.

B. Existing Conditions

The existing 1-story structure consists of a main building with one main space and smaller ancillary rooms. Structure of the building consists of a precast concrete double tees roof slab supported on load bearing CMU/brick veneer exterior walls. Floor is concrete slab-on-grade. Structural calculations of the existing structure are not available.

C. Scope of Demolition

Demolition will be limited to careful removal of the existing door and frame and the widening of the opening to facilitate the removal of the existing Chiller #3. This includes the existing screen door and frame attached to the HM frame. Door, frame and all hardware are to be salvaged and reinstalled after installation of new chiller.

D. Scope of New Work:

New work will consist of making repairs to the existing CMU/brick and floor slab plus the new aluminum louver sidelite to infill at the widened opening. See Mechanical Narrative for more detail

E. Phasing:

The project will not require a phased approach since the Owner will move all operations elsewhere during the course of demolition and repairs. The sequence of renovation work will be addressed by the A/E in more detail in the Mechanical narrative.

A/E Chiller #3 Replacement, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

F. Codes and Standards:

1. VA Design Guides-Technical Information Library
2. VA Directive 1061
3. International Building Code, Latest Edition with Louisiana Amendments
4. International Fire Code/ Life Safety Code, Latest Edition with Louisiana Amendments
5. NFPA 101 – Latest Edition
6. VA Accessibility Standards
7. NFPA 99
8. NFPA 13 – Latest Edition

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VI. Fire Protection

(Not Applicable)

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VII. Plumbing (Not Applicable)

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VIII. Mechanical

VIII. Mechanical

A. Design Requirements:

1. The scope of this project is to provide a replacement for Chiller #3 located at the central chilled water plant. The replacement will include removal and introduction of major equipment, pumping modifications to incorporate new chiller, and controls integration.
2. The design will require connection to existing systems as follows:
 - a. Existing chilled water loops and equipment will be used and interconnected with the new chiller.
 - b. All upgrades will include DDC controls integrated to the existing Siemens BAS system.
3. Design criteria. In addition to “Applicable Codes and Standards” listed below, the following VA required design criteria is to be used for design development:
 - a. Department of Veterans Affairs HVAC Design Manual, 2022.
 - b. VA Physical Security and Resiliency Design Manual (PSRDM), 2020.
4. Design documents and other deliverables. As required by the contract the following products will be completed and delivered:
 - a. This design analysis.
 - b. Construction drawings.
 - c. Construction specifications.
 - d. Construction cost estimate.

B. Existing Conditions:

1. An initial survey was performed 1/27/2022 to review existing conditions. See TAB B – Site Survey for additional information.
2. Chiller 3 utilizes two pumps for flow. P-6 serves as the chilled water pump and P-7 serves as the condenser water pump.

3. P-6 was found to have a lot of corrosion and had reportedly not been operated in roughly 5 years. (B&G 6x8x12; 1560 GPM; 152 FT; HP not listed).
4. P-7 was found to be in reasonable shape but in need of touchup painting and grouting. (B&G 8x10x17.5; 1950 GPM; 83 FT; 60 HP)
5. Much of the original chilled water piping had damaged insulation and resulting condensation was apparent. It is recommended to replace select areas while insulation trade contractors are onsite.

C. Owner Furnished or Installed Items:

1. Owner furnished items: None.
2. Owner installed items: None.
3. Existing items to be reused: Existing DDC control system will be used to integrate the new chiller.

D. Applicable Codes and Standards: *(The primary resource used in this design will be VA publications. Where additional information is required to complete the design the below codes and standards will be used.)*

1. ASHRAE Standard 90.1 – 2013 Energy Standard for Buildings Except Low-Rise Residential Buildings.
2. ASHRAE Standard 170-2017– Ventilation of Health Care Facilities.
3. ASME B31.1 – 2017 Building Pressure Piping.
4. ASME B31.9 – 2017 Building Services Piping.
5. International Mechanical Code (IMC), 2018 with Addendum.
6. NFPA 70 – National Electric Code.
7. NFPA 90A – 2018 Installation of Air Conditioning and Ventilating Systems.
8. NFPA 90B – 2018 Installation of Warm Air Heating and Air Conditioning Systems.
9. NFPA 110 – Standard for Emergency and Standby Power Systems.
10. SMACNA - HVAC Air Duct Leakage Test Manual, 2012

11. SMACNA - HVAC Duct Construction Standards - Metal and Flexible, 2005

E. Technical Analysis:

1. Phasing requirements: This project has minimal phasing requirements.
2. Critical path issues: None currently.
3. Climatic Conditions: From the VA HVAC Design Manual 2017 with 2019 Revision the climatic design conditions are as follow:
 - a. Winter: 27.4 F db
 - b. Winter extreme: 21.5 F db
 - c. Summer: 97.2/77.1 db/wb
 - d. Refer to TAB A.1.
4. VA HVAC Criteria per space.
 - a. This project has no space requirements and is primarily equipment replacement.
5. Chilled Water Plant.
 - a. The chilled water plant has 4 water cooled chillers served by multiple celled cooling towers located in the adjacent equipment yard. Chillers 1, 2, and 4 are functioning to provide chilled water to the campus. Chiller 3 is offline and non-functional. The recommendation is to replace this chiller in kind.
 - b. Chiller 3: This 650-ton York machine is the oldest chiller in the plant. It has not functioned properly in several years.
 - c. Chiller capacity is low for the campus in the summer months. Likely this project will not be completed before summer so the facility will require temporary chillers to supplement the existing equipment to meet summer load. As such it is recommended to replace Chiller 3.
 1. New Chiller: 650 Tons; 1560 GPM; EWT 52 F; LWT 42 F.
 - a. The 65% design included a layout basis of Daikin WME106CSCSNA. This unit fits through the door, is magnetic bearing and can turn down to 10% capacity. **The**

VA rejected this chiller selection due to the chiller having magnetic bearing compressor.

- b. The new layout basis is a Carrier 19DV. This model is larger than the existing openings and requires widening of the existing opening. Additionally, it requires the unit to be shipped dry (charged with nitrogen) and assembled in the field due to existing height constraints.
 - d. One challenge is introducing the equipment into the space. Previous equipment replacements have required equipment access doors to be removed, widened, and replaced. This has successfully been done by extending the existing lintel and adding a new louver in place of the removed wall. The VA is pleased with this approach and the A/E will proceed with the same strategy if necessary. However, the original layout basis chiller fit through the door. **The new layout basis does not fit through the door and widening will be part of the base project.**
 - e. The chiller will be delivered dry and unassembled. The manufacturer or manufacturer representative will be required to assemble and charge the chiller.
 - f. The intent of the project is to replace only the chiller, however the chilled water pump that is existing (P-6) is recommended to be replaced. This pump has much wear and corrosion apparent and has not been operated in 5 years.
 - 1. New Chilled Water Pump (P-6): 1560 GPM; 150 Ft.HD (head matches existing); 100 HP.
- F. Information for Discussion / RFI:
- 1. None.
- G. HVAC Team Items to Complete:
- 1. None.
- H. Equipment Catalog Sheets:
- 1. TAB C – Major Equipment Sections

I. Appendixes:

1. TAB A.1 – Weather and Climate Data
2. TAB B – On-Site Survey
3. TAB C – Major Equipment Sections

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Alexandria VAMC, Alexandria, LA.

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IX. Electrical

IX. Electrical

1. Applicable Codes, Standards & Guidelines

The engineering calculations will be based on the latest recommendations of ASHRAE, NEC and good engineering practices consistent with industry standards. Mechanical equipment electrical requirements will be evaluated based on the current standards.

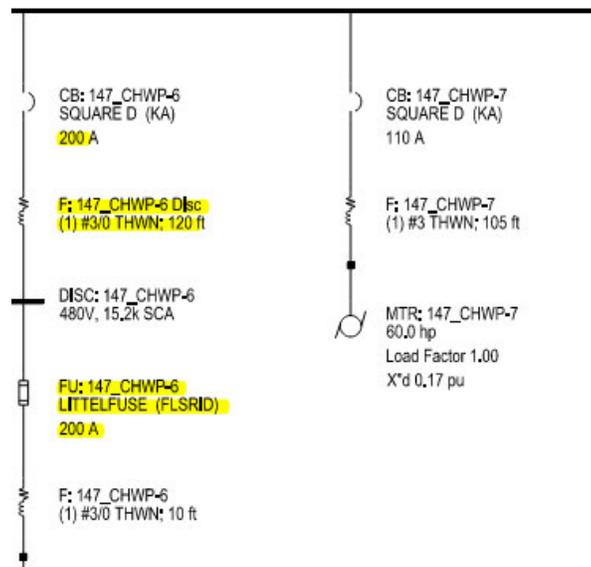
Codes applicable to the design are as follows.

- International Energy Code.
- NFPA 70 - National Electrical Code.
- NFPA 99 – Health Care Facilities
- NFPA 101 – Life Safety Code
- VA Electrical Design Manual for Hospital Projects

1.1 Electrical Service

Demolition:

Existing electrical power connected to the existing chiller #3 and chilled water pump #6 shall be demolished back to source. All exiting conductors and accessories shall be demolished. Conduits concealed in concrete walls or slabs shall be abandoned. Wiring in abandoned conduits shall be demolished back to source. Existing circuit breakers and disconnect switches shall be demolished back to source. Existing conductors shall be pulled back to the branch circuit protective device and such device shall be identified as a spare. Existing conduit shall be reused if adequate for the new circuit breaker to be installed.



Existing One-line diagram showing 200A breaker for CHWP-6



Existing MCC disconnect and starter for CWHP-6

New Work:

A new 480V 3-phase 800A NEMA-3R rated disconnect switch shall be provided for new chiller #3. Disconnects shall be circuited to spare breakers or a new 1200A adjustable circuit breaker shall be provided in existing main distribution panelboard spaces that became available during demolition. Circuit breaker shall be adjusted to 800A per the manufacturer's recommendation. The electrical contractor shall provide all new conductors, conduits and appurtenances required for the complete installation of new mechanical equipment. Should a larger circuit breaker be needed, new feeders and conduits shall be provided to serve the new mechanical equipment.

A new 480V 3-phase 200A VFD shall be provided for the new chilled water pump #6. VFD shall be circuited to spare breakers, or a new 200A circuit breaker shall be provided in existing MCC spaces that became available during demolition. The electrical contractor shall provide all new conductors, conduits and appurtenances required for the complete installation of new mechanical equipment. Should a larger circuit breaker be needed, new feeders and conduits shall be provided to serve the new mechanical equipment.

All motors shall be provided with motor controllers and disconnect switches and major equipment will be provided with non-fused safety switches, based on equipment nameplate

data. All motor controllers/starters shall be at minimum NEMA Size 1, unless required otherwise due to environmental conditions. All disconnect switches shall be heavy-duty NEMA Type 1.

Electrical Grounding System

Existing building grounding shall remain. No modifications shall be made to the existing building grounding.

1.2 Materials

The wiring conductors shall be required to meet the load criteria as well as the voltage drop criteria. In accordance with the codes (ASHRAE 90.1, and NEC) impose a maximum 5% voltage drop from the service to any load.

All branch circuit wiring will be with 600 volt rated copper conductors, type THWN or THHN, minimum #12 AWG in minimum 3/4-inch conduits. All interior conduits will be Electrical Metallic Tubing (EMT) unless required otherwise by the code. All exterior exposed conduits will be Galvanized Rigid Steel (GRS) and all exterior underground conduits will be PVC Schedule 40, unless required otherwise by code.

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Alexandria VAMC, Alexandria, LA.

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Appendix A Functional Program

Please note that the requirements specified in the attached document are from the original scope of work provided to TMM. Subsequently, these requirements have been adjusted to reflect budgetary limitations.



DEPARTMENT OF VETERANS AFFAIRS
Network Contracting Office - 16
Alexandria VA Health Care System
2495 Shreveport Hwy, 71N
Pineville, LA 71360

Toland Mizell Molnar
Attn: Mr. James Molnar
2412 Ferrand St, Suite 5
Monroe, LA 71201

Dear Mr. Molnar:

Our Architect-Engineer Evaluation Board has selected your firm for negotiation of a contract to provide appropriate design services for Project No. 502-21-110 "Replace Chiller #3" for the VA Alexandria Health Care System, 2495 Shreveport Hwy, 71N, Pineville, LA 71360 under Solicitation No. 36C25621R0126.

Your firm is requested to contact the Staff Engineer, COR, Mr. Ken Dillard, at 318-466-2465 or via email at kenneth.dillard1@va.gov , to ascertain the exact nature of the specified work to be performed.

[REDACTED]

[REDACTED]

[REDACTED]

[REDACTED]

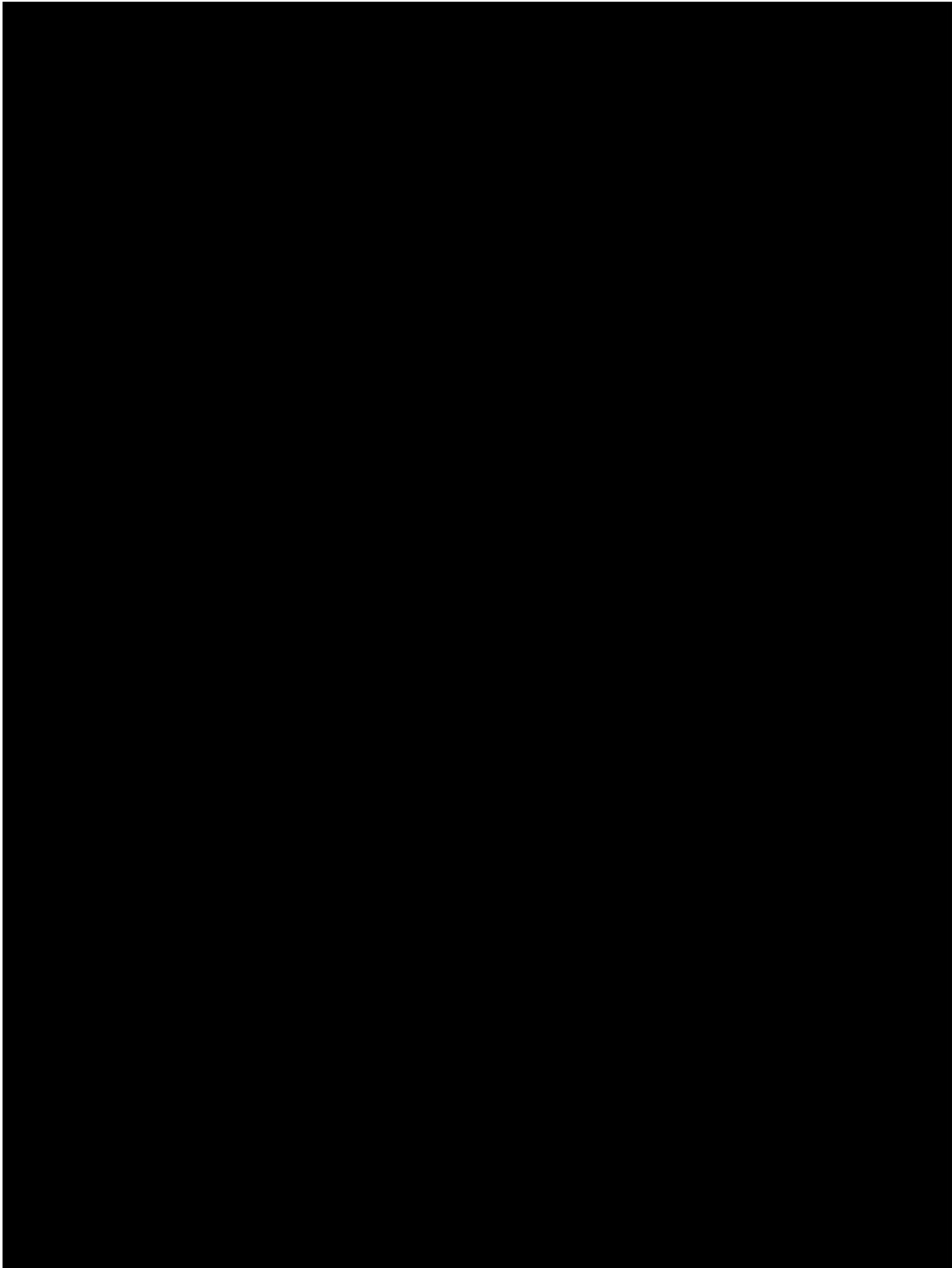
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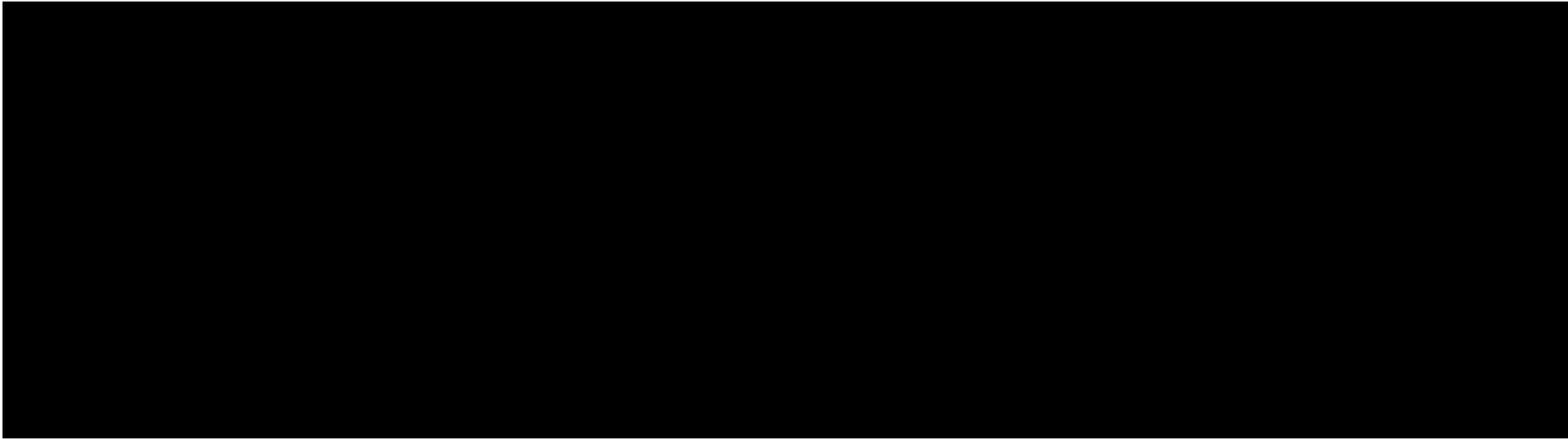
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Appendix B Project Comments & Responses

Building 147 – Replace Chiller #3, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

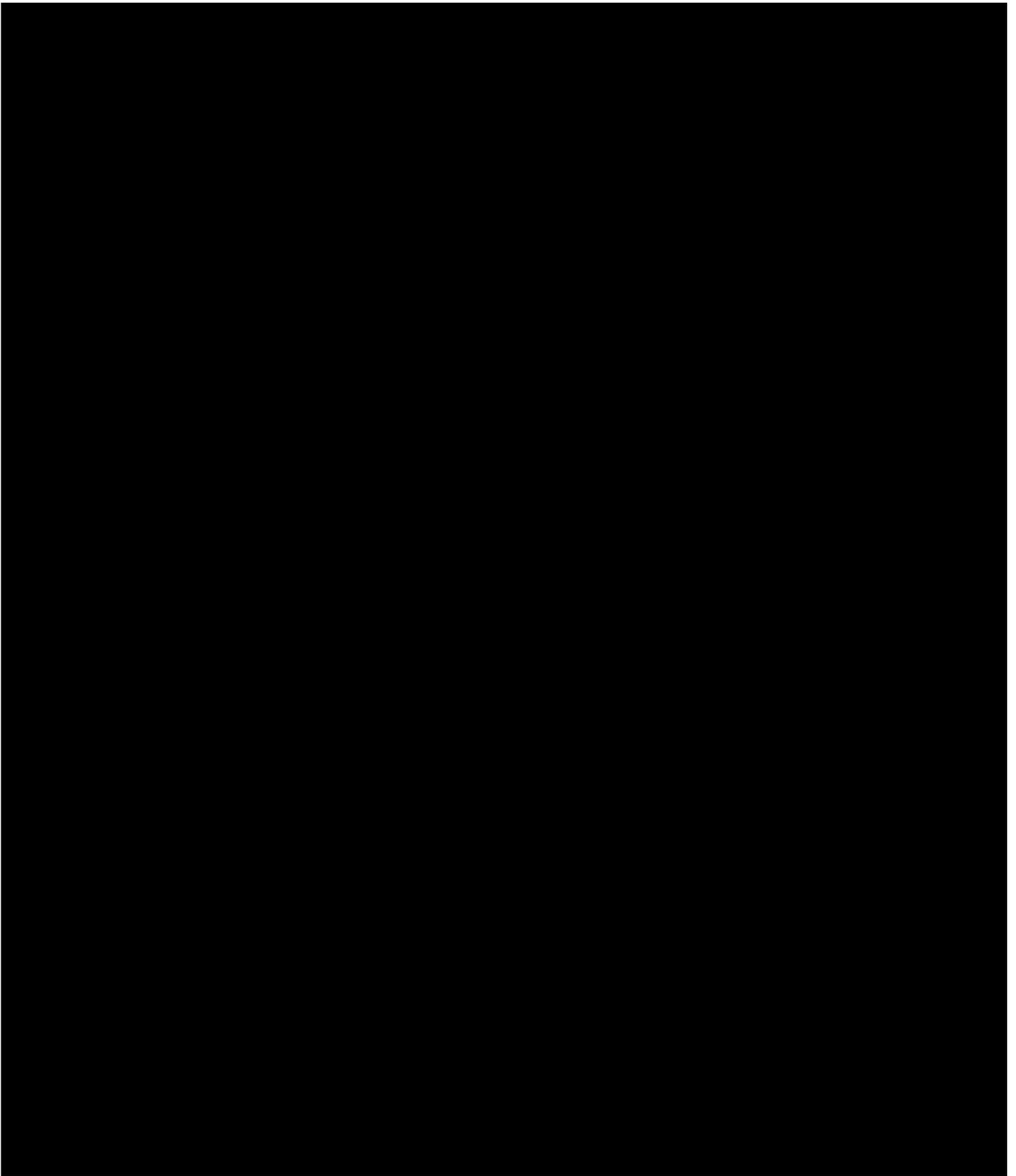
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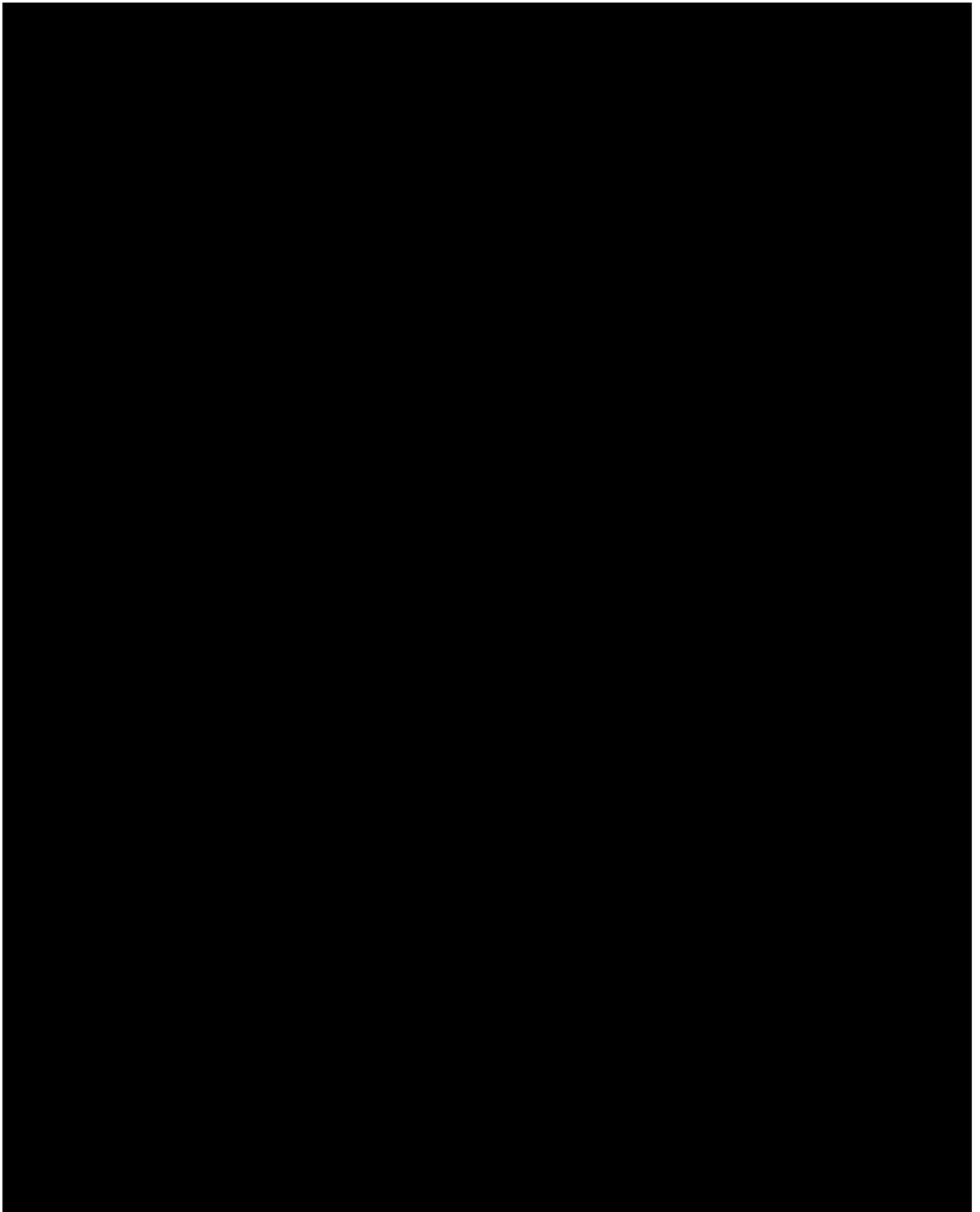
Appendix C Meeting Minutes

Building 147 – Replace Chiller #3, Project No. 502-21-110
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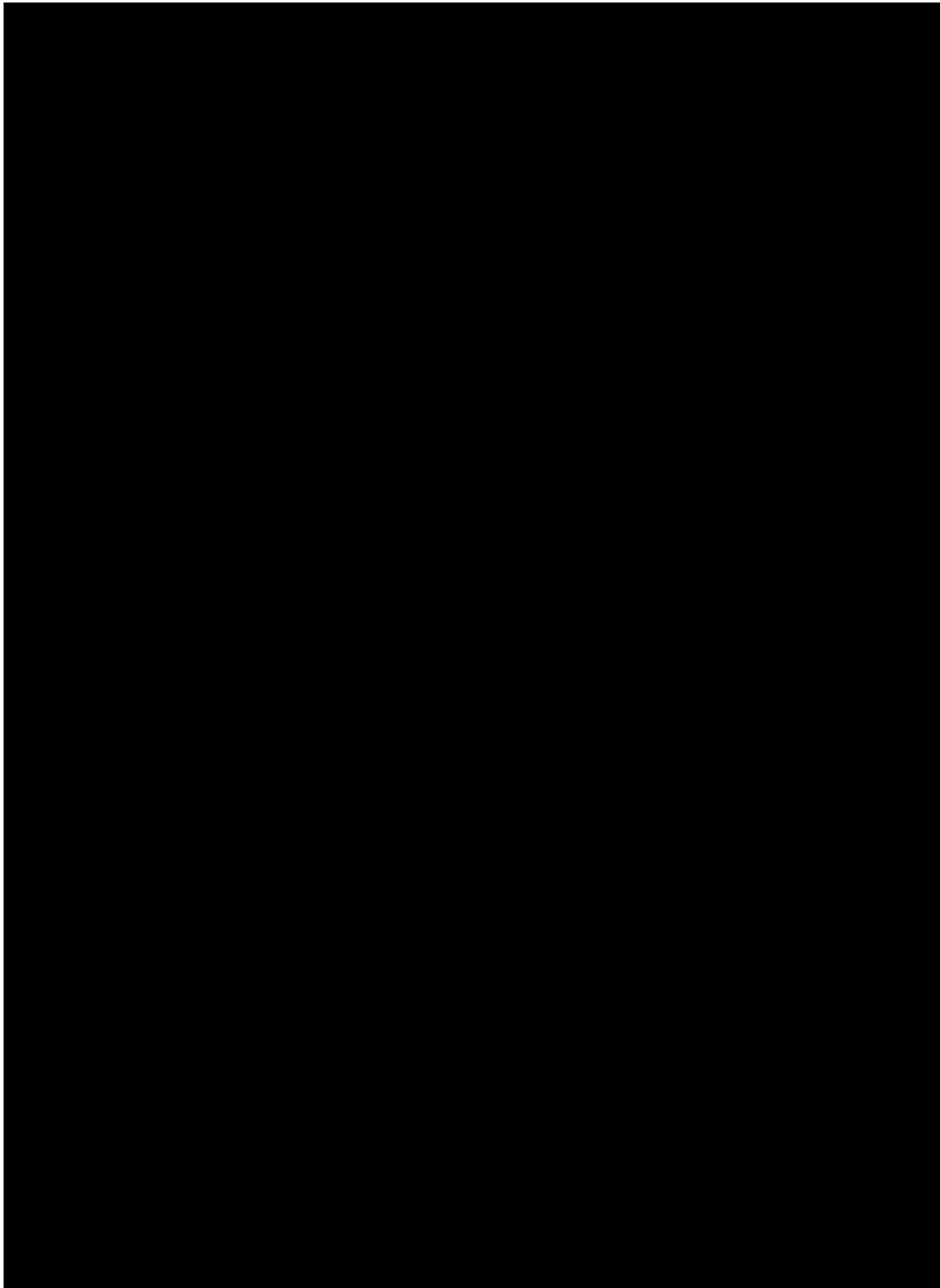
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Appendix D Cost Estimate









Building 147 – Replace Chiller #3, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

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Appendix E Specifications

(Under Separate Cover)

Building 147 – Replace Chiller #3, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

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Appendix F Asbestos & Lead-based Paint Survey (Not Applicable)

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Appendix G Mechanical

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Alexandria VAMC, Alexandria, LA.

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TAB A.1 – Weather Data

Location	Weather Station	North Latitude	MSL Elevation	Col. 1a 0.4%		Col. 1b 99.6%	Col. 2a 1%		Col. 2b 99%	Col. 3 Wet Bulb		Annual Extreme Daily-Mean Db			
				Temperatures										Maximum	Minimum
				Summer		Winter	Summer		Winter	0.4%	1%				
				Db	Wb	Db	Db	Wb	Db						
KANSAS															
Leavenworth	Kansas City Intl AP, MO	39.30	1024	95.8	76.8	2.0	92.5	76.2	7.2	79.8	78.3	99.7	-4.5		
Topeka	Topeka/Billard Municipal AP	39.07	886	97.1	76.2	3.1	93.9	75.9	8.7	79.0	77.8	101.1	-4.0		
Wichita	Wichita/Mid- Continent AP	37.65	1339	100. 1	73.7	7.4	97.0	73.8	12.2	77.7	76.5	128.5	72.9		
KENTUCKY															
Lexington	Lexington Bluegrass AP	38.04	988	91.6	73.9	8.3	89.6	73.6	13.6	77.3	76.1	127.5	73.1		
Louisville	Louisville	38.18	489	93.8	76.3	10.2	91.5	75.0	15.9	78.7	77.5	97.1	3.2		
LOUISIANA															
Alexandria	Alexandria Intl AP	31.34	79	97.2	77.1	27.4	94.7	77.3	29.3	80.7	79.8	100.2	21.5		
New Orleans	New Orleans Lakefront AP	30.04	10	93.3	78.7	35.6	91.8	78.2	38.6	81.4	80.6	96.9	29.7		
Shreveport	Shreveport Regional AP	32.45	259	98.5	76.2	25.2	96.0	76.3	28.4	79.4	78.6	101.3	19.3		
MAINE															
Togus	Augusta AP	44.32	361	87.5	70.9	-3.2	83.8	69.3	1.3	73.5	71.6	108.3	69.1		



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TAB B – Site Survey

Existing Conditions

Thursday, January 27, 2022 9:30 AM

The project is to remove chiller number three which is a York 650 ton chiller.

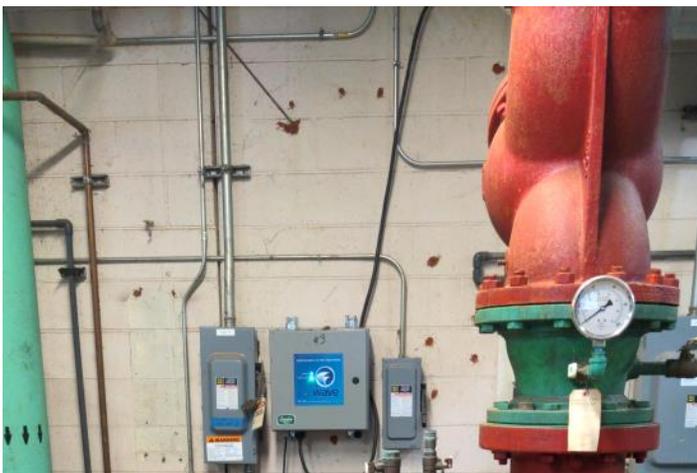
Pumping configuration

Condenser water is provided directly to the chiller through pump number 7 a HSC pump that is in working order appears to be in good shape.

Chilled water is through a common loop that is on the roof. The VA said each pump stages on with its associated chiller. So each chiller has its own pump. Chiller number three is served by a HSC pump number 6. It is stated that this pump has not been operated in 5 years. It is recommended to replace this pump size it for coordinated flow with the new chiller number three.

Condenser water pump number 7







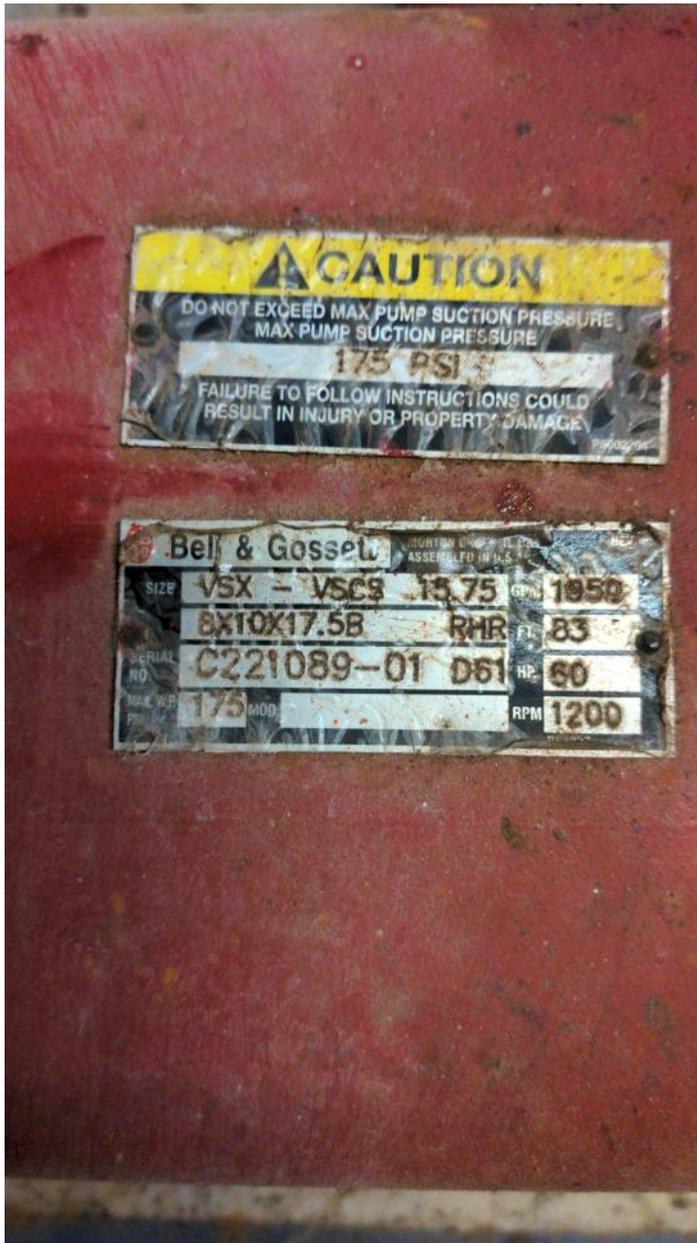
FOR LIFTING
COMPONENT
THEY ARE A
FAILURE TO FC
INSTRUCTIONS
IN INJURY OR

BALDOR • RELIANCE
SuperE Motor

CAT. NO. EM2548T		CUST. P/N		ENCL. OPSB	
SPEC. 44E057W300G1		CC 010A	FRAME 404T	SER. NO. C1510071247	
HP 60	CLASS F	HZ 60		BEARING SIZE	
R.P.M. 1185	PH 3	DES. A		OPP. END 6312	DRIVE END 6316
VOLTS 230/460		CODE H		GREASE POLYREX EM	
MPPS 152/76		USABLE AT 208 T60 A		THERMOSTAT LEADS (WIRE PROVIDED) 115 230 460 575 60 30 15 11.2 USE A MANUAL MOMENTARY START SWITCH ONLY	
RATING 40C AMB - CONT		SER. F 1.15		THERMOSTAT LEADS	
NEMA NOM. EFF. 94.5 % PF		79 %		115 230 460 575 60 30 15 11.2	
LEADS H1-H2		MAX. SPACE HEATER TEMP °C		USE A MANUAL MOMENTARY START SWITCH ONLY	

BALDOR ELECTRIC CO. • FT. SMITH, AR. MFG. IN U.S.A.





chilled water pump number 6











Chiller#3













SOLID STATE MOTOR STARTER

TYPE "1" ENCLOSURE

MODEL **SSS26LK-46B**

SERIAL NO. **DOS-006**

PART NO. **371-02507-225**

NAMEPLATE
VOLTAGE

OPERATING
VOLTAGE RANGE

200	180 - 220
230	208 - 254
380	342 - 402
400	360 - 423
415	374 - 440
460	414 - 508
575	520 - 635

STARTER ELECTRICAL RATINGS

VOLTS - PHASE - HZ **460-3-60**

MAX. FLA **850**

MAX. START CURRENT **2600**

MIN. FLA **125**

MAX FUSE/CB **1600**

MAX. HP **900**

MAX. LRA **5780**

SUITABLE FOR USE ON A CIRCUIT
CAPABLE OF DELIVERING NOT MORE
THAN **42000** RMS SYMMETRICAL
AMPERES, **460** RMS VOLTS

OIL PUMP SUPPLY

VOLTS-PHASE-HZ **460-3-60** RATED AMPS: **3.5**

CONTROL SUPPLY TRANSFORMER

VOLTS-PHASE-HZ **120-1-60** RATED AMPS: **16.5**

YORK INTERNATIONAL CORPORATION
YORK PA 17405

029-22932 REV. -









P/N 026-41701-000

MODEL YORK24-12X5-4170i

UL File No. SA9729

Y005034

Mfg Date 05.12.21

S/N S/N: H024AC34905030

Made in U.S.A

MAX. DESIGN PRES. PSI 450 (31)

MAX. DESIGN TEMP. °F 350 (177)

LISTED
HEAT EXCHANGER
3P77







YORK MAXE

LIQUID CHILLING SYSTEM

UNIT MODEL
WIRING DIAGRAM

	EVAP	COND.
REFRIG. DWP, PSIG:	<input type="text" value="235"/>	<input type="text" value="235"/>
LIQUID DWP, PSIG:	<input type="text" value="150"/>	<input type="text" value="150"/>
NO. OF PASSES:	<input type="text" value="2"/>	<input type="text" value="2"/>
SHELL TEST PRESS., PSIG:	<input type="text" value="306"/>	<input type="text" value="306"/>
TUBE DESCRIPTION:	<input type="text" value="271"/>	<input type="text" value="260"/>

REFRIGERANT CHARGE, LBS.
CHARGED: FACTORY FIELD

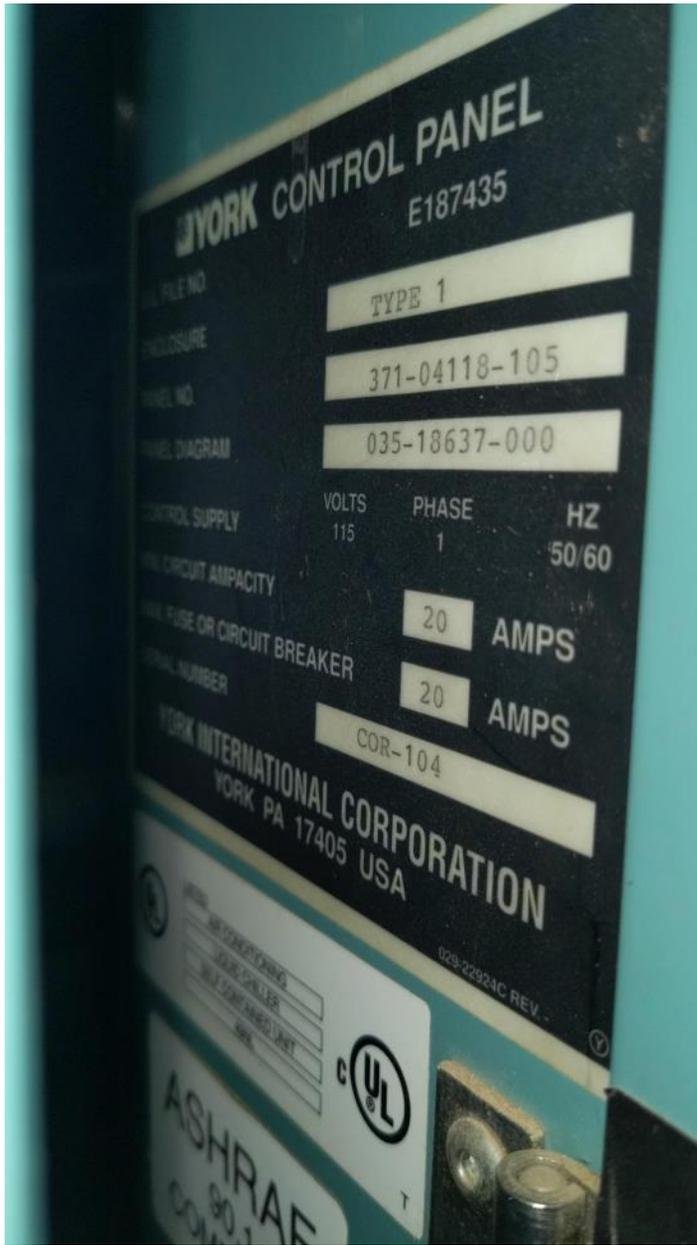
CHARGE WITH YORK REFRIGERANT OIL

SEE STARTER NAMEPLATE AND CONTROL
PANEL NAMEPLATE FOR ELECTRICAL DATA.
FOR REMOTE STARTER SEE YORK STD.

STARTER SUPPLIED BY: FACTORY FIELD
FIELD SUPPLY: VOLTS PHASE HERTZ

MIN. CIRCUIT AMPACITY
MAX. DUAL ELEMENT FUSE AMPS
MAX. CIRCUIT BREAKER AMPS





Chiller #1





Carrier

A United Technologies Company

MODEL NUMBER 19XRV70704W6LHH64S

SERIAL NUMBER 4412Q22161

REFRIGERATION MACHINE

	MODEL NUMBER	SERIAL NUMBER
MACHINE	19XRV70704W6LHH64S	4412Q22161
COMPRESSOR	4W6	4412Q22161
COOLER	70	4412Q221614
CONDENSER	70	4412Q221615
ECONOMIZER		
VFD	19XVE0960475B2F-	N/R
REFRIGERANT R-134A	2249 LBS.	1020 KGS.
	FACTORY CHARGED	
TEST PRESSURE	204 PSI	1406 KPA
DESIGN PRESSURE	185 PSI	1275 KPA
CLR. WATER PRESSURE	150 PSI	1035 KPA
COND. WATER PRESSURE	1035 PSI	1035 KPA
RATED TONS	1000	
RATED kW	588	

Carrier Charlotte
 9701 Old Statesville Road
 Charlotte, North Carolina 28269
 ASSEMBLED IN USA with domestic
 and globally sourced components.
 Production Year: 2012

CHILLER

#1

Handwritten note on a piece of paper, partially visible, containing the word "Check" and some other illegible characters.

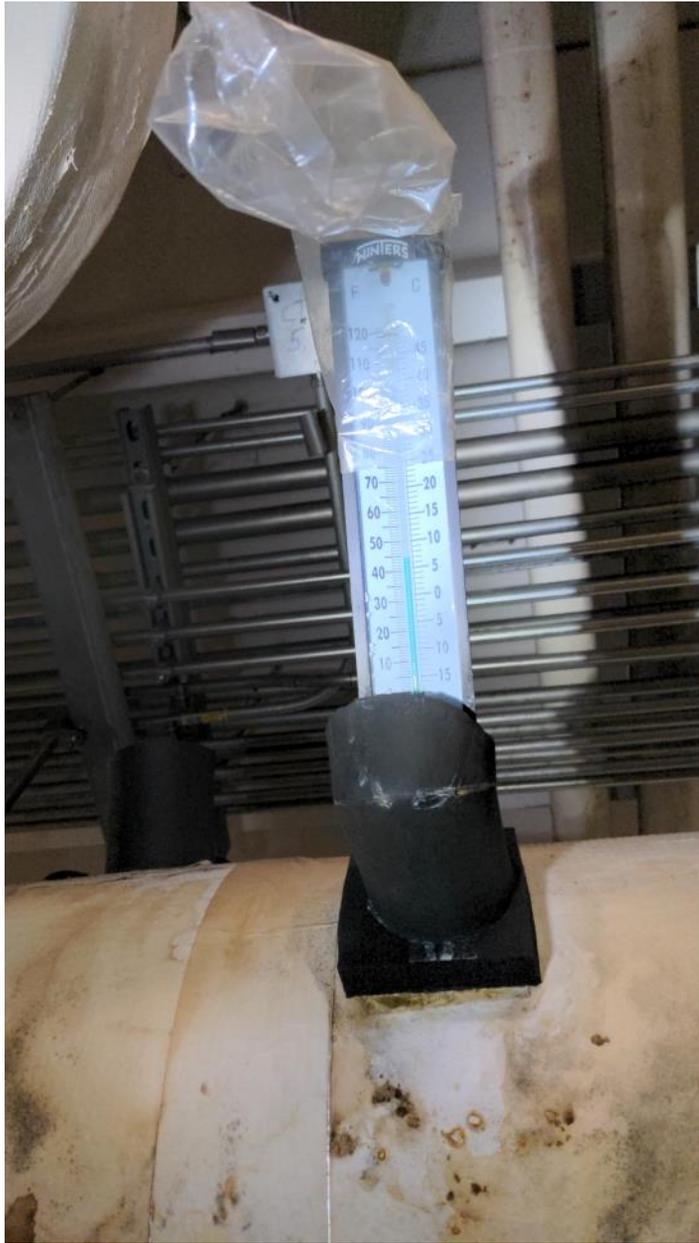
ANNUALLY STOPPED - PRESS 01-27-22 11:38
ON OR LOCAL TO START 42122.0 HOURS

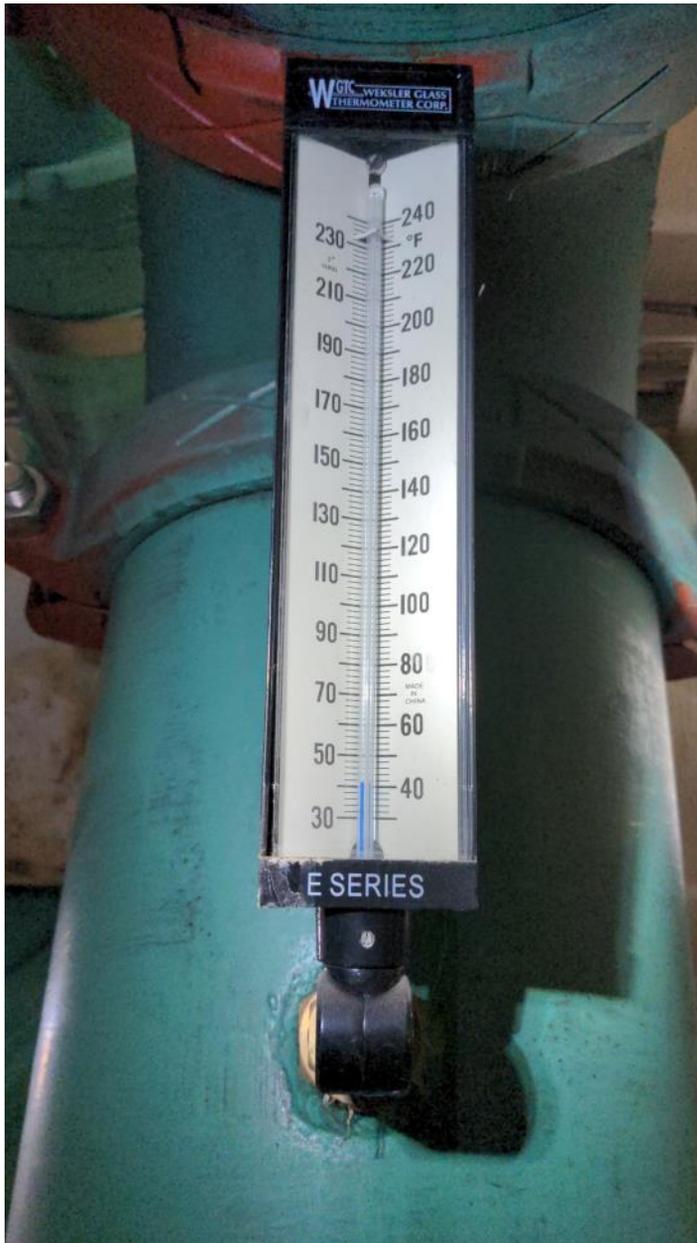
CHW IN	CHW OUT	EVAP REF
47.7	45.5	41.0
CDW IN	CDW OUT	COND REF
40.9	40.8	43.0
OILPRESS	OIL TEMP	SAMPS IN
-0.3	150.0	0.0

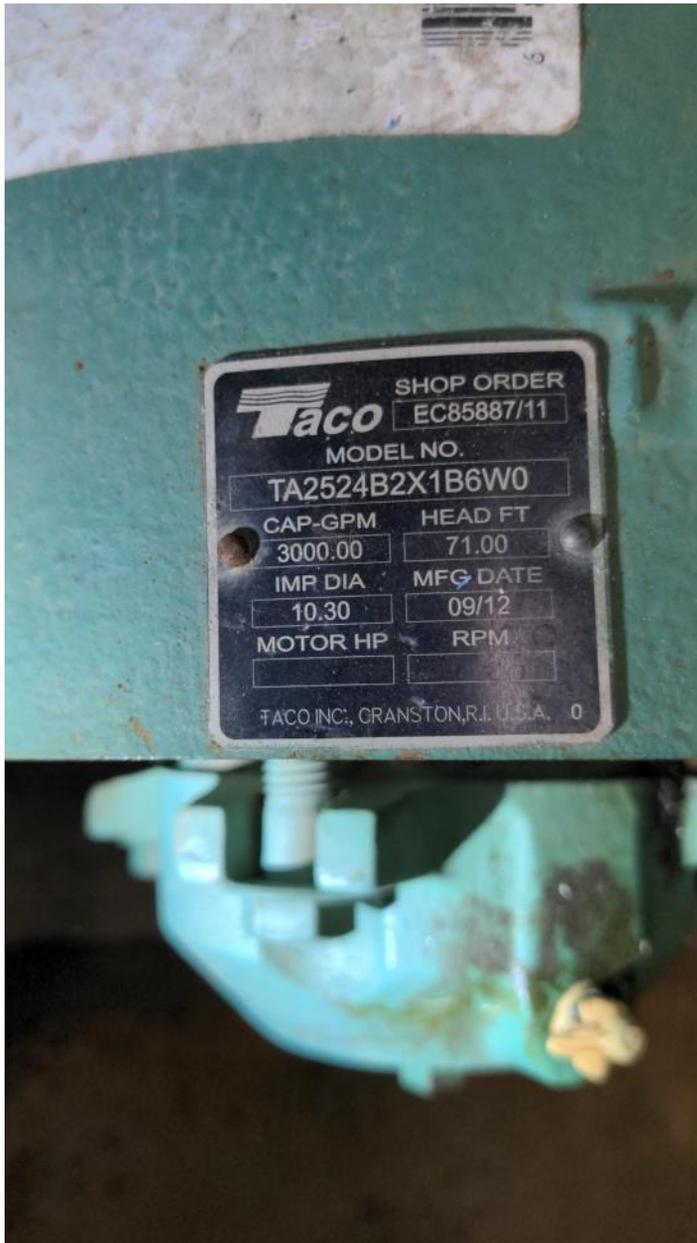
CCN LOCAL RESET MENU





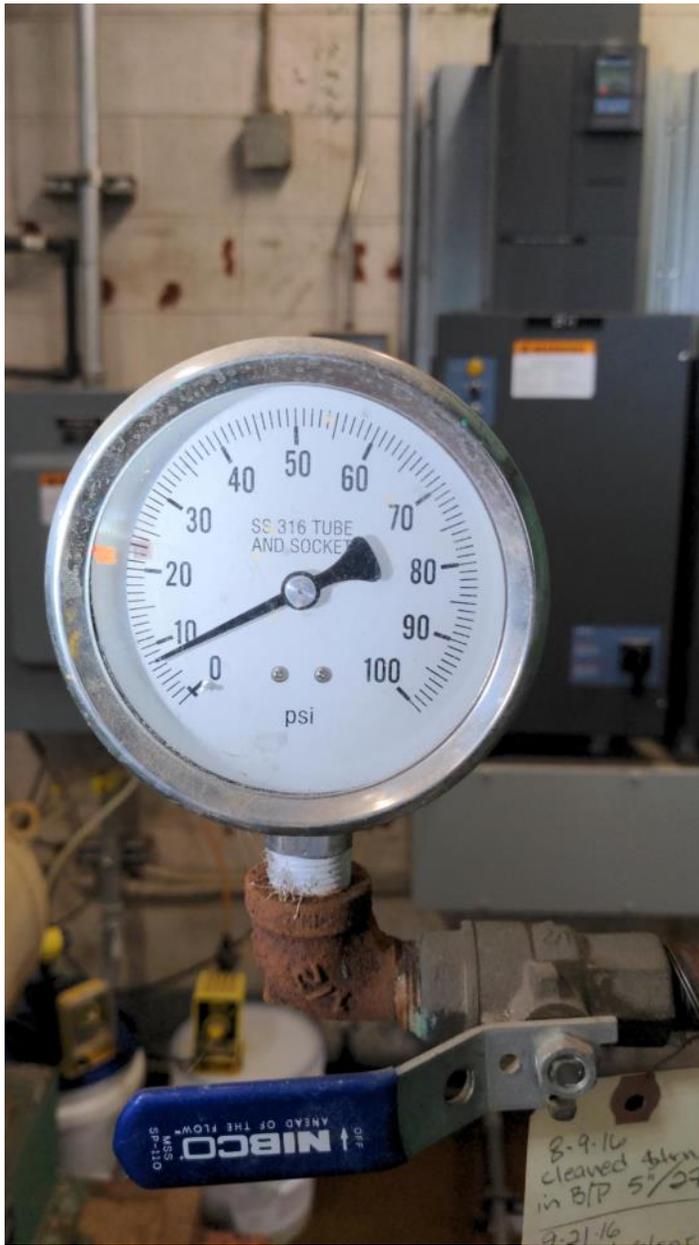


















Chiller 2

Hot has
below 90°



Chiller #2 in service 5 May 2016

SMART

CHILLER #2



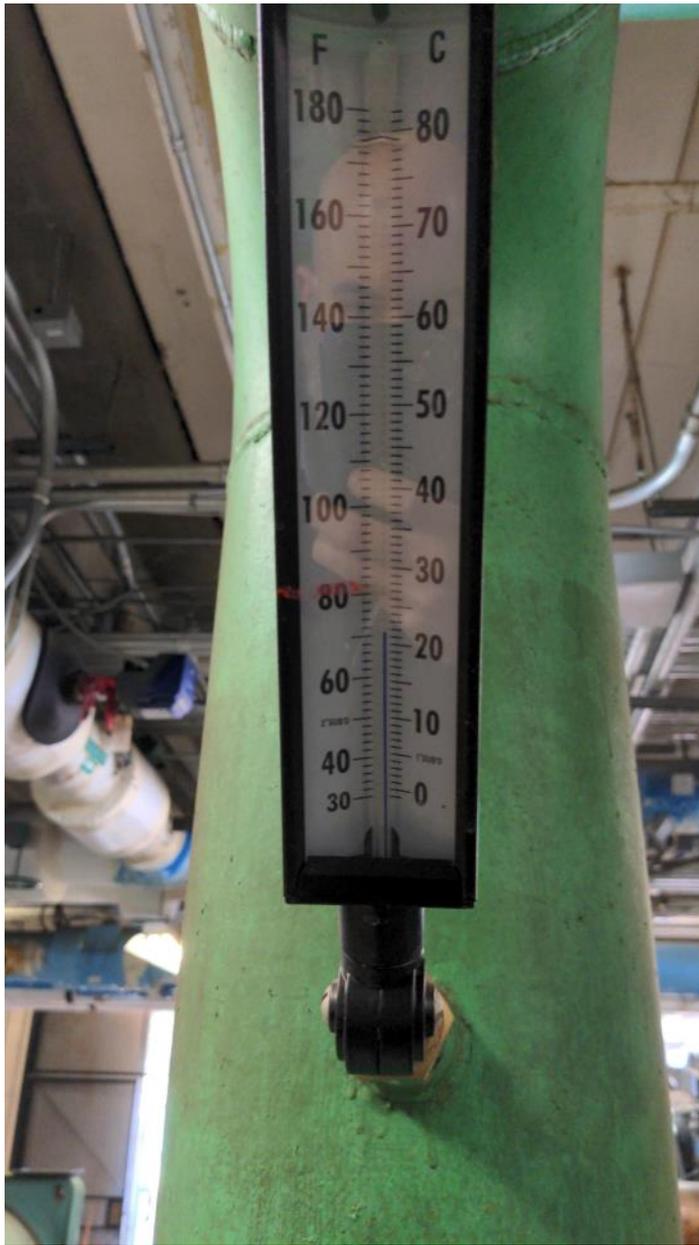
<input type="checkbox"/> U W	Certified by:	2722	
		Smardt Inc.	SHELL
	Year	2015	MAWP 1320kPa at 53°C (191psi at 127°F)
	S/N	CO-15126	MDMT -10°C at 1320kPa (14°F at 191psi)
TUBES (non-code)			
DWG. No.	11-0476 Rev.A		MAWP 1000kPa at 52°C (145psi at 126°F)
CRN / DAN	N/A		MDMT -10°C at 1000kPa (14°F at 145psi)
<p>CAUTION: The heat exchanger design has been evaluated for the range of conditions listed on form U5 of the MDR. It shall be re-evaluated for conditions outside this range before being operated at them.</p> <p>1800 Trans Canada Highway, Montreal, QC, Canada</p>			

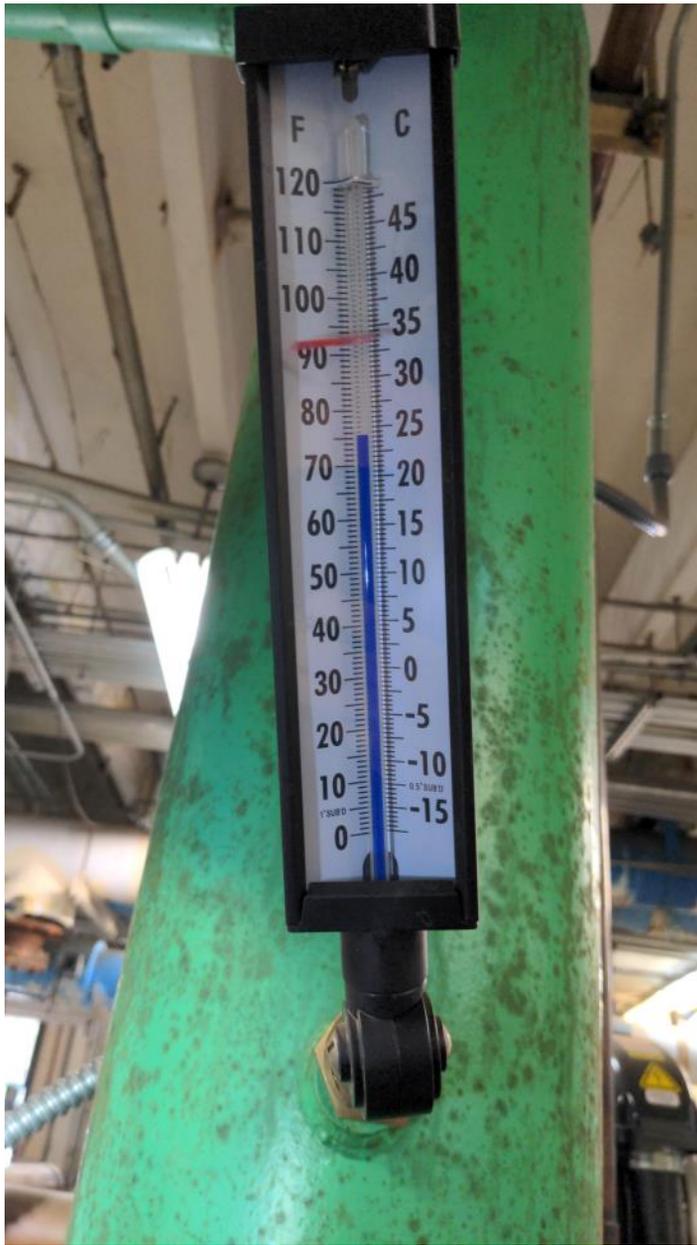






TACO		SHOP ORDER
		458782/3
MODEL NO.		
TA2530B2Q1B6L0		
CAP-GPM	HEAD FT	
IMP DIA	MFG DATE	
10.70	02/16	
MOTOR HP	RPM	
75	1750	
TACO INC., CRANSTON, R.I. U.S.A. 0		









Chiller 4





Carrier
A United Technologies Company

MODEL NUMBER 19XRV5P51436LCH64S
SERIAL NUMBER 4212Q22160

REFRIGERATION MACHINE

MACHINE	MODEL NUMBER	SERIAL NO.
COMPRESSOR	19XRV5P51436LCH64S	4212Q22160
COOLER	436	4212Q22160
CONDENSER	5P	4212Q221604
ECONOMIZER	51	4212Q221605
VFD	19XVE0550455B2F-	N/R
REFRIGERANT R-134A	1087 LBS.	493 KGS.

TEST PRESSURE	FACTORY CHARGED	
DESIGN PRESSURE	204 PSI	1406 KPA
CLR. WATER PRESSURE	185 PSI	1275 KPA
COND. WATER PRESSURE	150 PSI	1035 KPA
RATED TONS	150 PSI	1035 KPA
RATED IKW	600	365

Carrier Charlotte
9701 Old Statesville Road
Charlotte, North Carolina 28269
ASSEMBLED IN USA with domestic
and globally sourced components.
Production Year: 2012

SAFETY CODE CERTIFICATION
THIS UNIT IS DESIGNED, CONSTRUCTED, AND TESTED IN CONFORMANCE
WITH ANSI/ASHRAE 15 (LATEST REVISION), SAFETY CODE FOR
MECHANICAL REFRIGERATION



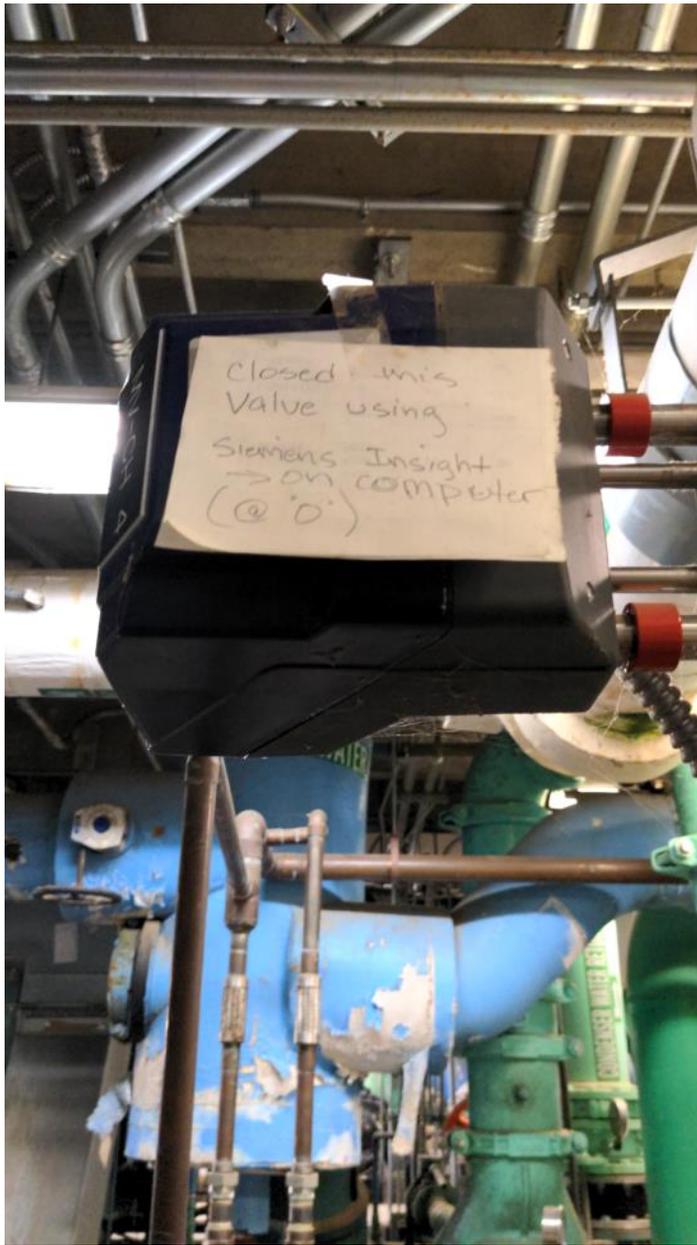
SELF CONTAINED
US LIQUID CHILLER,
57R9











Closed this
Valve using
Siemens Insight
→ on computer
(@ 0)





Carrier

A United Technologies Company

ASSEMBLED IN USA with domestic and globally sourced components

CENTRIFUGAL COMPRESSOR

S/N 4212Q22160	MODEL 02XR-436LCH64S	
TEST PRESSURE (GAGE)	204 PSI 1407 KPa	
MAX WORKING PRESSURE (GAGE)	185 PSI 1276 KPa	
REFRIGERANT R-134A		
VOLTS AC/PH/Hz 460 / 3 / 60		
RL AMPS 546	LR AMPS Y - 1149 D - 3358	OLT AMPS 589









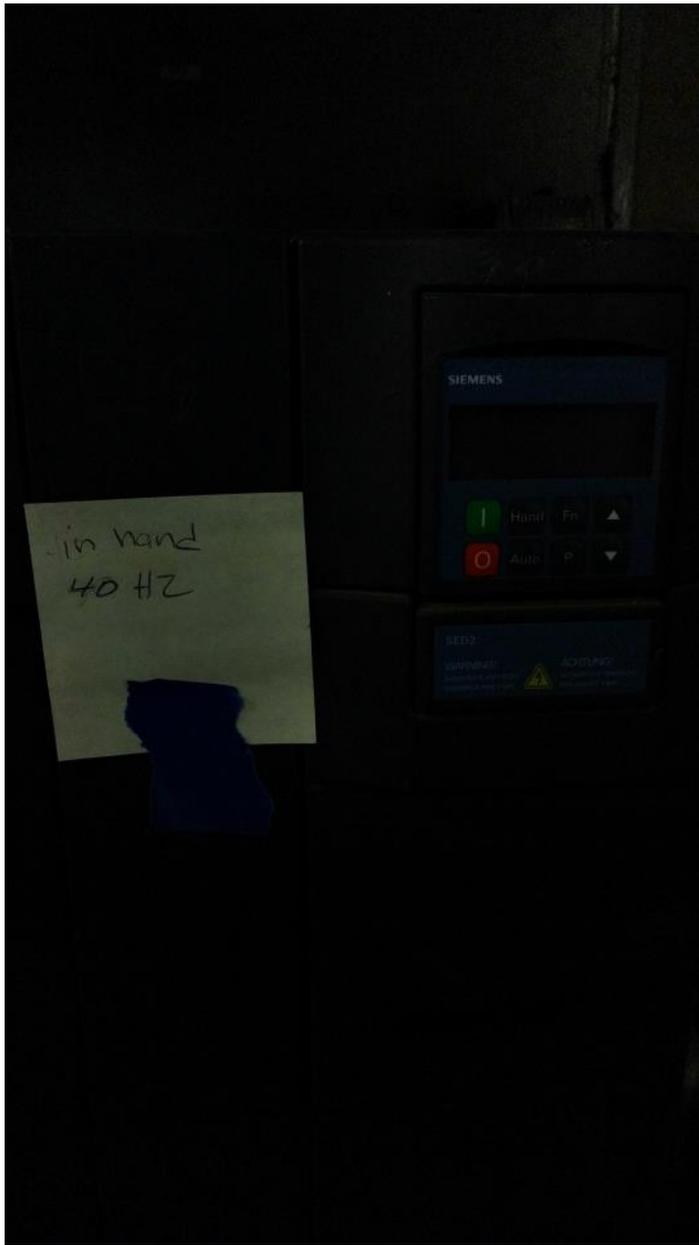
Electrical













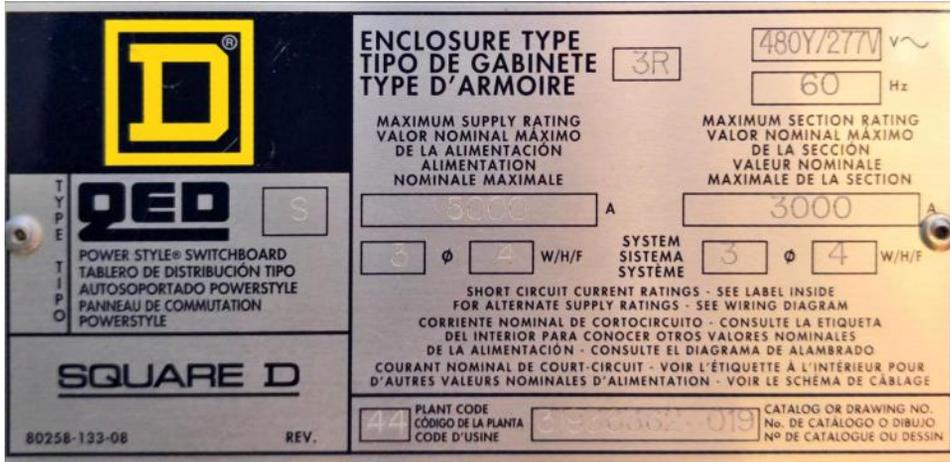


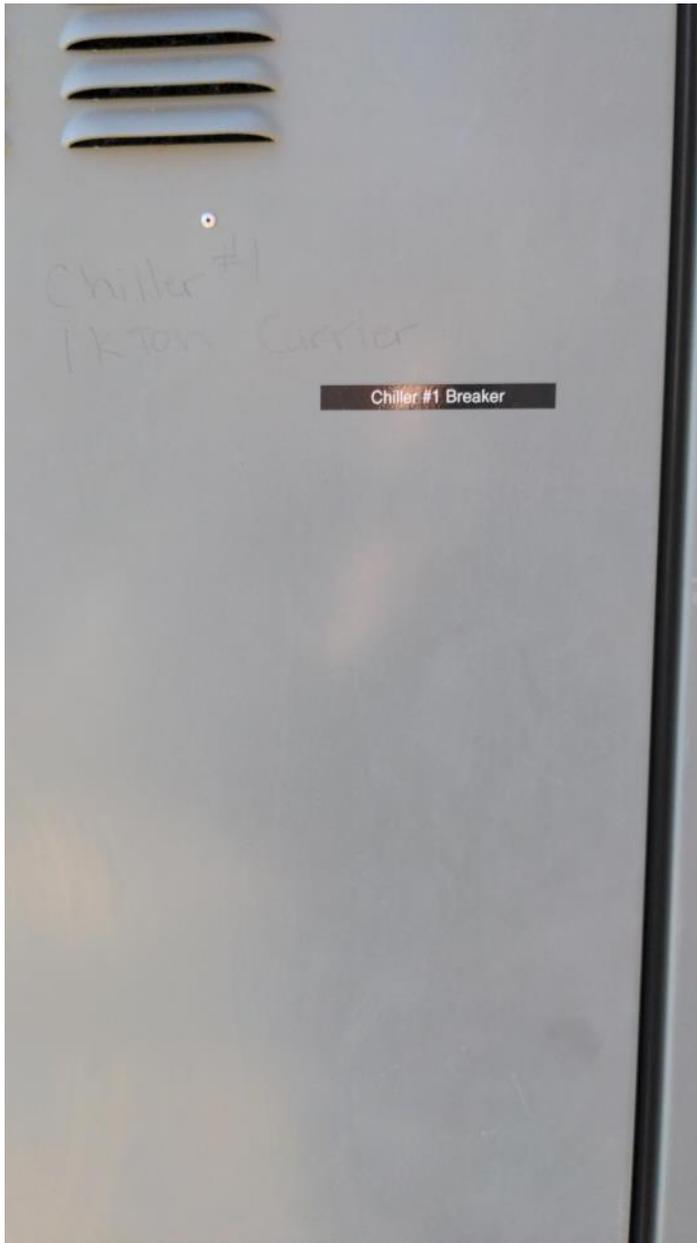






 SQUARE D 80258-133-08 REV.	ENCLOSURE TYPE TIPO DE GABINETE TYPE D'ARMOIRE	<input type="text" value="3R"/>	<input type="text" value="480Y/277V"/> \sim	
	MAXIMUM SUPPLY RATING VALOR NOMINAL MÁXIMO DE LA ALIMENTACIÓN ALIMENTACIÓN NOMINALE MAXIMALE	<input type="text" value="5000"/> A	MAXIMUM SECTION RATING VALOR NOMINAL MÁXIMO DE LA SECCIÓN VALEUR NOMINALE MAXIMALE DE LA SECTION	<input type="text" value="5000"/> A
	POWER STYLE® SWITCHBOARD TABLERO DE DISTRIBUCIÓN TIPO AUTOSOPORTADO POWERSTYLE PANNEAU DE COMMUTATION POWERSTYLE	<input type="text" value="S"/>	SYSTEM SISTEMA SYSTÈME	<input type="text" value="3"/> ϕ <input type="text" value="4"/> W/H/F
	SHORT CIRCUIT CURRENT RATINGS - SEE LABEL INSIDE FOR ALTERNATE SUPPLY RATINGS - SEE WIRING DIAGRAM CORRIENTE NOMINAL DE CORTOCIRCUITO - CONSULTE LA ETIQUETA DEL INTERIOR PARA CONOCER OTROS VALORES NOMINALES DE LA ALIMENTACIÓN - CONSULTE EL DIAGRAMA DE ALAMBRADO COURANT NOMINAL DE COURT-CIRCUIT - VOIR L'ÉTIQUETTE À L'INTÉRIEUR POUR D'AUTRES VALEURS NOMINALES D'ALIMENTATION - VOIR LE SCHEMA DE CÂBLAGE			
PLANT CODE CÓDIGO DE LA PLANTA CODE D'USINE		<input type="text" value="44"/>	<input type="text" value="31936362-019"/>	CATALOG OR DRAWING NO. No. DE CATALOGO O DIBUJO Nº DE CATALOGUE OU DESSIN













Model 6000 (Series 6000) © 2014, Eaton Power Quality www.eaton.com

• OK • Replace Module / Sustituya el módulo / Remplacez le module

Push-to-Test / Botón de prueba / Bouton pression pour vérifier

A0 B0 C0

Current / Corrente / Courant

SurgeLOGIC™ Surge Protective Device

Dispositivo de Protección contra Supertensiones Transitorias / Dispositif de Protection contre les Surintensités

⚠ DANGER

⚠ PELIGRO

⚠ DANGER











Spectra RMS™

**H.I.C.
CIRCUIT BREAKER**

Cat. No. SKL836BD1Z00
1200 Amp. Max. 600 VAC
3 Pole
Rating Plug Type: SRPK1200B

Lug Data: 75°C Wire
Cat. No: TCAL125

Wire Range: 250 - 350kcmil CU
250 - 500kcmil AL

Torque: 375 lb-in

REPLACE LUG COVER





Power Break® II

STD BREAK CIRCUIT BREAKER

CATALOG NO. SSF40B440
SERIAL NO. 175022951400101
4000 AMP. MAX. 600V AC
3 POLE
RATING PLUG TYPE: TR40B

INTERRUPTING RATING
RMS SYM. AMPS 50/60 Hz

VOLTS KA

240	100
480	100
600	85



SHORT TIME RATING
42 KA 1/2 SEC

E631=

AmeriGrid Solutions

1-330-795-3041

Electrical Testing Engineering and Maintenance

Inspected by: _____
Date: **11-6-17** MSR#: **✓**

Tested by: **LAW**

Americanpowergrid.com







Electrical

Thursday, January 27, 2022 10:54 AM







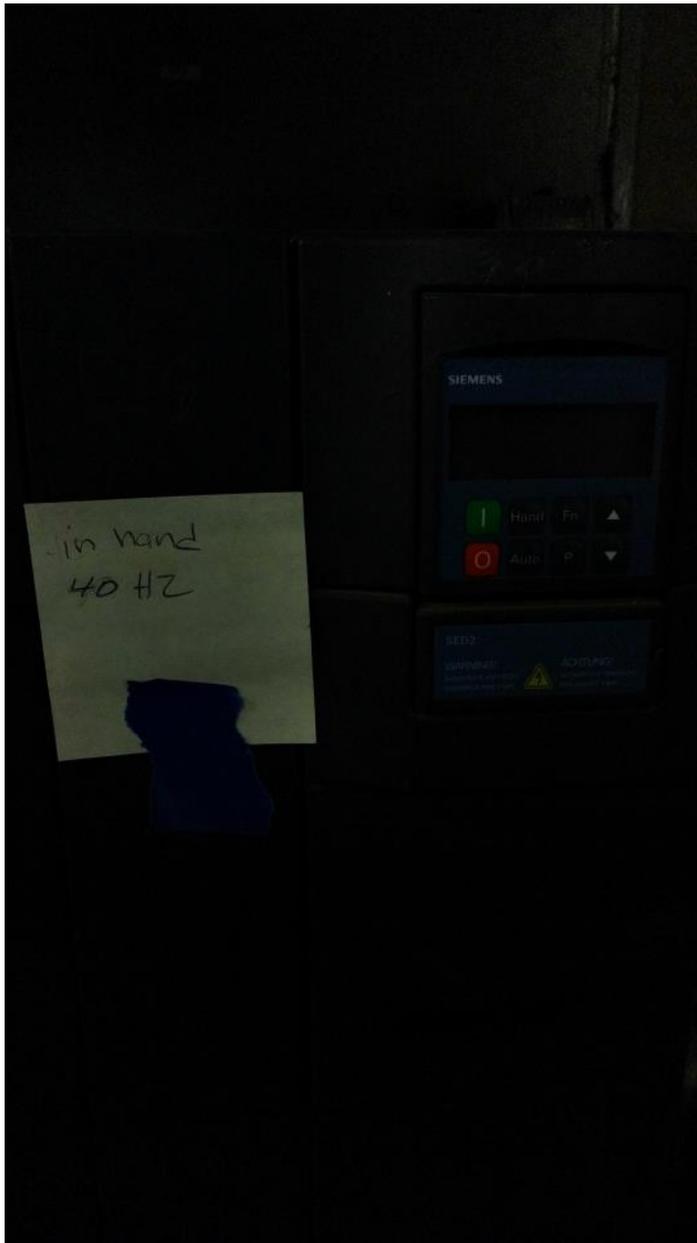
Electrical

















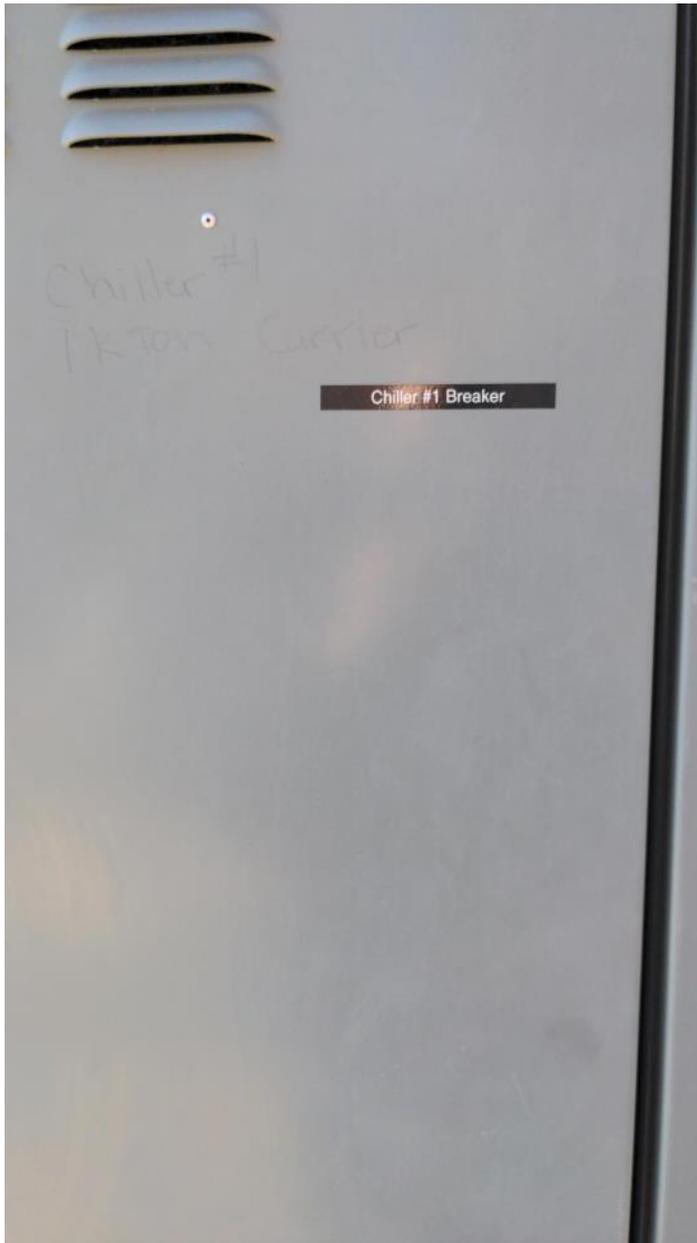




	ENCLOSURE TYPE TIPO DE GABINETE TYPE D'ARMOIRE		3R	480Y/277V \sim
				60 Hz
POWER STYLE® SWITCHBOARD TABLERO DE DISTRIBUCIÓN TIPO AUTOSOPORTADO PANNEAU DE COMMUTATION POWERSTYLE	MAXIMUM SUPPLY RATING VALOR NOMINAL MÁXIMO DE LA ALIMENTACIÓN ALIMENTATION NOMINALE MAXIMALE		5000 A	5000 A
			3 ϕ W/H/F	4 ϕ W/H/F
SHORT CIRCUIT CURRENT RATINGS - SEE LABEL INSIDE FOR ALTERNATE SUPPLY RATINGS - SEE WIRING DIAGRAM CORRIENTE NOMINAL DE CORTOCIRCUITO - CONSULTE LA ETIQUETA DEL INTERIOR PARA CONOCER OTROS VALORES NOMINALES DE LA ALIMENTACIÓN - CONSULTE EL DIAGRAMA DE ALAMBRADO COURANT NOMINAL DE COURT-CIRCUIT - VOIR L'ÉTIQUETTE À L'INTÉRIEUR POUR D'AUTRES VALEURS NOMINALES D'ALIMENTATION - VOIR LE SCHEMA DE CÂBLAGE				
80258-133-08	REV.	44	PLANT CODE / CÓDIGO DE LA PLANTA / CODE D'USINE	31936362-019
			CATALOG OR DRAWING NO. No. DE CATALOGO O DIBUJO N° DE CATALOGUE OU DESSIN	



 <p>SQUARE D</p> <p>80258-133-08 REV.</p>	<p>ENCLOSURE TYPE TIPO DE GABINETE TYPE D'ARMOIRE</p> <p>3R</p>	<p>480Y/277V \sim</p> <p>60 Hz</p>	
	<p>MAXIMUM SUPPLY RATING VALOR NOMINAL MÁXIMO DE LA ALIMENTACIÓN ALIMENTATION NOMINALE MAXIMALE</p> <p>3000 A</p>	<p>MAXIMUM SECTION RATING VALOR NOMINAL MÁXIMO DE LA SECCIÓN VALEUR NOMINALE MAXIMALE DE LA SECTION</p> <p>3000 A</p>	
	<p>TYPE QED S</p> <p>POWER STYLE® SWITCHBOARD TABLERO DE DISTRIBUCIÓN TIPO AUTOSOPORTADO POWERSTYLE PANEAU DE COMMUTATION POWERSTYLE</p>	<p>SYSTEM SISTEMA SISTÈME</p> <p>3 ϕ 4 W/H/F</p>	<p>3 ϕ 4 W/H/F</p>
	<p>SHORT CIRCUIT CURRENT RATINGS - SEE LABEL INSIDE FOR ALTERNATE SUPPLY RATINGS - SEE WIRING DIAGRAM CORRIENTE NOMINAL DE CORTOCIRCUITO - CONSULTE LA ETIQUETA DEL INTERIOR PARA CONOCER OTROS VALORES NOMINALES DE LA ALIMENTACIÓN - CONSULTE EL DIAGRAMA DE ALAMBRADO COURANT NOMINAL DE COURT-CIRCUIT - VOIR L'ÉTIQUETTE À L'INTÉRIEUR POUR D'AUTRES VALEURS NOMINALES D'ALIMENTATION - VOIR LE SCHEMA DE CÂBLAGE</p>		
<p>PLANT CODE CÓDIGO DE LA PLANTA CODE D'USINE</p> <p>44</p>	<p>CATALOG OR DRAWING NO. No. DE CATALOGO O DIBUJO Nº DE CATALOGUE OU DESSIN</p> <p>44-2333-019</p>		





















Spectra RMS™

**H.I.C.
CIRCUIT BREAKER**

Cat. No. SKL836BD1Z00
1200 Amp. Max. 600 VAC
3 Pole
Rating Plug Type: SRPK1200B

Lug Data: 75°C Wire
Cat. No: TCAL125

Wire Range: 250 - 350kcmil CU
250 - 500kcmil AL

Torque: 375 lb-in

REPLACE LUG COVER



WARNING

Arc Flash and Shock Risk
Appropriate PPE Required

EFFECTION		SHOCK PROTECTION	
lim	18 in	Shock Hazard	480 VAC
i	10.2 cal/cm ²	Exposure	60
tday	48 in	Glove Class	62 in
	Level 3	Limited Approach	42 in
		Restricted Approach	12 in

Refer to the Safety Manual for protective clothing and work.

MDP 3-Prod 9-16P-6161760 MDP June 19, 2018

MDP
MAIN BREAKER

Power Break™ II

480V BREAK CIRCUIT BREAKER
CIRCUIT BREAKER, 480V AC, 100A, 1-PHASE
LIMITING ENERGY
100A/100A, 100A/100A
100A/100A, 100A/100A
100A/100A, 100A/100A

AmeriGrid Solutions
1-800-755-3041
11-6-17
LAW

ON

Push Off



Power Break® II

STD BREAK CIRCUIT BREAKER

CATALOG NO. SSF40B440
SERIAL NO. 175022951400101
4000 AMP. MAX. 600V AC
3 POLE
RATING PLUG TYPE: TR40B

INTERRUPTING RATING
RMS SYM. AMPS 50/60 Hz

VOLTS KA

240	100
480	100
600	85



SHORT TIME RATING
42 KA 1/2 SEC

E631=

AmeriGrid Solutions

1-330-795-3041

Electrical Testing Engineering and Maintenance

Inspected by: _____
Date: **11-6-17** MSR#:

Tested by: **LAW**

Americanpowergrid.com

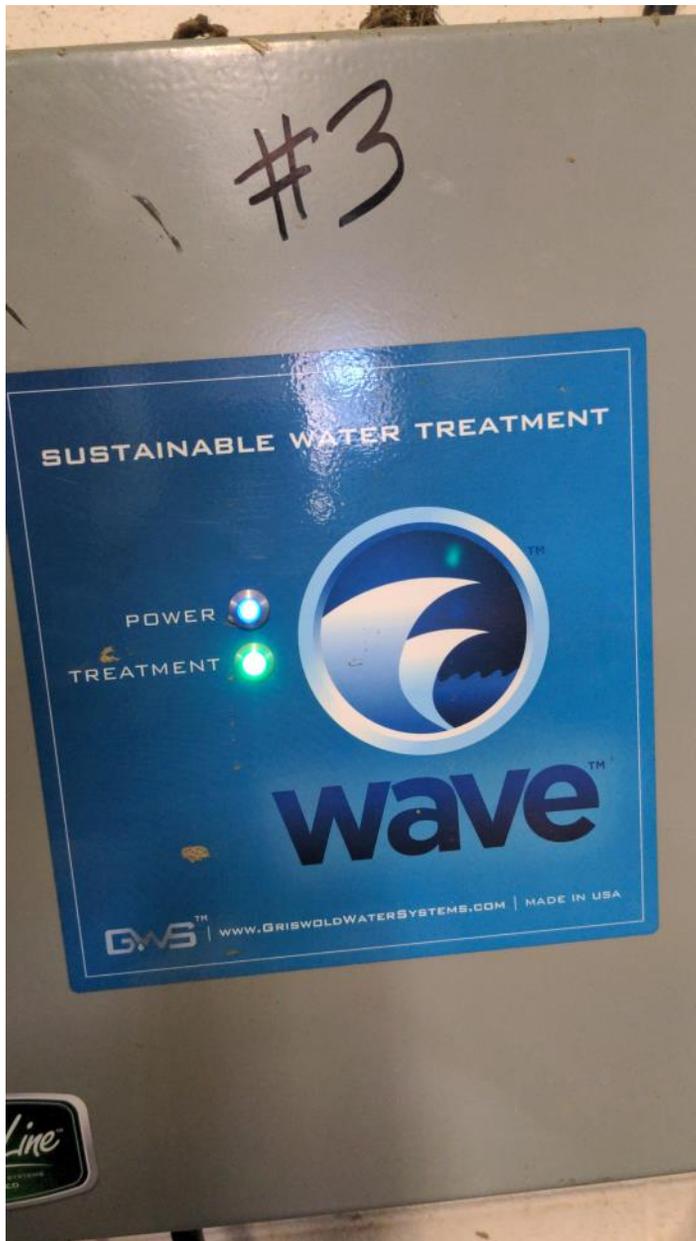






Water treatment

Thursday, January 27, 2022 10:57 AM



EC Architecture

Thursday, January 27, 2022 11:09 AM

Previous door widening











Existing to be widened











Remove remove existing doors and replace in kind or replace with new roll up door. Either are acceptable to the VA. Previous projects where chillers were introduced the height of the opening remain the same however they had to widen the opening instead of filling with brick, the contractor put in louver. This is acceptable to the VA as well.

Mechanical Scope

Thursday, January 27, 2022 11:18 AM

Remove existing chiller number three.

Disconnect chilled water supply downstream of control valve mv-ch-3. Replace control valve and all associated controls. Onikon flow meter to be repaired and made operational according to new sequence. Flow meter to be reused.





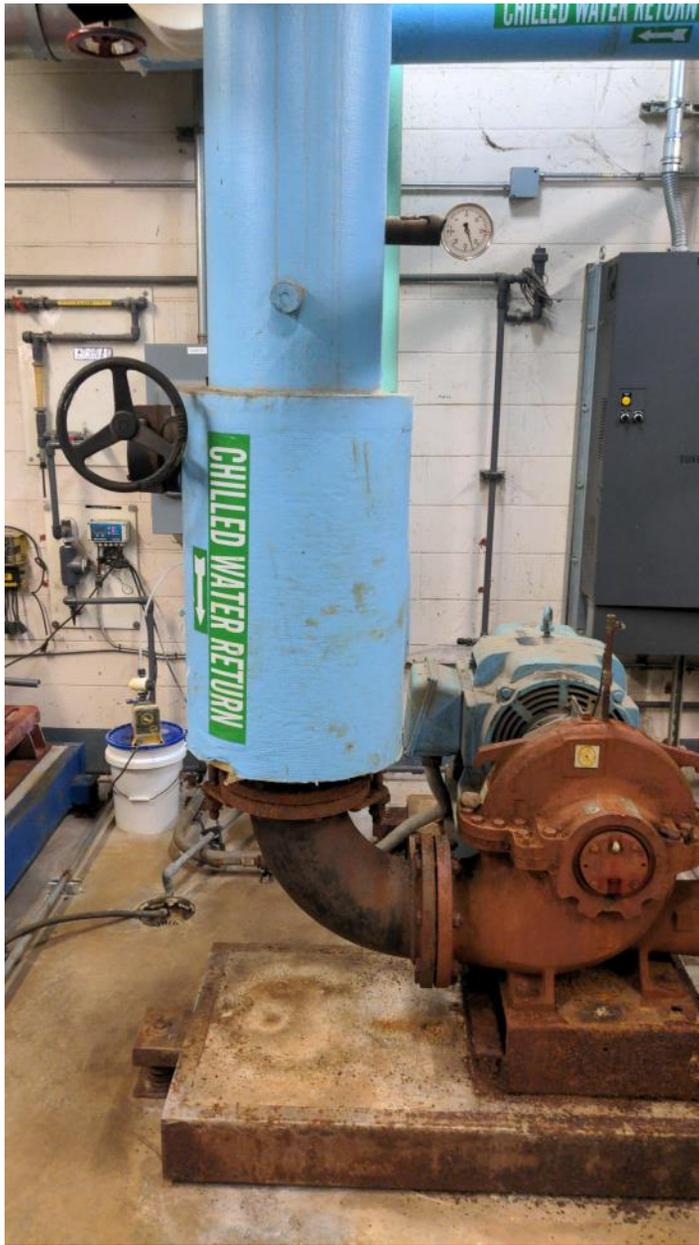
Remove chill water return from isolation valve the chiller 3. She'll want to return maybe isolated from or at rather the chill water loop on the roof. Replace isolation valve. Replace thermometers and gauges.





Remove pump number 6 and its entirety. Exercise valves to ensure blocking capability before beginning work. Remove existing isolation base and prepare for new installation. Isolation valves to remain. Replace gauges and repair pipe. Remove and replace variable speed drive associated with pump.







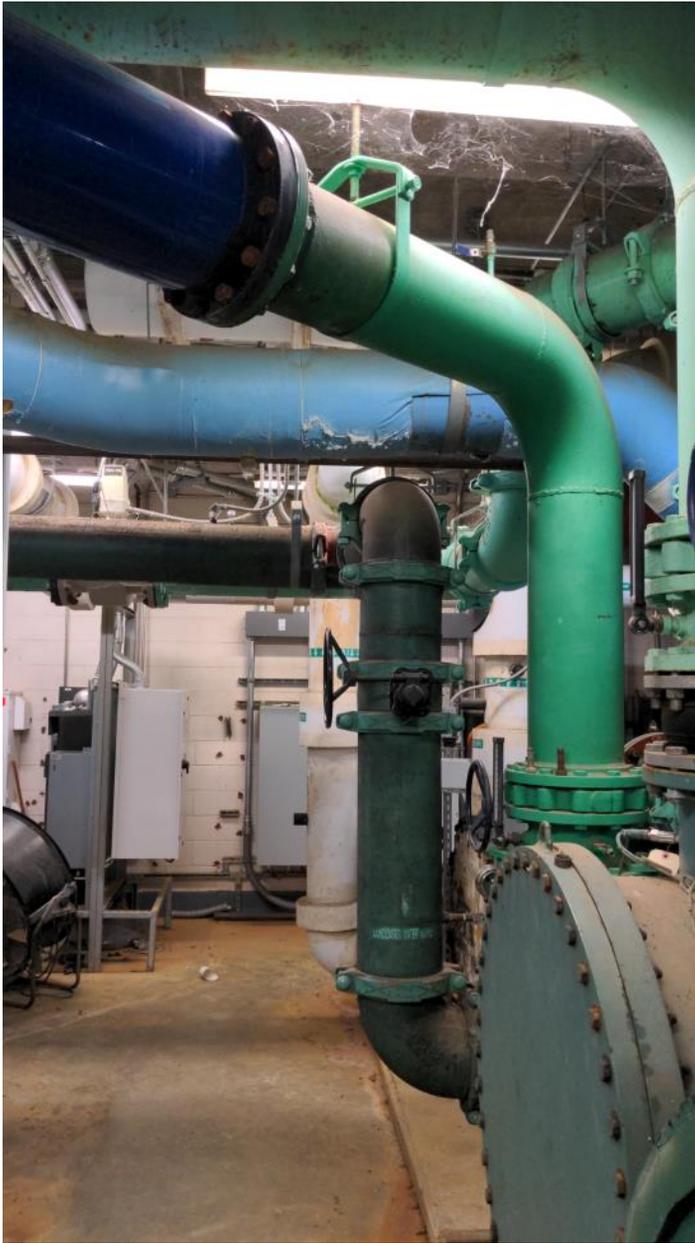


Remove abandoned chillwater treatment connected to inlet and outlet of pump number 6.



Condensed water pump number 7 to remain

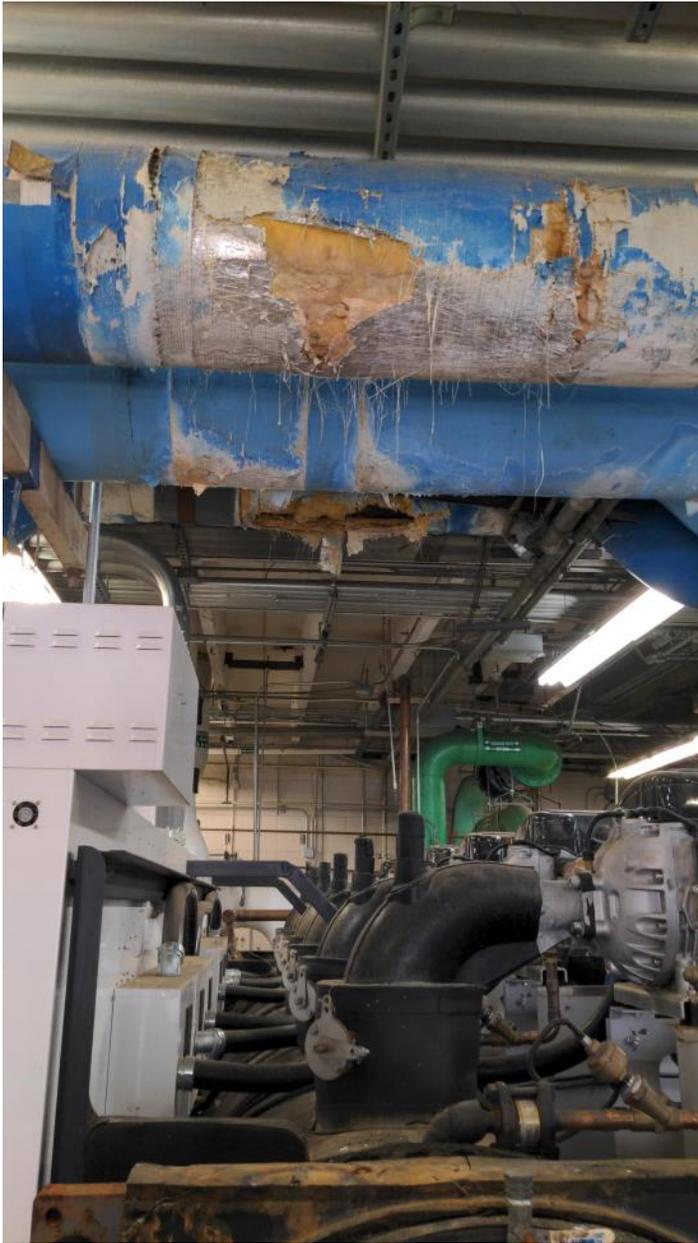
Green green line in line wave water treatment to remain reconfigure piping to connect to new chiller from flange at water treatment outlet.



Remove and replace insulation on chilled water piping in area of work and above chiller number two and as noted on drawings.











Controls

Thursday, January 27, 2022 11:35 AM

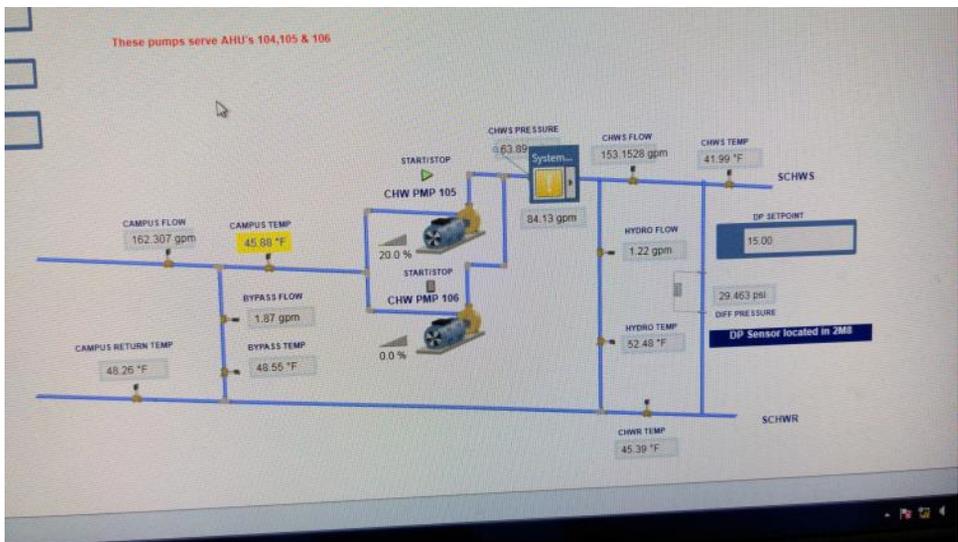
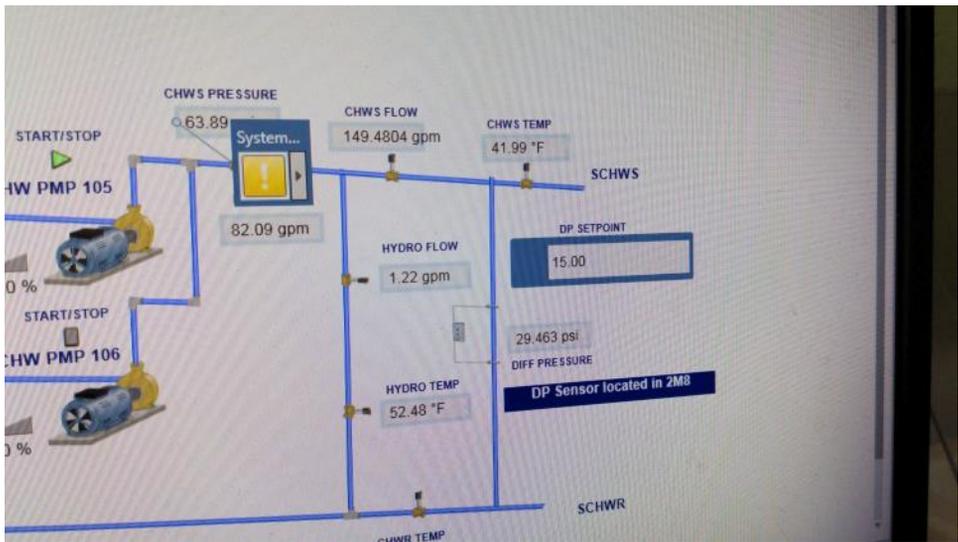
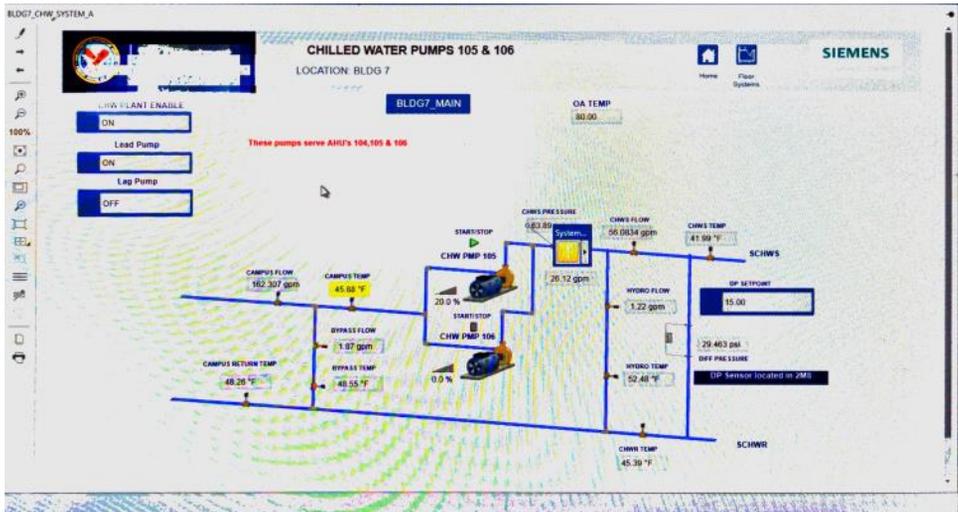
Chiller shall operate under its own control Siemens system shall have capability to overwrite set points start stop unit open close isolation valves and read all data from chiller control panel.

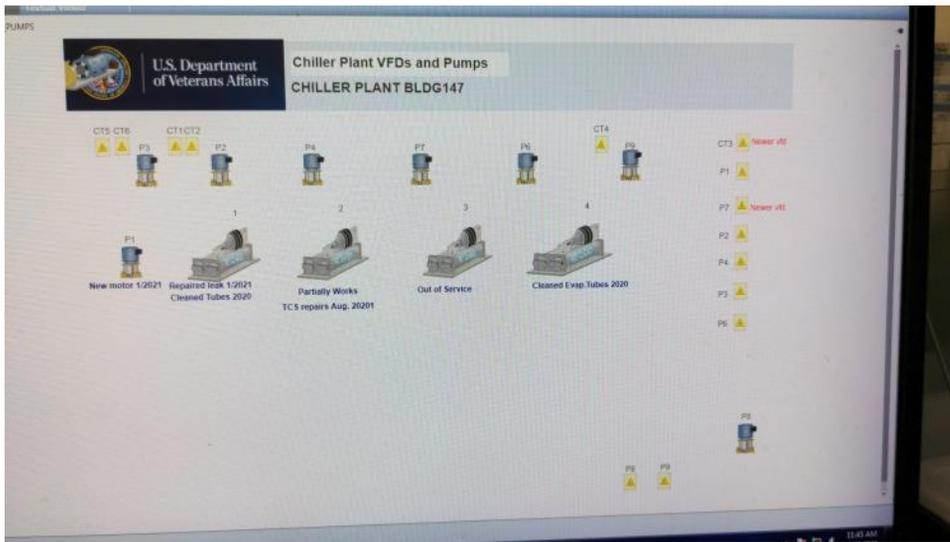
Provide new control panel for chiller remove existing.

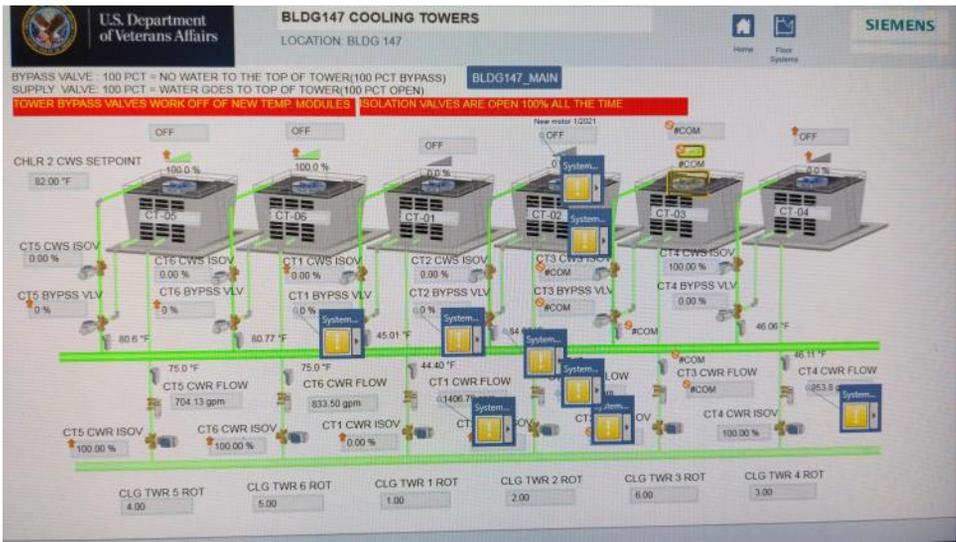


Reuse existing onicon flow meter on chill water supply. Repair and re-sequence.

Reuse existing flow meter on condenser water piping no work required.







BLDG147 COOLING TOWERS REPAIR AND MAINT.
LOCATION: BLDG 147

CT-01 CT-02	DATE:	REPAIRS:	MAINT:
	01/02/21	Replaced Motor on tower 2	
	03/20/21		Cleaned towers
	4/20/21 & 5/20/21		Cleaned tower Took 2 days to clean very dirty

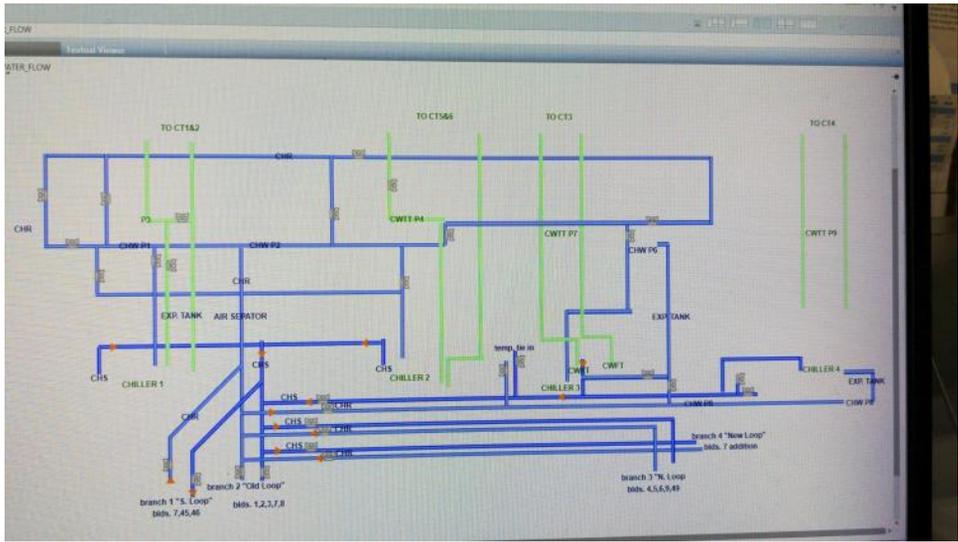
BLDG147 PLANT REQUIRED CALCULATIONS
LOCATION: BLDG 147

CHLR TONS ONLINE: 2000.00 Ton
CHLR LOAD TONS: 2695.77 Ton
CHLR REQUIRED: 3.00
CHWR AVER: 45.02 °F
CHWR VFD SPEED: 50.00 %
CHWP ADD TIMER: 0.00 s
CHWP SUB TIMER: 0.00 s
CHWP ADD SETPT: 900.00 s
CHWP SUB SETPT: 900.00 s
CHWS TEMP NORTH: 27.14 °F
CHWS TEMP SOUTH: 32. °F
CHWR TEMP NORTH: 47 °F
CHWR TEMP SOUTH: 50 °F

CHWP REQUIRED: 4.00
CLG TWR FANS REQUIRED: 5.00
CHWP1 ALARM: ON
CHWP2 ALARM: OFF
CHWP 6 ALARM: OFF
CHWP 8 ALARM: OFF

CHW SETPT: 44.00 °F
CHW OFFSET: 2.00 °F
ADD TIMER SETPT: 15.00 min
ADD TIMER: 0.00 min

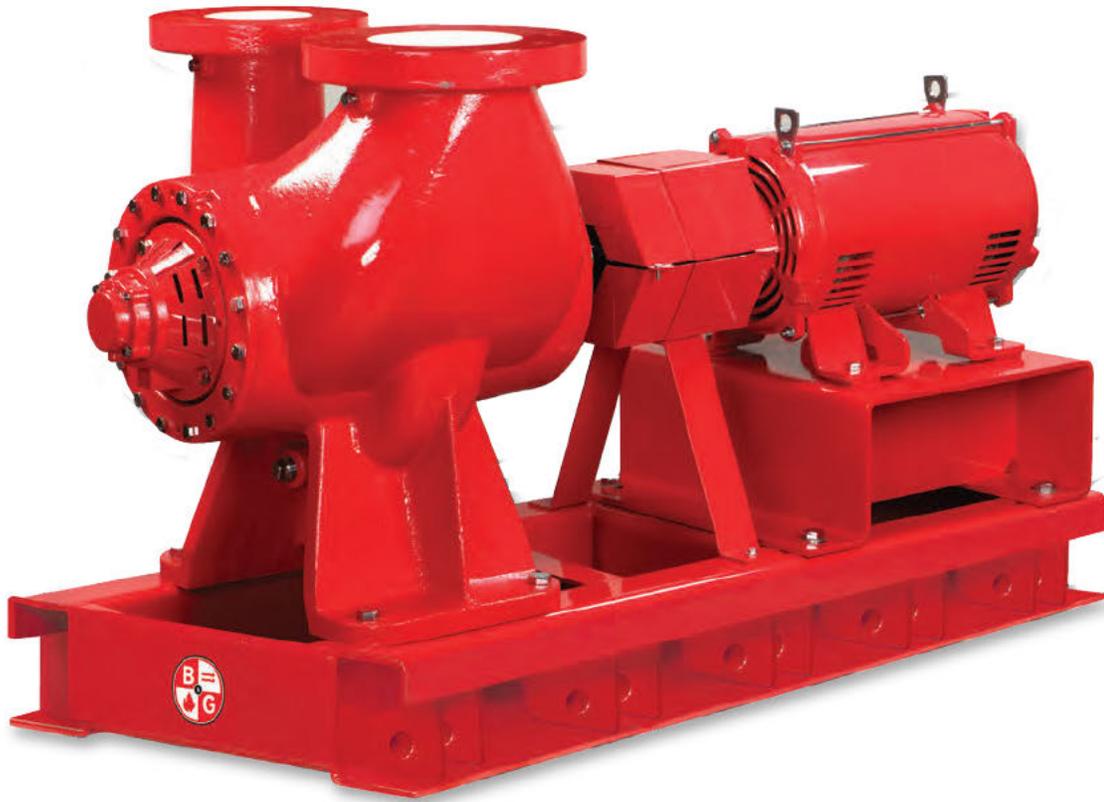
SUBTRACT SETPT: 15.00 min
SUBTRACT: -795.77 Ton
SUB TIMER SETPT: 15.00 min
SUBTRACT TIMER: 0.00 min
SUB VARIABLE: 100.00 Ton



Building 147 – Replace Chiller #3, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

Bid Documents Submittal

TAB C – Major Equipment Selections



VSX

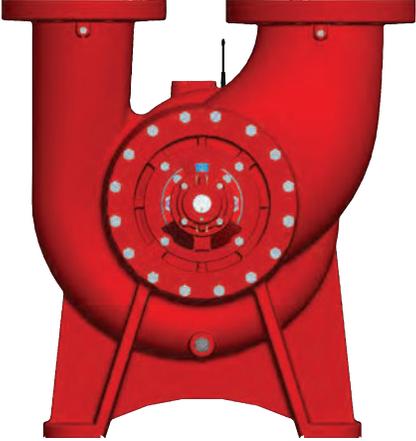
DOUBLE-SUCTION, SPLIT-CASE PUMPS



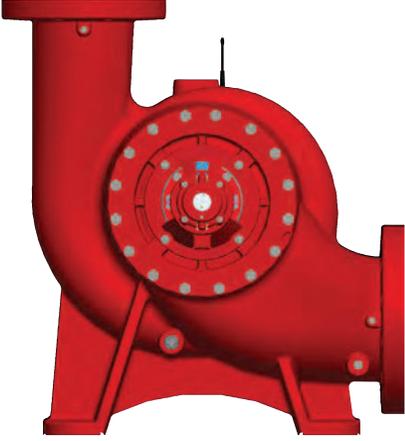
B-475E

 **Bell & Gossett**
a xylem brand

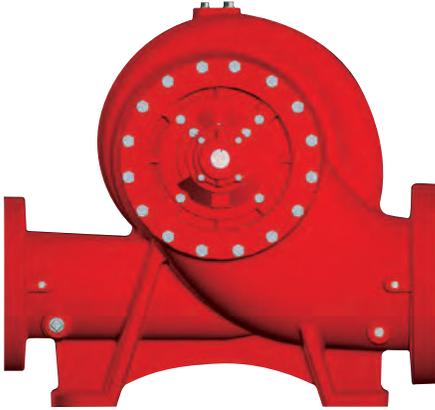
You asked for the impossible. So we built it.



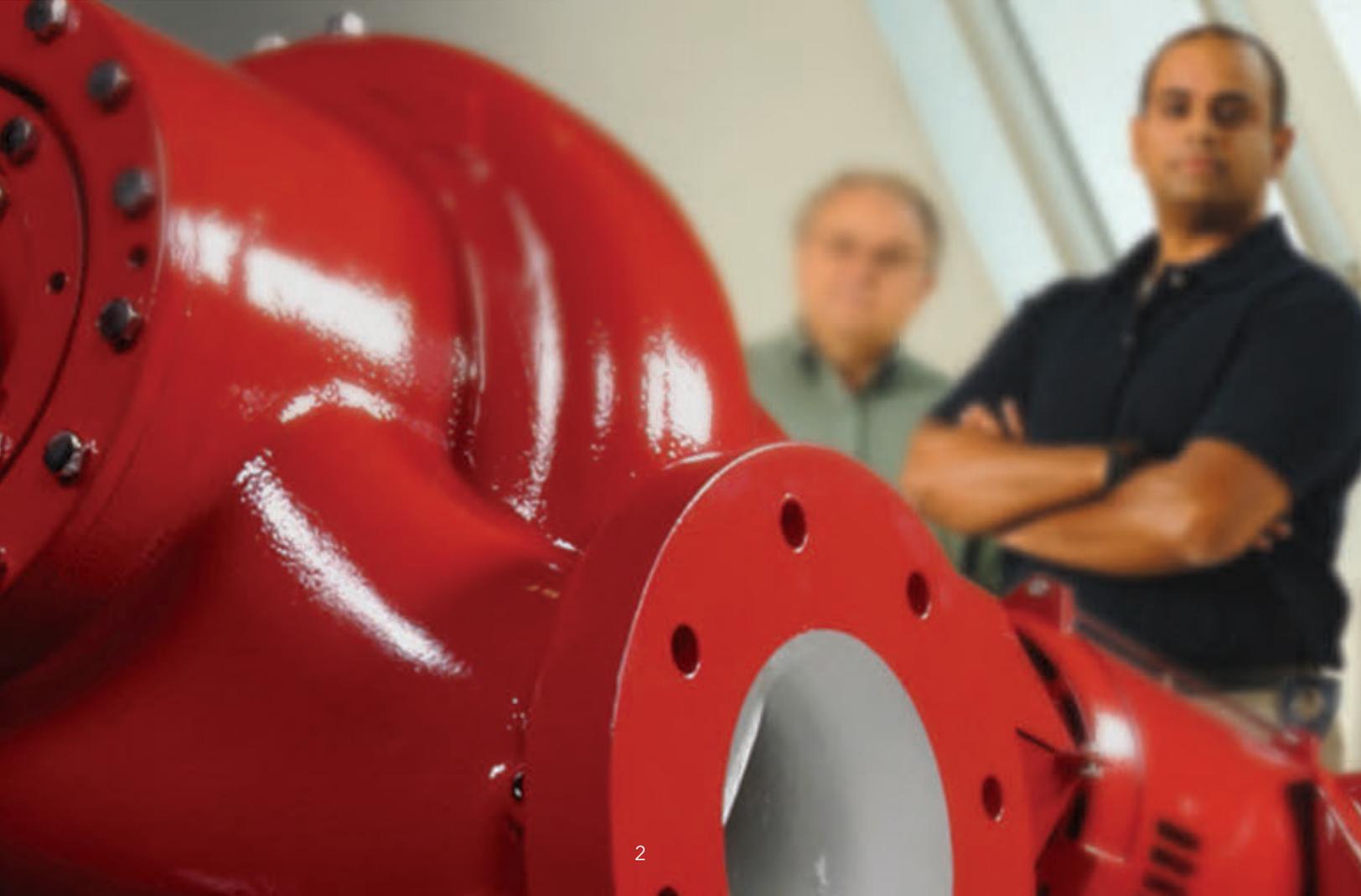
VSC



VSCS



VSH



At Bell & Gossett, we have over 100 years of experience in designing pumps and knowing how they work in complex systems. But when it comes to the challenges specifiers and contractors face every day in the field, no one knows pumps better than our customers. So when we began designing our new VSX pump, we knew we needed to talk to the experts. We needed to talk to you.

We took a revolutionary approach to design. We listened to you.

You told us what you wanted: a truly innovative pump. One that saves space, is highly efficient and installs easily. You asked for total reliability and enough flexibility to suit nearly any application.

Our new VSX platform retains the innovative and time-proven space saving VSC® design. The VSC model has reduced your pump room footprint by up to 40% when compared to traditional split-case and vertical inline pumps. The time-proven VSC design now becomes even better within the new VSX platform. The high-efficiency VSX hydraulic design and robust construction deliver proven Bell & Gossett reliability, and the unique platform design provides complete installation versatility. Simply select your hydraulic requirements and then pick any one of three different suction and discharge flange orientations that best suit your installation requirements. In short, VSX is everything you asked for, and then some.

Here's what you told us.

"Mechanical room space keeps shrinking. I need one pump that can provide me with the flexibility to meet my space challenges."

"We want a pump that was engineered using today's technology, not something that was designed 40 years ago and adapted for today's uses."

"Hydraulic performance and efficiency are the key points that we look at when selecting the right pump to match up with the chillers and towers. Why can't we have a pump that runs at 1750 RPM but acts like it is at 1150 RPM to match up with our real-world application requirements?"

"Serviceability and minimal downtime are important to us in our operations. Simplicity of maintenance is absolutely our number one issue."



"Labor is the only item I can control, so installation time is critical to me."

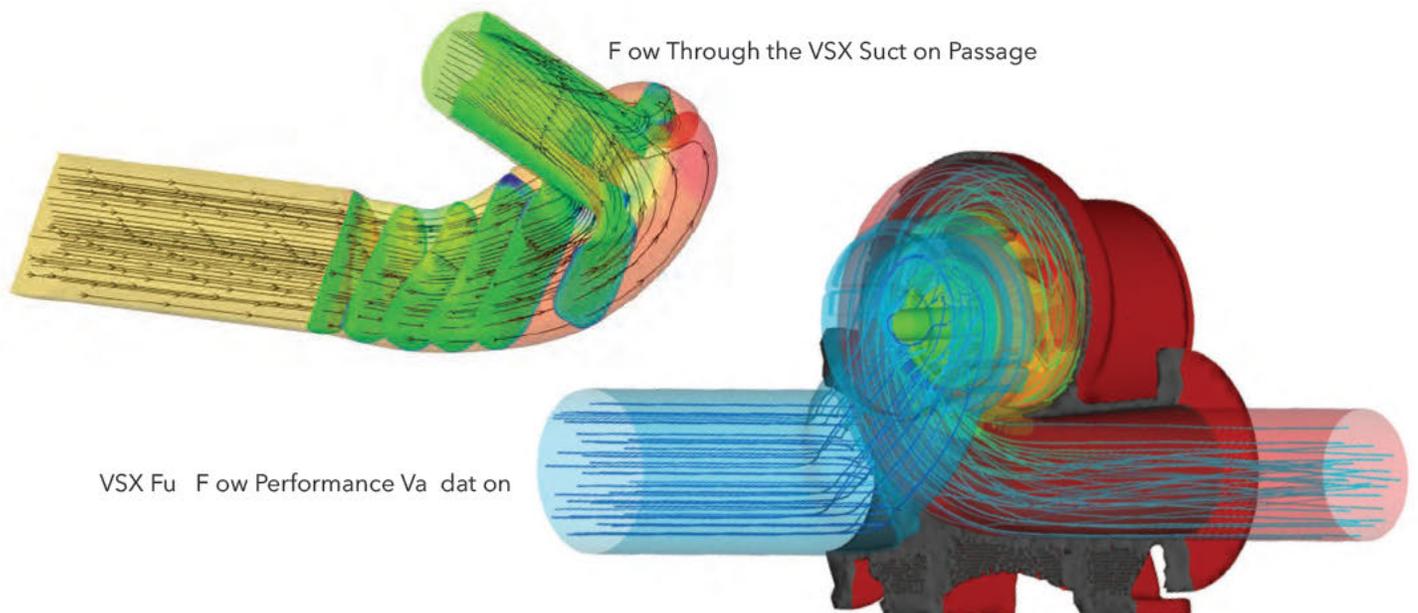
You asked for a completely new hydraulic design. We gave it to you.

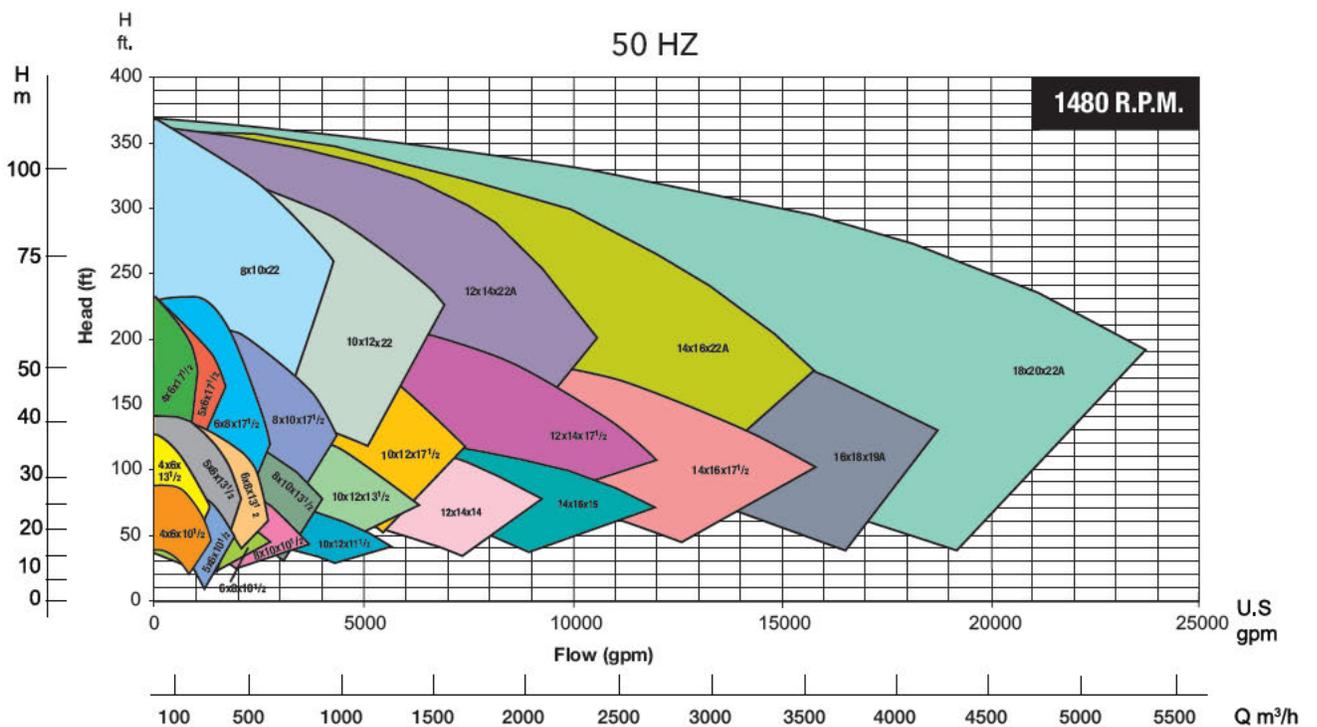
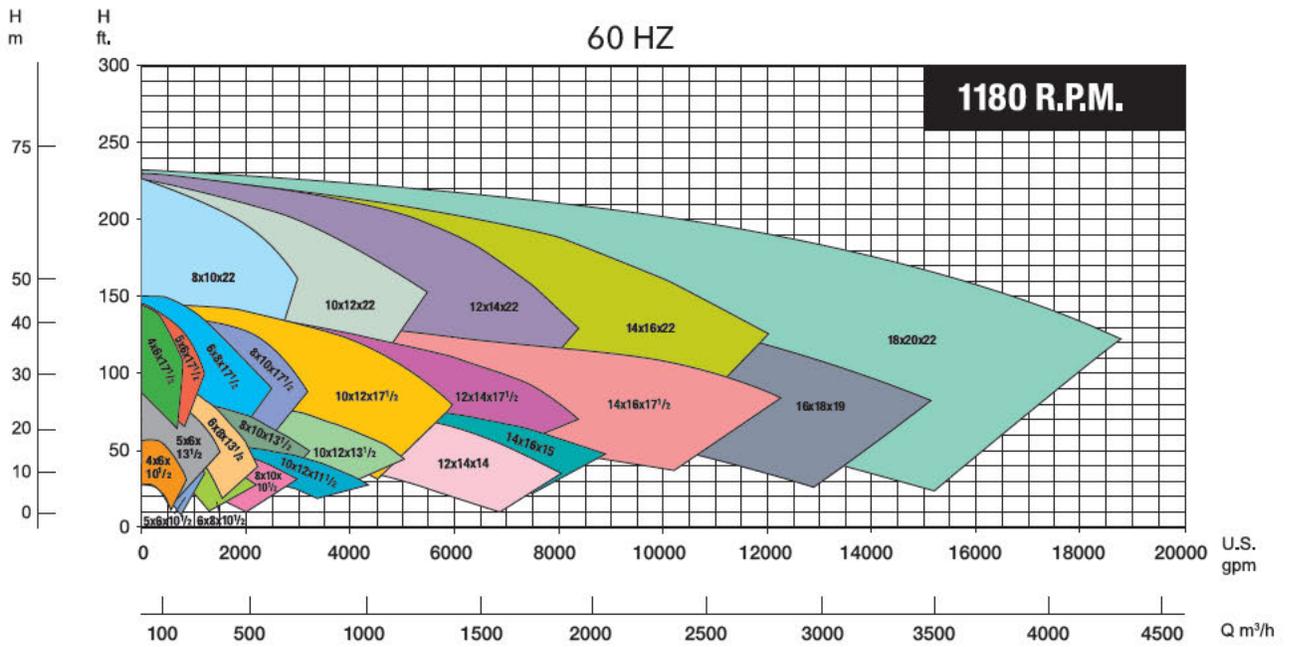
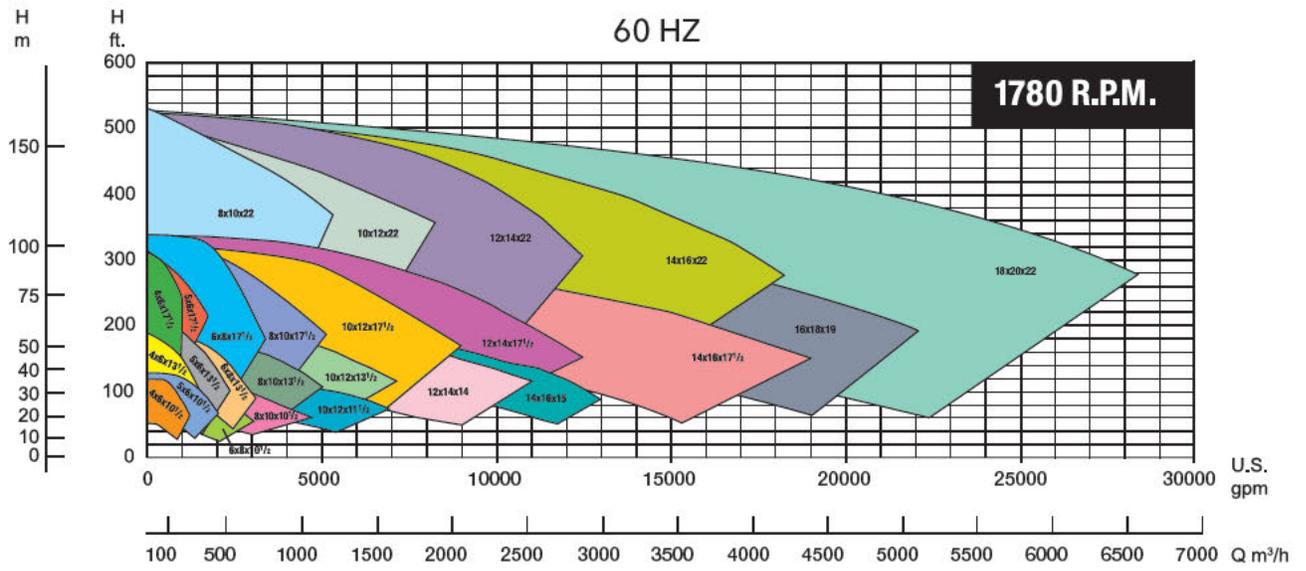
Until now, pump flow in traditional split-case pumps was maximized in the industry by pushing extreme velocities through the same 50-year-old pump volute.

Today, hydraulic design begins with the end requirements in mind. When we engineered the VSX platform, we started by identifying your chiller, cooling tower and general pumping requirements. We matched the best efficiency points (BEP) to common chiller and tower sizes, and normal industrially-specified flows and head conditions.

- Hydraulic coverage to meet all your design requirements
- Flows from 500 to 28,000 GPM / 114 to 6400 m³/h
- Heads from 30 to 530 feet / 9 to 161 meters
- Working pressures of 175 and 300 psig / 12 and 20 Bar
- ANSI flange ratings of 125# and 250#
- Temperature 0 to 300°F / -17 to 148°C

We took our tried-and-true design methods and digitized them by simulating the pumps using Computational Fluid Dynamics (CFD). CFD is no simple task; it takes a bank of 18 computers up to 30 hours to solve equations for just one hydraulic analysis. By using this technique, you can actually see the flows and pressures like never before. Where our engineers once relied solely on years of experience, CFD helps out by letting them see exactly what's going on inside the pump. They can analyze several different design ideas rapidly to bring the optimal design to the table for every pump, whether it is in the VSC, VSH or VSCS flange configuration.



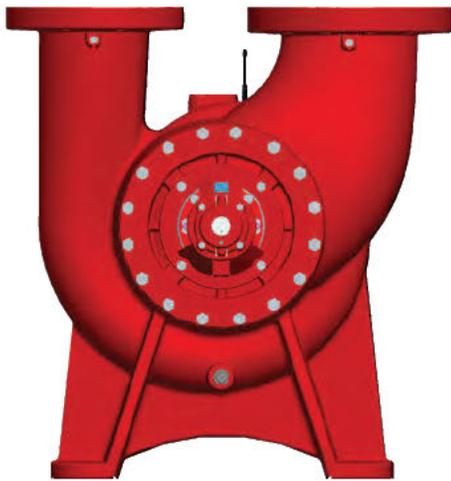


You said you needed one pump that does it all.

When it comes to the VSX platform, our revolutionary approach to hydraulic design is just the beginning. This one-of-a-kind pump has what it takes to make your job easier, simplifying installation and minimizing service.

Multiple suction and discharge flange orientations

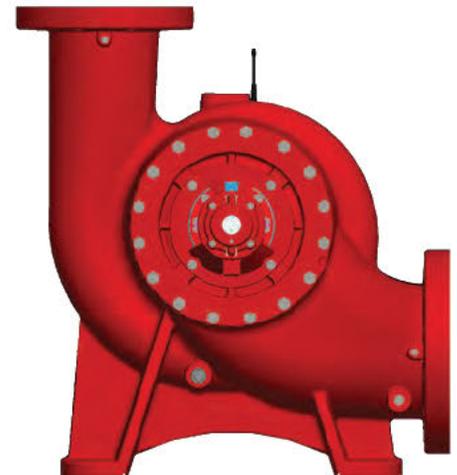
Only the VSX platform offers so many installation options, thanks to its revolutionary design. Using CFD technology, we can deliver identical performance in any flange configuration. The result is that you can maximize your piping possibilities and meet a broad hydraulic range for chillers, towers, distributive and general pumping requirements.



VSC®



VSH™



VSCS®

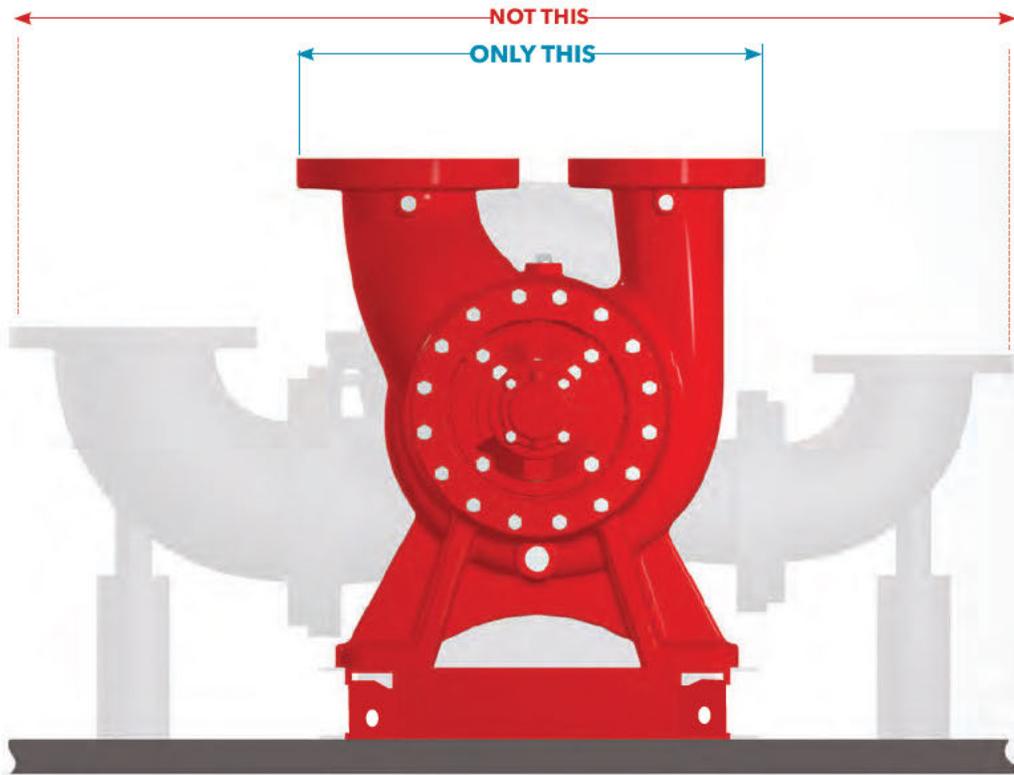
Vertical Static Flange Loading

The vertical split-case volute design of the VSX provides optimum nozzle loading capability that others just can't match. In the vertical flange configuration (VSC model), the pump flanges easily support the weight of heavy piping directly on its nozzles. The pump flanges sit directly under the load - right where it's needed the most - without impairing pump operation.

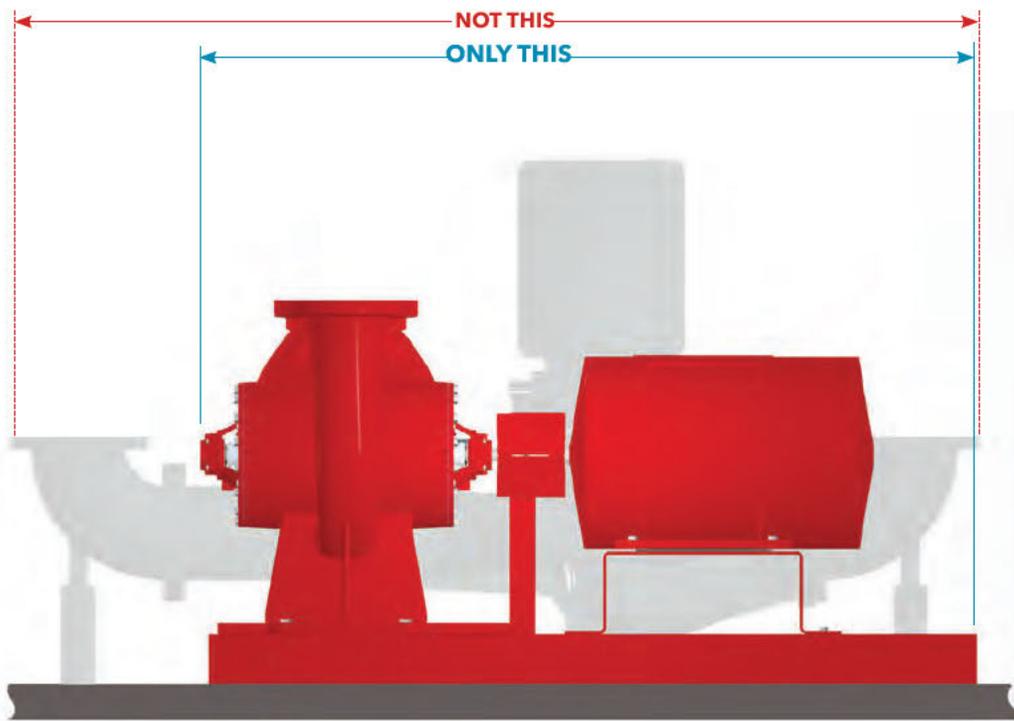


Space-Saving Footprint

VSX lets you reduce the overall equipment footprint by up to 40 percent over horizontal double-suction and large inline pumps with the VSC configuration. The VSX pump optimizes the advantages of vertical piping applications by eliminating the space-robbing elbows, protruding accessories and pipe supports.



VSC Versus Traditional Horizontal Split-Case Pump



VSC Versus Traditional Vertical Inline Pump

You were looking for simplicity. So we found it

One of the most remarkable things about the new VSX pump is how easy it is to install and maintain. Based on your input, we looked at every component of our pumps with the goal of streamlining and simplifying. Our goal: to make your life easier.



A Simplified Service

No rigging or heavy-duty material handling equipment is necessary to gain access to the bearing, mechanical seals or shaft sleeves. Unlike older horizontal double-suction, split-case and larger vertical inline pumps, there are no heavy top casings or motors to lift off, saving cost, reducing risk and providing easy access to the rotating assembly.

The VSX platform makes bearing, mechanical seal and shaft sleeve inspection easy. The VSX design makes these components readily accessible from both sides of the pump, so that you can service from the inboard* or outboard side without disturbing the piping or the motor. You can also replace just one bearing, mechanical seal or sleeve without disturbing the other side.

B Maintenance-Free Bearings

Our maintenance-free bearings eliminate the need for regular maintenance, documentation logs, over-greasing problems and the risk of mixing greases that can cause early failure. This design not only reduces maintenance time and costs, but also helps extend pump life.

C New One-Piece Unitized Seal

Bell & Gossett's new one-piece unitized seal eliminates multiple seal components and simplifies replacement. Because it uses a one-piece elastomeric bellows, it has fewer parts than competitive seals, resulting in significantly fewer installation errors.

D Groutless Base Plate

Another industry first! Our new groutless baseplate design saves valuable time and money to speed installation. Advanced finite element analysis and design provides a modern state-of-the-art baseplate that is rock-solid.

E Alignment-Friendly Coupling[†]

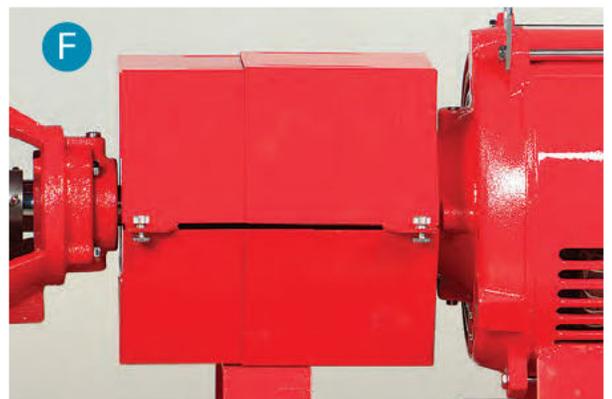
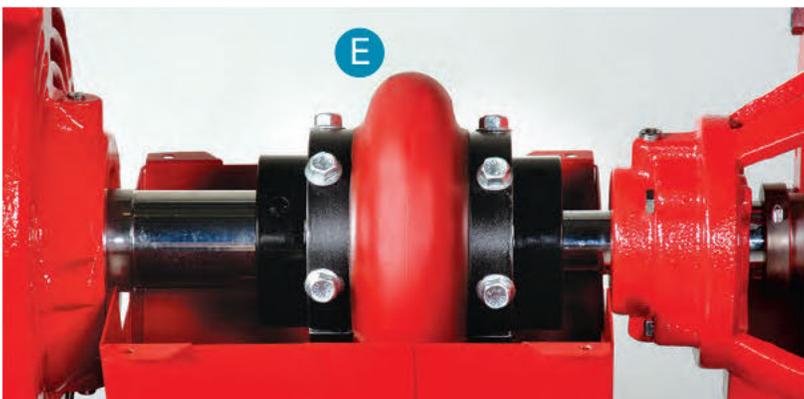
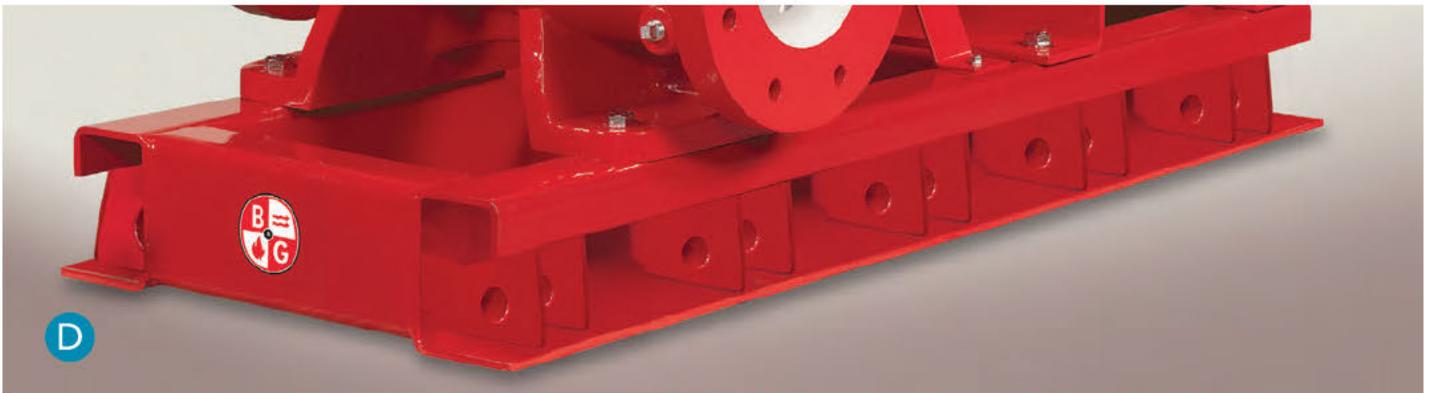
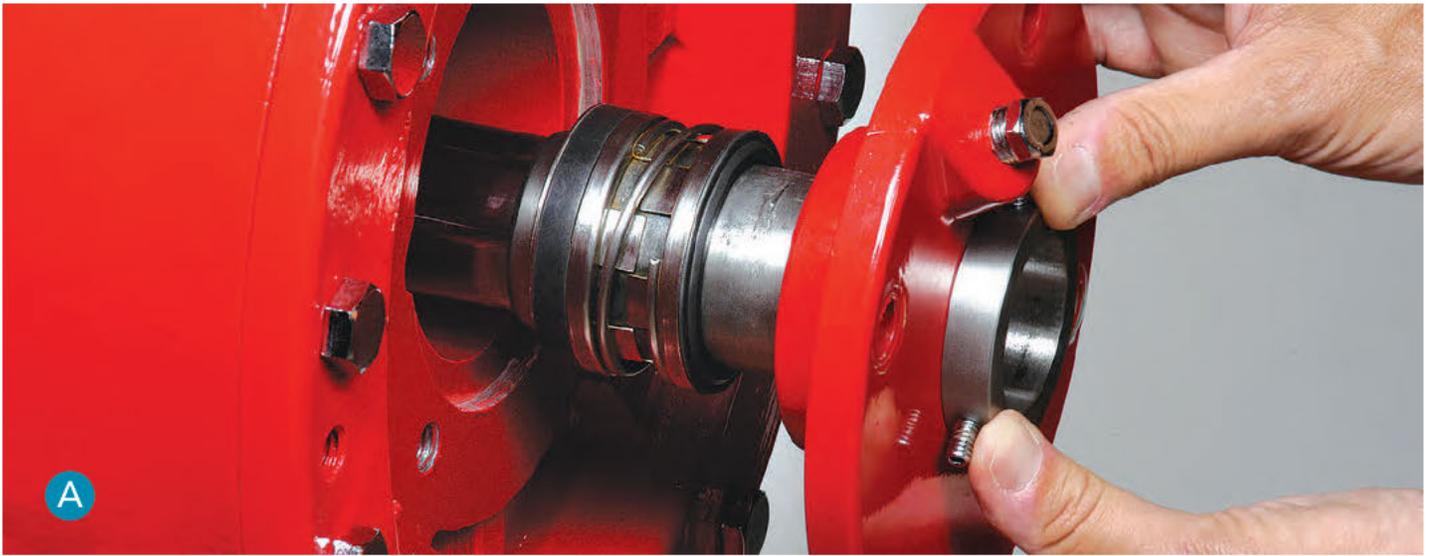
Reduce installation time further by maximizing your alignment protection. Designed with finite element analysis to maximize performance and overall life. Easy split element design avoids movement of hubs during inspection or replacement. Moreover, an ANSI/OSHA coupling guard surrounds the coupling to protect personnel.

F ANSI/OSHA Coupling Guard

ANSI B15.1 and OSHA 1910.219 compliant coupling guard shields the coupler during operation. The VSX coupler guard is dual designed and contains viewing windows for inspection of the coupling. No more than .25 inch opening in the guard around the rotating assembly is visible.

*Requires optional spacer coupling

† Up to 1000 HP at 750 RPM



Service and support from the most trusted name in the industry – Bell & Gossett.

The Bell & Gossett name has always stood for uncompromising quality and dependability. That's evident in the way every one of our centrifugal pumps is built and backed by our outstanding customer service and support team.

Your local Bell & Gossett representative is available any time and is an experienced professional with a wealth of technical expertise. In addition to expert system and product application assistance and a wide product inventory warehoused locally, we offer ESP-Systemwize software selection program.



ESP-Systemwize is a Bell & Gossett web-based software that helps you design HVAC systems accurately, effectively and very quickly. You get fast, precise equipment selection, pump performance curves and equipment schedules, submittals, specifications and more.

ESP-Systemwize includes:

- Centrifugal Pumps
- Air/Dirt Separators
- Heat Exchangers
- Pump Suction Diffuser and Triple Duty Valve
- Expansion Tanks
- PIC Valves

With more than 100 years' experience as an industry leader, we know how to design, build, and support centrifugal pumps. Our hallmarks are excellence and dependability.



An award-winning pump

The VSX pump continues to receive major awards:

- Consulting-Specifying Engineer Product of the Year
Silver Award in the HVAC Category
- AHR Innovation Award
Honorable Mention in the Heating Category
- India Water Digest Water Award
Best Water R&D and Technological Breakthrough
- HPAC Engineering Readers' Choice Award
- Plumbing Engineer – One of the Top Ten Products



**Bell & Gossett. We make the difference.
By design.**

Pump us for information

Our comprehensive Web site (www.bellgossett.com) makes it easier for you to find, specify and understand the VSX pump, fluid handling systems, parts, specifications and applications.

Xylem |'zīləm|

- 1) The tissue in plants that brings water upward from the roots;
- 2) a leading global water technology company.

We're a global team unified in a common purpose: creating advanced technology solutions to the world's water challenges. Developing new technologies that will improve the way water is used, conserved, and re-used in the future is central to our work. Our products and services move, treat, analyze, monitor and return water to the environment, in public utility, industrial, residential and commercial building services, and agricultural settings. With its October 2016 acquisition of Sensus, Xylem added smart metering, network technologies and advanced data analytics for water, gas and electric utilities to its portfolio of solutions. In more than 150 countries, we have strong, long-standing relationships with customers who know us for our powerful combination of leading product brands and applications expertise with a strong focus on developing comprehensive, sustainable solutions.

For more information on how Xylem can help you, go to www.xylem.com

We value your feedback. Please take our 3 question survey at bellgossett.com/survey to let us know how we are doing.

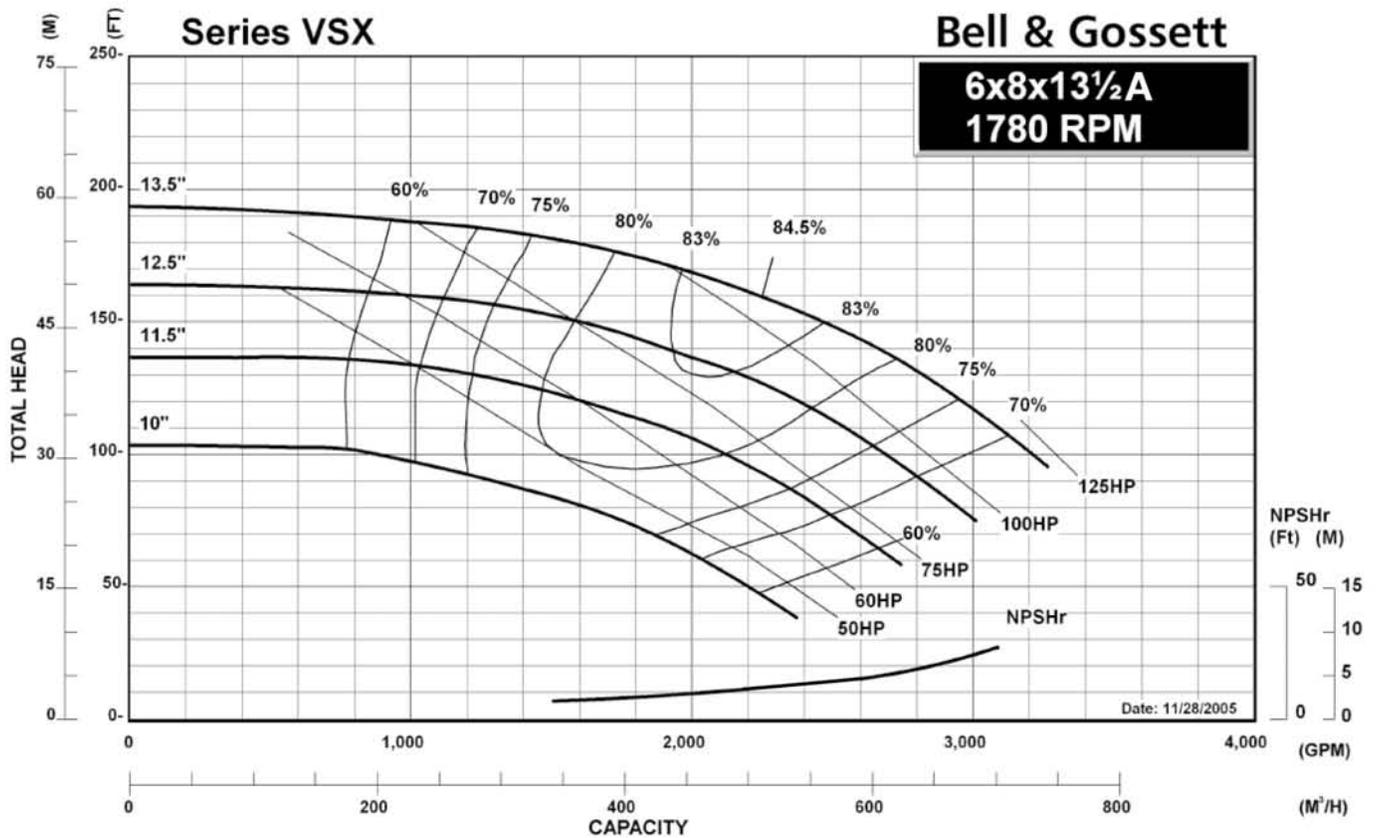
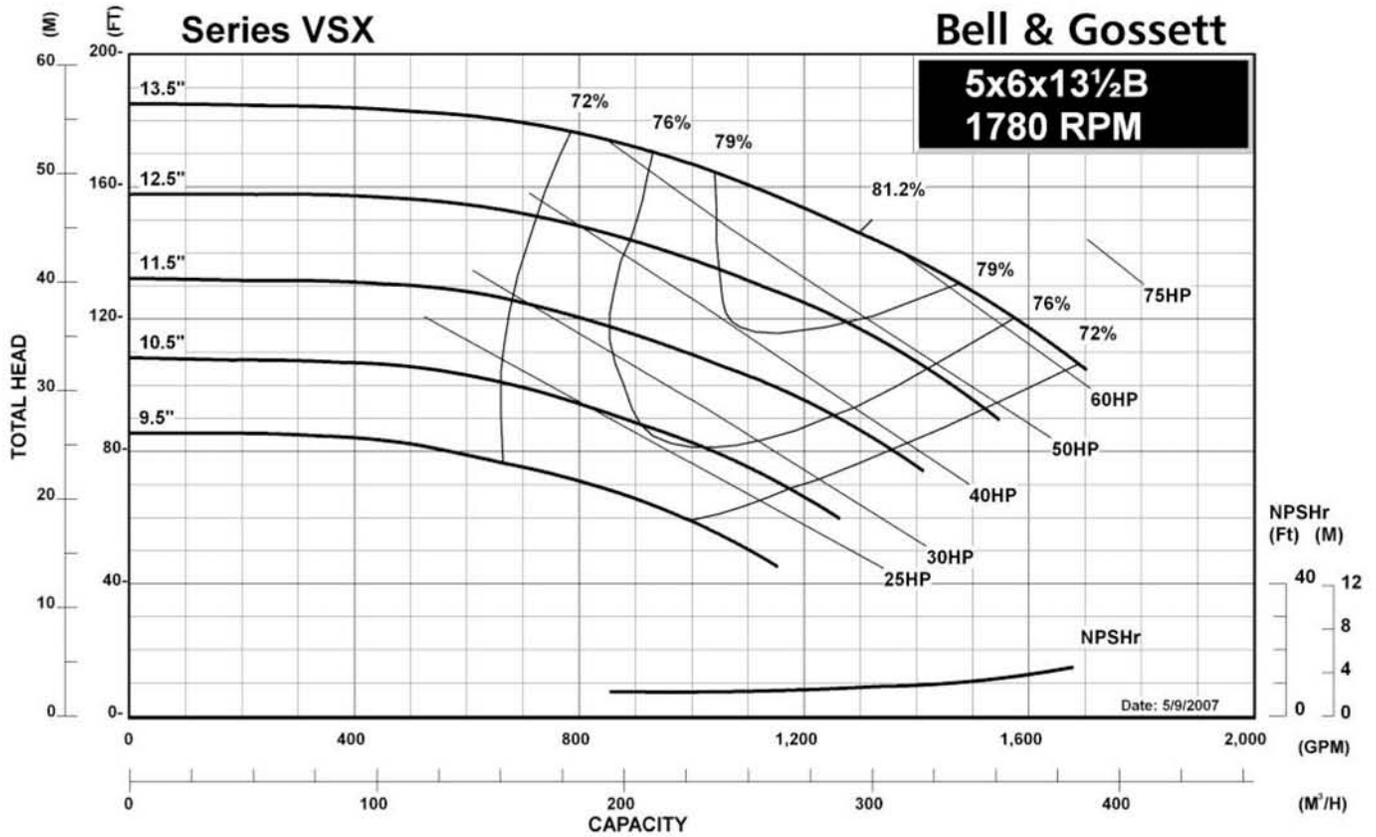


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SERIES VSX

1780 RPM PUMP CURVES





Performance Report
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

Prepared By: William Harris



19DV-G44G444425B5

Chiller	
Starter / VFD	Unit Mounted VFD (Harmonic Mitigation)
Refrigerant Type	R-1233zd(E)
Operation Type	Cooling
Manufacturing Source	Charlotte, NC USA

Configuration	Evaporator	Condenser
Size	G44	G44
Waterbox Type	Marine Waterbox, 150 psi	Marine Waterbox, 150 psi
Passes	2	2
Nozzle Arrangement	C - 2 Pass, Inlet on 2nd Stage End	R - 2 Pass, Inlet on 2nd Stage End
Tubing	3/4" 0.025" Copper Cross Hatched Evaporator Tubes	3/4" 0.025" Copper Cross Hatched Condenser Tubes
Fluid Type	Fresh Water	Fresh Water
Fouling Factor	0.000100 (hr-sqft-F)/BTU	0.000250 (hr-sqft-F)/BTU
Tube Quantity	549	702

Compressor	
Size	442
Weights	
Total Rigging Weight	34995 lb
Total Operating Weight	41679 lb
Refrigerant Weight	1637 lb
Motor & Motor Controller	
Size	NBH
Line Voltage-Phase-Hertz	460(V)-3-60(Hz)
VFD Code	5
VFD Size	D
VFD Lug Size	4x4 / 0 - 500 kcmil ABB KT7X1200 - 3
Flow Controls	
Condenser Float Valve	HN
Economizer Float Valve	LN
Envelope Stability Control	Installed
Economizer Bypass	Not Installed
Dimensions	
Length	213 in
Width	108 in
Height	120 in



Performance Report
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

Prepared By: William Harris

Full Load	
Full Load Performance	Percent Load
Chiller	
Percent Load	100.00
Chiller Capacity	650.0 Tons
Chiller Power	334.9 kW
Chiller Efficiency	0.5152 kW/TonR
Chiller COP _R	6.827 kW/kW
IPLV _{IP}	0.2906 kW/TonR
IPLV, COP _R	12.10
NPLV _{IP}	0.3096 kW/TonR
NPLV, COP _R	11.36
Evaporator	
Entering Temperature	52.00 °F
Leaving Temperature	42.00 °F
Flow Rate	1557 gpm
Pressure Drop	17.4 ft H ₂ O
Fouling Temp Adj	0.18 °F
Condenser	
Heat Rejection	8943 MBH
Leaving Temperature	94.30 °F
Entering Temperature	85.00 °F
Flow Rate	1933 gpm
Pressure Drop	17.8 ft H ₂ O
Fouling Temp Adj	0.50 °F
Motor & Motor Controller	
Motor Rated Load KW	323 kW
Motor Amps	488
Motor OLTA	527
Chiller Line Amps	444
Chiller Inrush Amps	444
Max Fuse/ CB Amps	800
Min Circuit Ampacity	555
Heat Exchangers	
Evaporator Tube Vel	5.53 ft/s
Cond Tube Vel	5.92 ft/s



Certified in accordance with the AHRI Water-Cooled Water-Chilling and Heat Pump Water-Heating Packages Certification Program, which is based on AHRI Standard 550/590 (I-P) and AHRI Standard 551/591 (SI). Certified units may be found in the AHRI Directory at www.ahridirectory.org.

<http://www.ahrinet.org/WCCL.aspx>

This unit complies with the efficiency requirements of ASHRAE 90.1 2013/2016 Path B.

1 Point LEED EAC4 can be earned when selecting the 19DV AquaEdge



Performance Report
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

Prepared By: William Harris

IPLV.IP				
	Percent Load			
Chiller				
Percent Load	100.00	75.00	50.00	25.00
Chiller Capacity	650.0 Tons	487.5 Tons	325.0 Tons	162.5 Tons
Chiller Power	319.7 kW	176.6 kW	80.79 kW	43.20 kW
Chiller Efficiency	0.4919 kW/TonR	0.3622 kW/TonR	0.2486 kW/TonR	0.2658 kW/TonR
Chiller COP _R	7.150 kW/kW	9.711 kW/kW	14.15 kW/kW	13.23 kW/kW
IPLV.IP	0.2906 kW/TonR	NA	NA	NA
IPLV, COP _R	12.10	NA	NA	NA
NPLV.IP	0.3096 kW/TonR	NA	NA	NA
NPLV, COP _R	11.36	NA	NA	NA
Evaporator				
Entering Temperature	54.00 °F	51.50 °F	49.00 °F	46.50 °F
Leaving Temperature	44.00 °F	44.00 °F	44.00 °F	44.00 °F
Flow Rate	1558 gpm	1558 gpm	1558 gpm	1558 gpm
Pressure Drop	17.4 ft H ₂ O			
Fouling Temp Adj	0.17 °F	0.13 °F	0.10 °F	0.06 °F
Condenser				
Heat Rejection	8891 MBH	6452 MBH	4176 MBH	2097 MBH
Leaving Temperature	94.30 °F	81.74 °F	69.35 °F	67.19 °F
Entering Temperature	85.00 °F	75.00 °F	65.00 °F	65.00 °F
Flow Rate	1922 gpm	1922 gpm	1922 gpm	1922 gpm
Pressure Drop	17.6 ft H ₂ O	17.7 ft H ₂ O	17.6 ft H ₂ O	17.6 ft H ₂ O
Fouling Temp Adj	0.50 °F	0.36 °F	0.24 °F	0.12 °F
Motor & Motor Controller				
Motor Rated Load KW	308 kW	168 kW	76 kW	40 kW
Motor Amps	470	313	223	189
Motor OLTA	508	NA	NA	NA
Chiller Line Amps	424	237	110	61
Chiller Inrush Amps	424	NA	NA	NA
Max Fuse/ CB Amps	800	NA	NA	NA
Min Circuit Ampacity	530	NA	NA	NA
Heat Exchangers				
Evaporator Tube Vel	5.53 ft/s	5.53 ft/s	5.53 ft/s	5.53 ft/s
Cond Tube Vel	5.88 ft/s	5.89 ft/s	5.90 ft/s	5.90 ft/s



Certified in accordance with the AHRI Water-Cooled Water-Chilling and Heat Pump Water-Heating Packages Certification Program, which is based on AHRI Standard 550/590 (I-P) and AHRI Standard 551/591 (SI). Certified units may be found in the AHRI Directory at www.ahrirectory.org.

<http://www.ahrinet.org/WCCL.aspx>

This unit complies with the efficiency requirements of ASHRAE 90.1 2013/2016 Path B.

1 Point LEED EAC4 can be earned when selecting the 19DV AquaEdge



Performance Report
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

Prepared By: William Harris

NPLV.IP				
	Percent Load			
Chiller				
Percent Load	100.00	75.00	50.00	25.00
Chiller Capacity	650.0 Tons	487.5 Tons	325.0 Tons	162.5 Tons
Chiller Power	334.9 kW	186.0 kW	86.67 kW	46.23 kW
Chiller Efficiency	0.5152 kW/TonR	0.3815 kW/TonR	0.2667 kW/TonR	0.2845 kW/TonR
Chiller COP _R	6.827 kW/kW	9.218 kW/kW	13.19 kW/kW	12.36 kW/kW
IPLV.IP	0.2906 kW/TonR	NA	NA	NA
IPLV, COP _R	12.10	NA	NA	NA
NPLV.IP	0.3096 kW/TonR	NA	NA	NA
NPLV, COP _R	11.36	NA	NA	NA
Evaporator				
Entering Temperature	52.00 °F	49.50 °F	46.99 °F	44.50 °F
Leaving Temperature	42.00 °F	42.00 °F	42.00 °F	42.00 °F
Flow Rate	1557 gpm	1557 gpm	1557 gpm	1557 gpm
Pressure Drop	17.4 ft H ₂ O			
Fouling Temp Adj	0.18 °F	0.13 °F	0.10 °F	0.07 °F
Condenser				
Heat Rejection	8943 MBH	6485 MBH	4196 MBH	2108 MBH
Leaving Temperature	94.30 °F	81.73 °F	69.35 °F	67.18 °F
Entering Temperature	85.00 °F	75.00 °F	65.00 °F	65.00 °F
Flow Rate	1933 gpm	1933 gpm	1933 gpm	1933 gpm
Pressure Drop	17.8 ft H ₂ O	17.9 ft H ₂ O	17.8 ft H ₂ O	17.8 ft H ₂ O
Fouling Temp Adj	0.50 °F	0.36 °F	0.24 °F	0.12 °F
Motor & Motor Controller				
Motor Rated Load KW	323 kW	178 kW	82 kW	43 kW
Motor Amps	488	320	222	185
Motor OLTA	527	NA	NA	NA
Chiller Line Amps	444	250	118	65
Chiller Inrush Amps	444	NA	NA	NA
Max Fuse/ CB Amps	800	NA	NA	NA
Min Circuit Ampacity	555	NA	NA	NA
Heat Exchangers				
Evaporator Tube Vel	5.53 ft/s	5.53 ft/s	5.53 ft/s	5.53 ft/s
Cond Tube Vel	5.92 ft/s	5.93 ft/s	5.93 ft/s	5.93 ft/s



Certified in accordance with the AHRI Water-Cooled Water-Chilling and Heat Pump Water-Heating Packages Certification Program, which is based on AHRI Standard 550/590 (I-P) and AHRI Standard 551/591 (SI). Certified units may be found in the AHRI Directory at www.ahrirectory.org.

<http://www.ahrinet.org/WCCL.aspx>

This unit complies with the efficiency requirements of ASHRAE 90.1 2013/2016 Path B.

1 Point LEED EAC4 can be earned when selecting the 19DV AquaEdge



AquaEdge Chiller Configuration
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

Prepared By: William Harris



19DV-G44G444425B5

Chiller	
Starter / VFD	Unit Mounted VFD (Harmonic Mitigation)
Refrigerant Type	R-1233zd(E)
Evaporator	
Size	G44
Waterbox Type	Marine Waterbox, 150 psi
Passes	2
Nozzle Arrangement	C - 2 Pass, Inlet on 2nd Stage End
Nozzle Options	Victaulic Evaporator Nozzles
Evaporator Hinges	None
Tubing	3/4" 0.025" Copper Cross Hatched Evaporator Tubes
Fluid Type	Fresh Water
Weights	
Total Rigging Weight	34995 lb
Total Operating Weight	41679 lb
Refrigerant Weight	1637 lb
Condenser	
Size	G44
Waterbox Type	Marine Waterbox, 150 psi
Passes	2
Nozzle Arrangement	R - 2 Pass, Inlet on 2nd Stage End
Nozzle Options	Victaulic Condenser Nozzles
Condenser Hinges	None
Tubing	3/4" 0.025" Copper Cross Hatched Condenser Tubes
Fluid Type	Fresh Water
Motor & Motor Controller	
Motor Size	NBH
Line Voltage-Phase-Hertz	460(V)-3-60(Hz)
Refrigerant Pump Voltage-Phase-Hertz	460(V)-3-60(Hz)
VFD Code	32VS Unit Mounted VFD (STD Tier)
VFD Size	D
Interrupt	100k Amp Interrupt
VFD Lug Size	4x4 / 0 - 500 kcmil ABB KT7X1200 - 3
Flow Controls	
Condenser Float Valve	HN
Economizer Float Valve	LN
Envelope Stability Control	Installed
Economizer Bypass	Not Installed
Dimensions	
Length	213 in
Width	108 in
Height	120 in
Shipment & Packaging	
Packaging Options	Chiller Bagging (Standard for domestic travel only)



AquaEdge Chiller Configuration
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

Prepared By: William Harris

Miscellaneous Options	
Evaporator Insulation	STD.
Sensor Package	Not Selected
Soleplates	Not Selected
Spring Isolation Package	Not Selected
Controls & Power	
HMI Communication Protocol	CCN / BACNet Enabled
Fourth IOB	Not Selected

Note: This list of specified chiller options does not include any of the separate-ship accessories (gasket kits, refrigerant cylinders, standalone pumpout units, or storage tank / pumpout units), nor any items specified on a Quote Control. These items are specified as part of a bid, and not in the chiller tag. The Pricing Report includes all of these items as specified in a bid.

1 Point LEED EAC4 can be earned when selecting the 19DV AquaEdge



Performance Report
Alexandria Veterans Health Care - CH-3_19MV (2)
Carrier NAC Sales Office

Date: 06/16/2022 4:54 PM

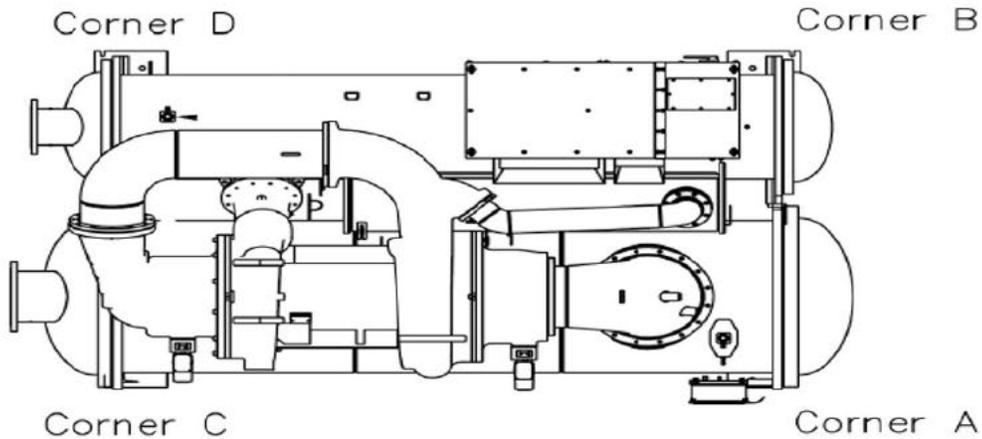
Prepared By: William Harris



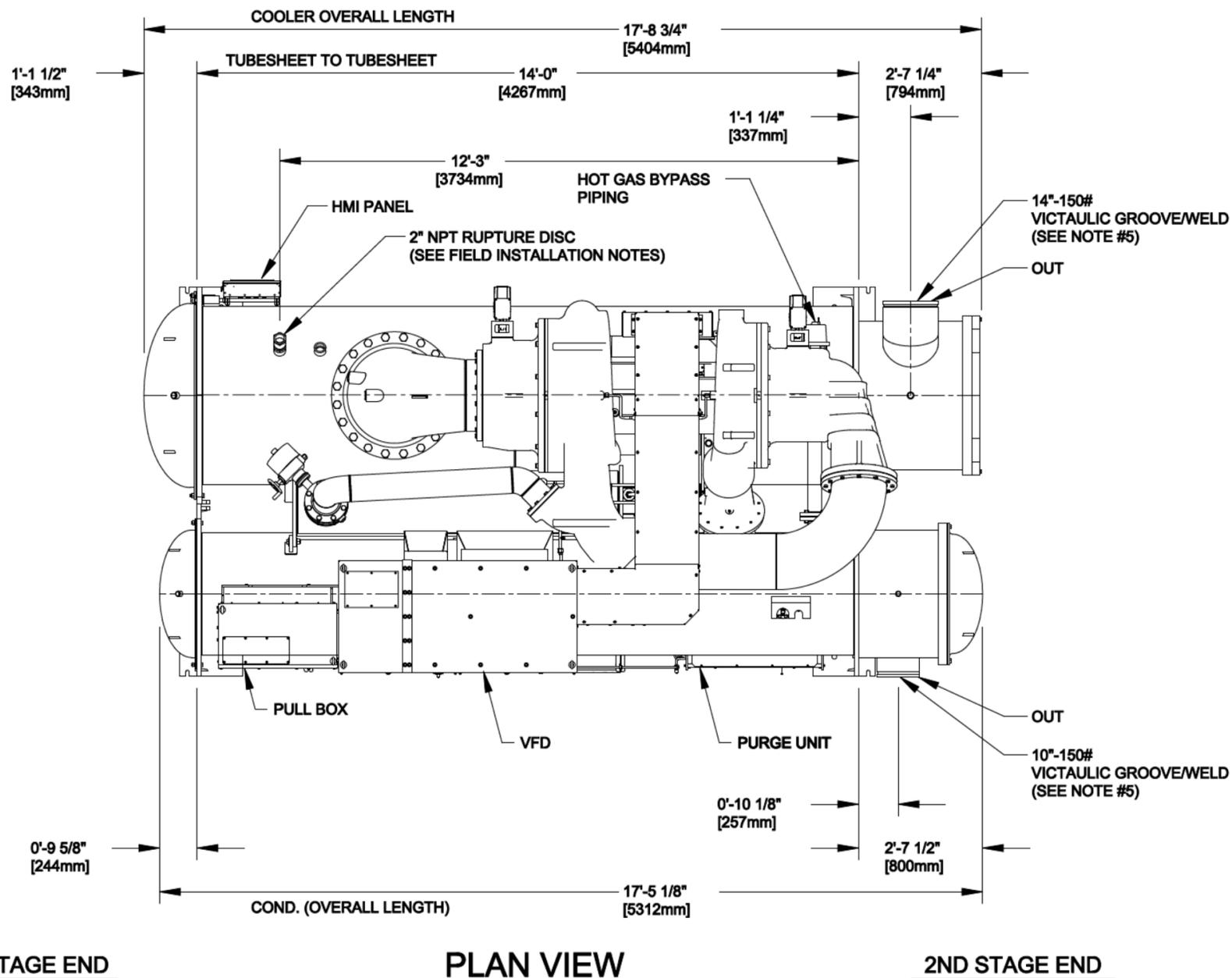
19DV-G44G444425B5

Chiller Rigging weight(no water no refrigerant weights)	34995.00 lb
Chiller Operation weight(with water and refrigerant weights)	41679.00 lb
Chiller Refrigerant weight	1637.00 lb

		Corner A	Corner B	Corner C	Corner D
Corner Weight	lb	10319	8878	12234	10248
Spring Corner		C12T-1D-14400	C12T-1D-10800	C12T-1D-16320	C12T-1D-14400
Spring Color		Gray	Dark Green	White	Gray



Note: Reference chiller shown, see chiller drawing for actual water box arrangement



1ST STAGE END

PLAN VIEW

2ND STAGE END

COMPRESSOR 19DV4	HEAT EXCHANGERS										MACHINE OPTIONS			
	COOLER: 19DVG40-44, G4A-G4E					CONDENSER: 19DVG42-44, G4C-G4E					HARMONIC FILTER	HOT GAS BYPASS	FREE COOLING	PULL BOX
MOTOR VOLTAGE	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END				
LOW	2	C	150 PSIG	404 LBS. [183 KG.]	668 LBS. [303 KG.]	2	R	150 PSIG	155 LBS. [70 KG.]	172 LBS. [78 KG.]	NOT SHOWN	✓	-	✓

- NOTES: 1. TOLERANCES ON NOZZLE LOCATIONS AND OVERALL DIMENSIONS ARE 1" [25mm].
2. CARRIER DOES NOT RECOMMEND PRE-FAB WATER PIPING.
3. IF HINGED WATERBOXES ARE INCLUDED, THE HINGES ARE NOT SHOWN ON THIS DRAWING AND MAY CHANGE THE MACHINE OVERALL LENGTH AND WIDTH. PLEASE CONSULT THE HINGED WATERBOX DRAWING FOR MORE DETAILS.
4. ✓ DENOTES APPLICABLE CONDITIONS
5. FOR VICTAULIC GROOVE DETAIL SEE DRAWING 19XR24VG.

BUYER:

JOB NAME:
Alexandria Veterans Health Care

SALES ENG.:

MODEL NO.:
19DV-G44G444425B5

JOB NO.:

P.O. NO.:

PREPARED BY:

ELECTRICAL
CHARACTERISTICS:
460/3/60

JOBSITE LOCATION:

SALES OFFICE:
NAC

REFRIGERANT NO.:
R-1233ZD

NOTES:

COMP'R: 19DV4
COOLER: 19DVG40-G44
19DVG4A-G4E
COND: 19DVG42-G44
19DVG4C-G4E
MACHINE ASSEMBLY
PLAN VIEW

DATE: 6/17/2022

REVISION:

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CARRIER DWG # 19DVG4-P

REV. C SHT 09F 01

DATE: 5/2/2019

SUPERSEDES DWG. DATED:
5/1/2018

19DVG4

SHT 09F 04

BUYER:

JOB NAME:
Alexandria Veterans Health Care

SALES ENG.:

MODEL NO.:
19DV-G44G444425B5

JOB NO.:

P.O. NO.:

PREPARED BY:

ELECTRICAL CHARACTERISTICS:
460/3/60

JOBSITE LOCATION:

SALES OFFICE:
NAC

REFRIGERANT NO.:
R-1233ZD

NOTES:

COMP'R: 19DV4
COOLER: 19DVG40-G44
19DVG4A-G4E
COND: 19DVG42-G44
19DVG4C-G4E
MACHINE ASSEMBLY
FRONT VIEW

DATE: 6/17/2022

REVISION:

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CARRIER DWG # 19DVG4-F

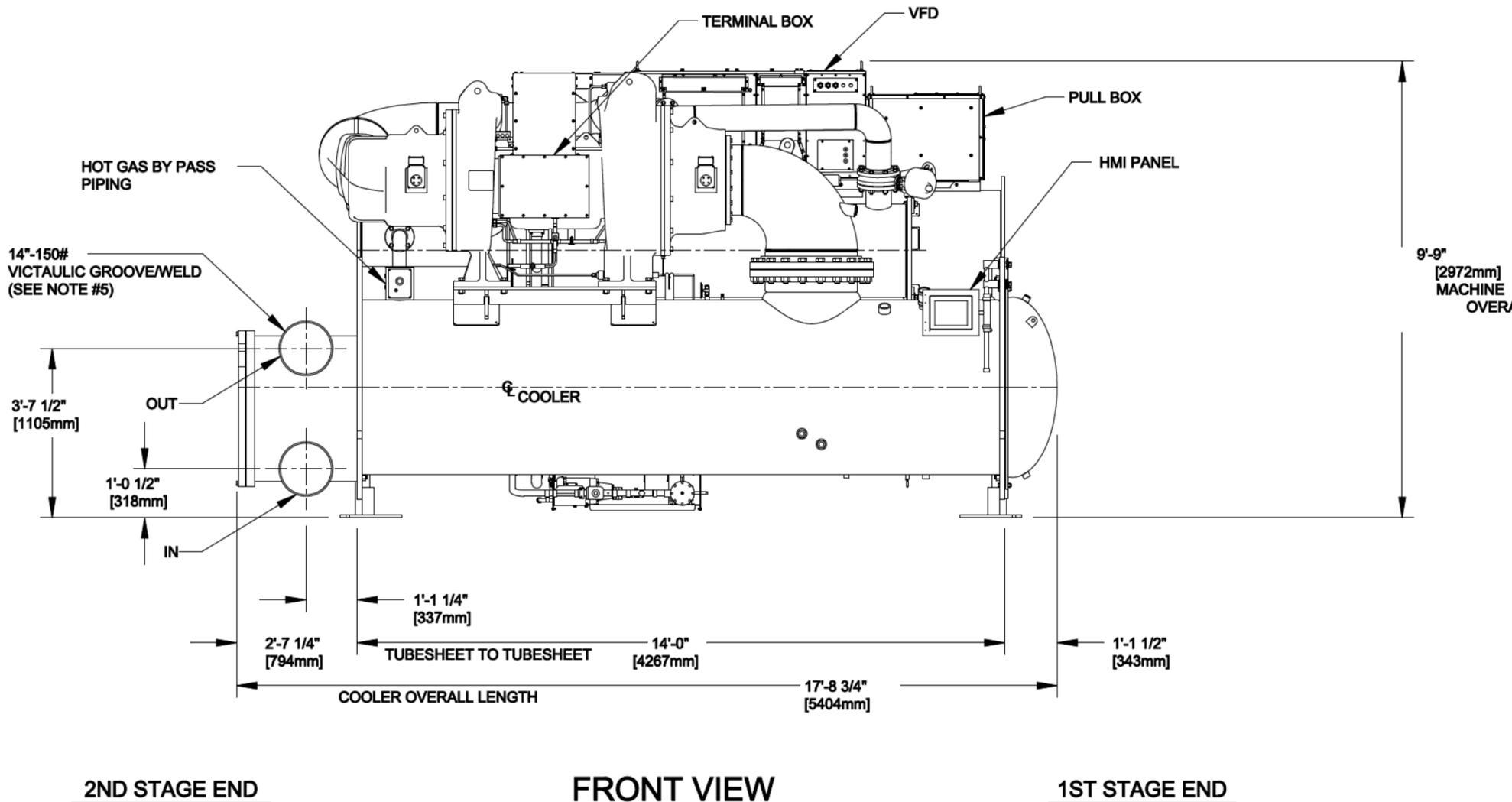
REV. C SHT 02F 01

DATE: 5/2/2019

SUPERSEDES DWG. DATED:
5/1/2018

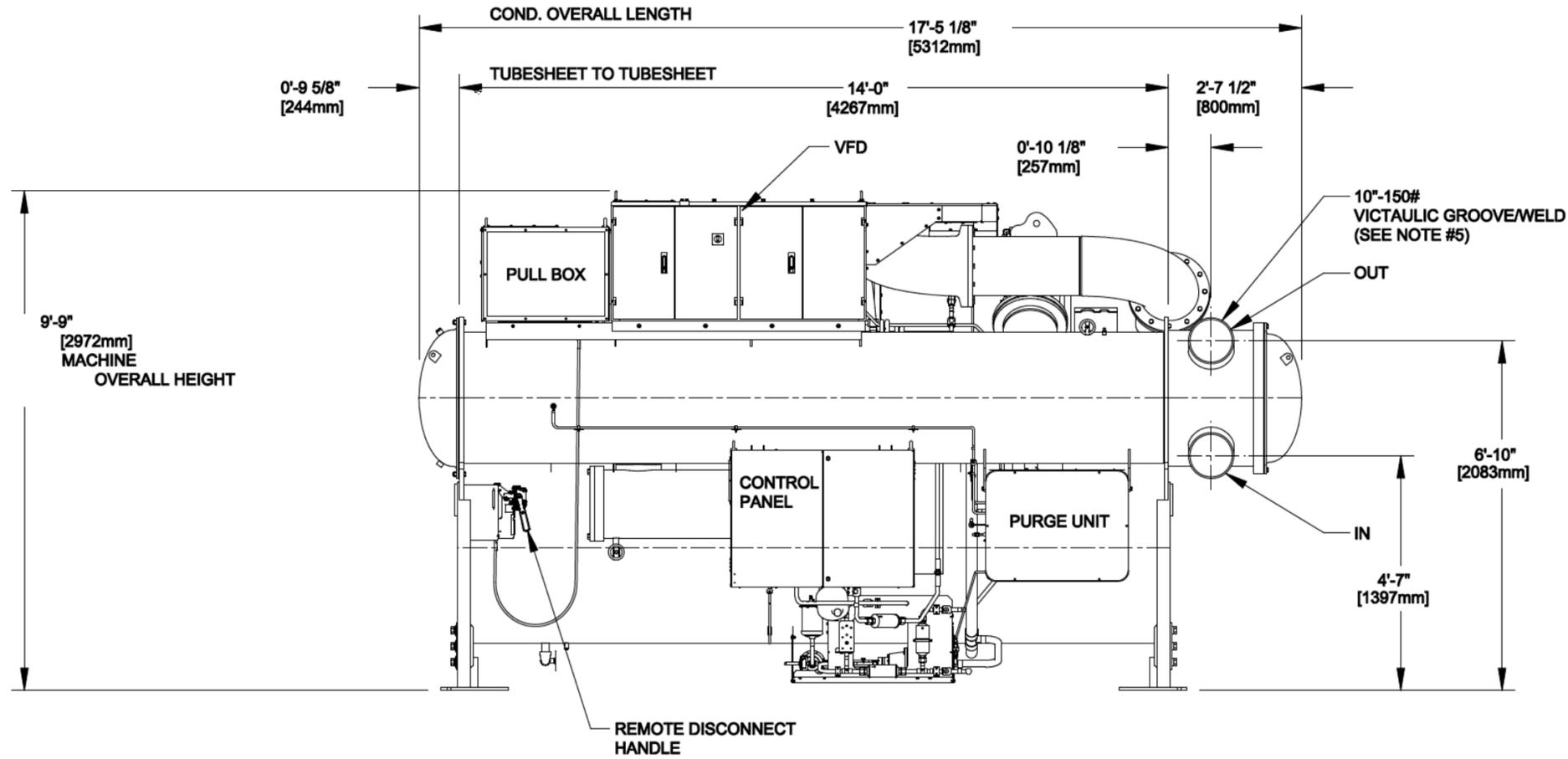
19DVG4

SHT 02F 04



COMPRESSOR 19DV4	HEAT EXCHANGERS										MACHINE OPTIONS			
	COOLER: 19DVG40-44, G4A-G4E					CONDENSER: 19DVG42-44, G4C-G4E					HARMONIC FILTER	HOT GAS BYPASS	FREE COOLING	PULL BOX
MOTOR VOLTAGE	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END				
LOW	2	C	150 PSIG	404 LBS. [183 KG.]	668 LBS. [303 KG.]	FOR CONDENSER ARRANGEMENTS SEE REAR VIEW					NOT SHOWN	✓	-	✓

- NOTES: 1. TOLERANCES ON NOZZLE LOCATIONS AND OVERALL DIMENSIONS ARE 1" [25mm].
2. CARRIER DOES NOT RECOMMEND PRE-FAB WATER PIPING.
3. IF HINGED WATERBOXES ARE INCLUDED, THE HINGES ARE NOT SHOWN ON THIS DRAWING AND MAY CHANGE THE MACHINE OVERALL LENGTH AND WIDTH. PLEASE CONSULT THE HINGED WATERBOX DRAWING FOR MORE DETAILS.
4. ✓ DENOTES APPLICABLE CONDITIONS
5. FOR VICTAULIC GROOVE DETAIL SEE DRAWING 19XR24VG.



1ST STAGE END

REAR VIEW

2ND STAGE END

COMPRESSOR 19DV4	HEAT EXCHANGERS										MACHINE OPTIONS			
	COOLER: 19DVG40-44, G4A-G4E					CONDENSER: 19DVG42-44, G4C-G4E					HARMONIC FILTER	HOT GAS BYPASS	FREE COOLING	PULL BOX
MOTOR VOLTAGE	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END				
LOW				FOR COOLER ARRANGEMENTS SEE FRONT VIEW		2	R	150 PSIG	155 LBS. [70 KG.]	172 LBS. [78 KG.]	NOT SHOWN	NOT SHOWN	-	✓

- NOTES: 1. TOLERANCES ON NOZZLE LOCATIONS AND OVERALL DIMENSIONS ARE 1" [25mm].
 2. CARRIER DOES NOT RECOMMEND PRE-FAB WATER PIPING.
 3. IF HINGED WATERBOXES ARE INCLUDED, THE HINGES ARE NOT SHOWN ON THIS DRAWING AND MAY CHANGE THE MACHINE OVERALL LENGTH AND WIDTH. PLEASE CONSULT THE HINGED WATERBOX DRAWING FOR MORE DETAILS.
 4. ✓ DENOTES APPLICABLE CONDITIONS
 5. FOR VICTAULIC GROOVE DETAIL SEE DRAWING 19XR24VG.

BUYER:

JOB NAME:
Alexandria Veterans Health Care

SALES ENG.:

MODEL NO.:
19DV-G44G444425B5

JOB NO.:

P.O. NO.:

PREPARED BY:

ELECTRICAL
CHARACTERISTICS:
460/3/60

JOB SITE LOCATION:

SALES OFFICE:
NAC

REFRIGERANT NO.:
R-1233ZD

NOTES:

COMP'R: 19DV4
 COOLER: 19DVG40-G44
 19DVG4A-G4E
 COND: 19DVG42-G44
 19DVG4C-G4E
 MACHINE ASSEMBLY
 REAR VIEW

DATE: 6/17/2022

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CARRIER DWG # 19DVG4-R

REV. C SHT 03F 01

DATE: 5/2/2019

SUPERSEDES DWG. DATED:
5/1/2018

19DVG4

SHT 03F 04

BUYER:

JOB NAME:
Alexandria Veterans Health Care

SALES ENG.:

MODEL NO.:
19DV-G44G444425B5

JOB NO.:

P.O. NO.:

PREPARED BY:

ELECTRICAL
CHARACTERISTICS:
460/3/60

JOB SITE LOCATION:

SALES OFFICE:
NAC

REFRIGERANT NO.:
R-1233ZD

NOTES:

COMP'R: 19DV4
COOLER: 19DVG40-G44
19DVG4A-G4E
COND: 19DVG42-G44
19DVG4C-G4E
MACHINE ASSEMBLY
1ST STAGE END VIEW

DATE: 6/17/2022

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CARRIER DWG # 19DVG4-1SE

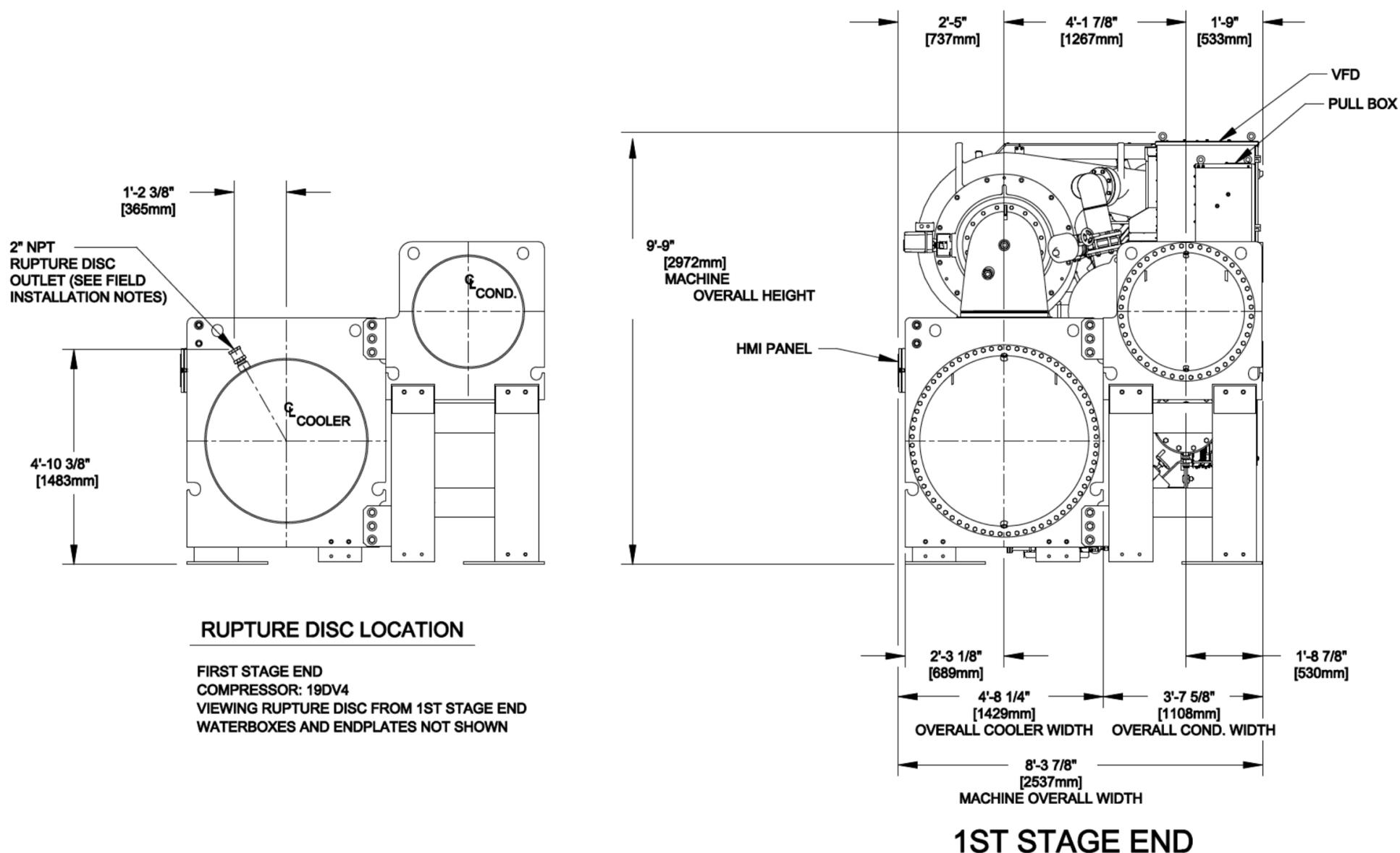
REV. C SHT 09F 01

DATE: 5/2/2019

SUPERSEDES DWG. DATED:
5/1/2018

19DVG4

SHT 09F 04



RUPTURE DISC LOCATION

FIRST STAGE END
COMPRESSOR: 19DV4
VIEWING RUPTURE DISC FROM 1ST STAGE END
WATERBOXES AND ENDPLATES NOT SHOWN

COMPRESSOR 19DV4	HEAT EXCHANGERS										MACHINE OPTIONS			
	COOLER: 19DVG40-44, G4A-G4E					CONDENSER: 19DVG42-44, G4C-G4E					HARMONIC FILTER	HOT GAS BYPASS	FREE COOLING	PULL BOX
MOTOR VOLTAGE	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END	PASS	NOZZLE CODE	PRESSURE	W/B COVER WT. 1ST. STAGE END	W/B COVER WT. 2ND. STAGE END				
LOW	2	C	150 PSIG	404 LBS. [183 KG.]	668 LBS. [303 KG.]	2	R	150 PSIG	155 LBS. [70 KG.]	172 LBS. [78 KG.]	NOT SHOWN	NOT SHOWN	-	✓

- NOTES: 1. TOLERANCES ON NOZZLE LOCATIONS AND OVERALL DIMENSIONS ARE 1" [25mm].
2. CARRIER DOES NOT RECOMMEND PRE-FAB WATER PIPING.
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4. ✓ DENOTES APPLICABLE CONDITIONS
5. FOR VICTAULIC GROOVE DETAIL SEE DRAWING 19XR24VG.

Building 147 – Replace Chiller #3, Project No. 502-21-110
Alexandria VAMC, Alexandria, LA.

Bid Documents Submittal

Appendix H Structural Calculations

Attachment 2
Structural Calculations

Innovative Engineering Inc.
3380 Trickum Road Bldg 500 Ste 100
Woodstock, GA 30188
770-517-5507

JOB TITLE VA Alexandria - Chiller Replacement

JOB NO. _____
CALCULATED BY SLM
CHECKED BY _____

SHEET NO. _____
DATE 4/24/22
DATE _____

CS2018 Ver 2018.11.30

www.struware.com

STRUCTURAL CALCULATIONS
FOR
VA Alexandria - Chiller Replacement

Alexandria, LA

Code Search

Code: International Building Code 2018

Occupancy:

Occupancy Group = I Institutional

Risk Category & Importance Factors:

Risk Category = IV
Wind factor = 1.00
Snow factor = 1.20
Seismic factor = 1.50

Type of Construction:

Fire Rating:
Roof = 0.0 hr
Floor = 0.0 hr

Building Geometry:

Roof angle (θ) 0.50 / 12 2.4 deg
Building length (L) 92.0 ft
Least width (B) 44.0 ft
Mean Roof Ht (h) 16.0 ft
Parapet ht above grd 0.0 ft
Minimum parapet ht 0.0 ft

Live Loads:

Roof
0 to 200 sf: 20 psf
200 to 600 sf: 24 - 0.02Area, but not less than 12 psf
over 600 sf: 12 psf

Floor:

Typical Floor 40 psf
Partitions 15 psf
Lobbies & first floor corridors 100 psf
Corridors above first floor 80 psf
Balconies (1.5 times live load) 60 psf

Innovative Engineering Inc.

3380 Trickum Road Bldg 500 Ste 100
Woodstock, GA 30188
770-517-5507

JOB TITLE VA Alexandria - Chiller Replacement

JOB NO. _____ SHEET NO. _____
CALCULATED BY SLM DATE 4/24/21
CHECKED BY _____ DATE _____

Wind Loads :

ASCE 7- 16

Ultimate Wind Speed	120 mph
Nominal Wind Speed	93 mph
Risk Category	IV
Exposure Category	B
Enclosure Classif.	Enclosed Building
Internal pressure	+/-0.18
Directionality (Kd)	0.85
Kh case 1	0.701
Kh case 2	0.585
Type of roof	Monoslope

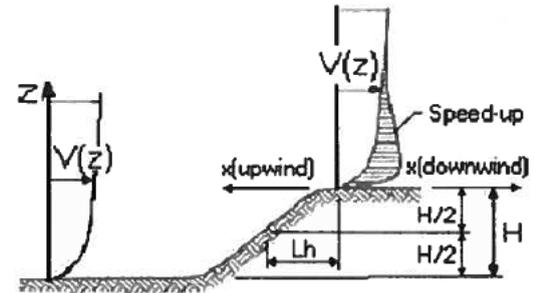
Topographic Factor (Kzt)

Topography	Flat
Hill Height (H)	80.0 ft
Half Hill Length (Lh)	100.0 ft
Actual H/Lh =	0.80
Use H/Lh =	0.50
Modified Lh =	160.0 ft
From top of crest: x =	50.0 ft
Bldg up/down wind?	downwind

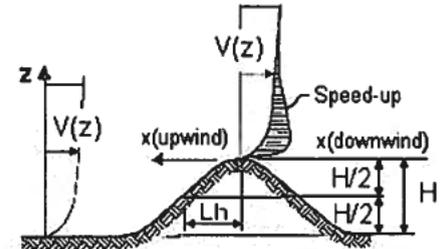
H/Lh = 0.50	K ₁ = 0.000
x/Lh = 0.31	K ₂ = 0.792
z/Lh = 0.10	K ₃ = 1.000

At Mean Roof Ht:

$Kzt = (1+K_1K_2K_3)^2 = 1.00$



ESCARPMENT



2D RIDGE or 3D AXISYMMETRICAL HILL

Gust Effect Factor

h =	16.0 ft
B =	44.0 ft
lz (0.6h) =	30.0 ft

Flexible structure if natural frequency < 1 Hz (T > 1 second).

If building h/B > 4 then may be flexible and should be investigated.

h/B = 0.36 Rigid structure (low rise bldg)

G = 0.85 Using rigid structure default

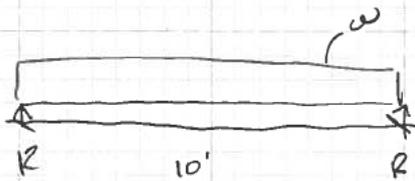
Rigid Structure

\bar{e} =	0.33
l_z =	320 ft
Z_{min} =	30 ft
c =	0.30
g_Q, g_v =	3.4
L_z =	310.0 ft
Q =	0.90
l_z =	0.30
G =	0.87 use G = 0.85

Flexible or Dynamically Sensitive Structure

3rdncy (η_1) =	0.0 Hz		
Damping ratio (β) =	0		
l/b =	0.45		
l/α =	0.25		
V_z =	77.3		
N_1 =	0.00		
K_n =	0.000		
R_h =	28.282	$\eta =$	0.000
R_B =	28.282	$\eta =$	0.000
R_L =	28.282	$\eta =$	0.000
g_R =	0.000		
R =	0.000		
Gf =	0.000		
			h = 16.01

New HSS Lintel



$$\Delta_{PTL} \leq l/600$$

$$P_{DL} = 9.26 \text{ k}$$

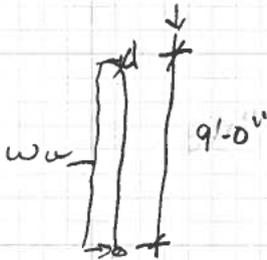
$$P_{UL} = 3.23 \text{ k}$$

$$w_{OL} = 1.82 \text{ klf}$$

$$w_{UL} = 645 \text{ plf}$$

USE HSS 12x8x1/4 LSH

MASONRY JAMBS



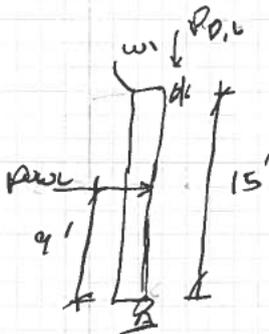
$$P_{DL} = 9.26 \text{ k}$$

$$P_{UL} = 3.23 \text{ k}$$

$$w_{UL} = 5 \text{ psf} \times 10\frac{1}{2} + 2' \cdot 50 \text{ psf}$$

USE (1) #5 @ JAMB

MASONRY KING JAMB



$$w_{OL} = 26 \text{ psf} \times 4' = 104 \text{ psf}$$

$$P_{UL} = 26 \text{ psf} \times \frac{10'}{2} \times \frac{4'}{2} = 525 \text{ k}$$

$$P_{DL} = \frac{44}{2} \times 83 \text{ psf} \times 4 = 7.3 \text{ k}$$

$$P_{UL} = \frac{44}{2} \times 20 \text{ psf} \times 4' = 1.76 \text{ k}$$



Innovative Engineering Inc.

PREPARED BY _____ DATE _____

CHECKED BY _____ DATE _____

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 and then using the "Printing &
 Title Block" selection.
 Title Block Line 6

Project Title:
 Engineer:
 Project ID:
 Project Descr:

Printed: 7 NOV 2022, 7:58AM

Steel Beam

File: VA Alex Chiller Replacement.ec6

Software copyright ENERCALC, INC. 1983-2020, Build:12.20.8.24

Lic. #: KW-06002837

INNOVATIVE ENGINEERING INC.

DESCRIPTION: New Opening Tube Option

CODE REFERENCES

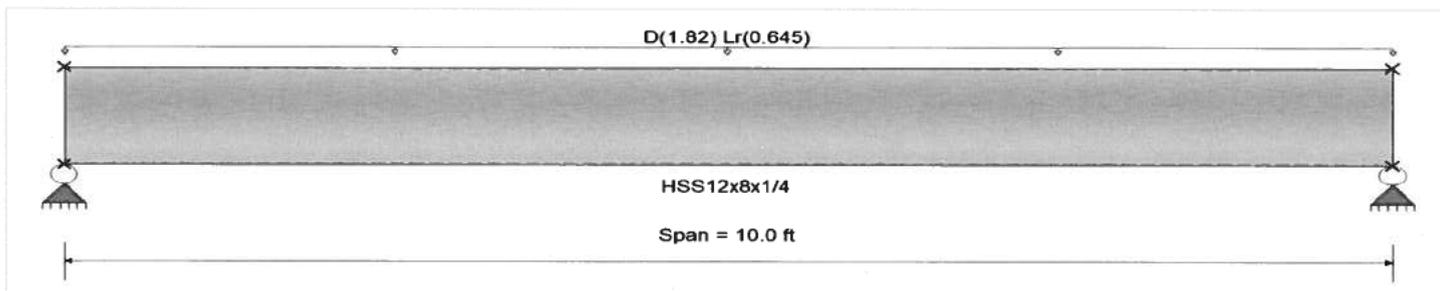
Calculations per AISC 360-16, IBC 2018, CBC 2019, ASCE 7-16

Load Combination Set : ASCE 7-16

Material Properties

Analysis Method : Allowable Strength Design
 Beam Bracing : Completely Unbraced
 Bending Axis : Minor Axis Bending

Fy : Steel Yield : 46.0 ksi
 E : Modulus : 29,000.0 ksi



Applied Loads

Service loads entered. Load Factors will be applied for calculations.

Beam self weight calculated and added to loading
 Uniform Load : D = 1.820, Lr = 0.6450 k/ft, Tributary Width = 1.0 ft

DESIGN SUMMARY

Design OK

Maximum Bending Stress Ratio =	0.650 : 1	Maximum Shear Stress Ratio =	0.222 : 1
Section used for this span	HSS12x8x1/4	Section used for this span	HSS12x8x1/4
Ma : Applied	31.220 k-ft	Va : Applied	12.488 k
Mn / Omega : Allowable	48.049 k-ft	Vn/Omega : Allowable	56.229 k
Load Combination	+D+Lr+H	Load Combination	+D+Lr+H
Location of maximum on span	5.000 ft	Location of maximum on span	0.000 ft
Span # where maximum occurs	Span # 1	Span # where maximum occurs	Span # 1
Maximum Deflection			
Max Downward Transient Deflection	0.051 in Ratio = 2,358 >=600.		
Max Upward Transient Deflection	0.000 in Ratio = 0 <600.0		
Max Downward Total Deflection	0.197 in Ratio = 609 >=600.		
Max Upward Total Deflection	0.000 in Ratio = 0 <600.0		

Maximum Forces & Stresses for Load Combinations

Load Combination	Segment Length	Span #	Max Stress Ratios		Summary of Moment Values						Summary of Shear Values				
			M	V	Mmax +	Mmax -	Ma Max	Mny	Mny/Omega	Cb	Rm	Va Max	Vny	Vny/Omega	
+D+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+D+L+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+D+Lr+H															
Dsgn. L = 10.00 ft		1	0.650	0.222	31.22		31.22	80.24	48.05	1.14	1.00	12.49	93.90	56.23	
+D+S+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+D+0.750Lr+0.750L+H															
Dsgn. L = 10.00 ft		1	0.608	0.208	29.20		29.20	80.24	48.05	1.14	1.00	11.68	93.90	56.23	
+D+0.750L+0.750S+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+D+0.60W+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+D+0.750Lr+0.750L+0.450W+H															
Dsgn. L = 10.00 ft		1	0.608	0.208	29.20		29.20	80.24	48.05	1.14	1.00	11.68	93.90	56.23	
+D+0.750L+0.750S+0.450W+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+0.60D+0.60W+0.60H															
Dsgn. L = 10.00 ft		1	0.289	0.099	13.89		13.89	80.24	48.05	1.14	1.00	5.56	93.90	56.23	
+D+0.70E+0.60H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	
+D+0.750L+0.750S+0.5250E+H															
Dsgn. L = 10.00 ft		1	0.482	0.165	23.16		23.16	80.24	48.05	1.14	1.00	9.26	93.90	56.23	

Title Block Line 1
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 Engineer:
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Steel Beam

File: VA Alex Chiller Replacement.ec6

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INNOVATIVE ENGINEERING INC.

Lic. #: KW-06002837

DESCRIPTION: New Opening Tube Option

Load Combination	Segment Length	Span #	Max Stress Ratios		Summary of Moment Values						Summary of Shear Values			
			M	V	Mmax +	Mmax -	Ma Max	Mnx	Mnx/Omega	Cb	Rm	Va Max	Vnx	Vnx/Omega
+0.60D+0.70E+H	Dsgn. L = 10.00 ft	1	0.289	0.099	13.89		13.89	80.24	48.05	1.14	1.00	5.56	93.90	56.23

Overall Maximum Deflections

Load Combination	Span	Max. "+" Defl	Location in Span	Load Combination	Max. "+" Defl	Location in Span
+D+Lr+H	1	0.1970	5.029		0.0000	0.000

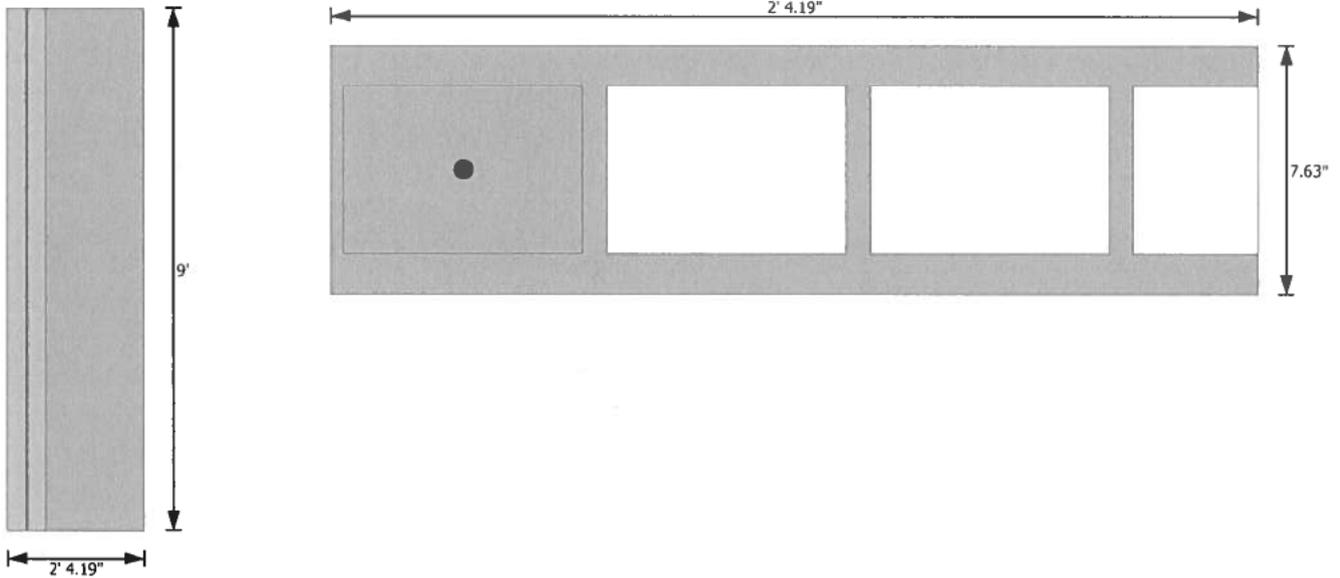
Vertical Reactions

Support notation : Far left is #1

Values in KIPS

Load Combination	Support 1	Support 2
Overall MAXimum	12.488	12.488
Overall MINimum	3.225	3.225
+D+H	9.263	9.263
+D+L+H	9.263	9.263
+D+Lr+H	12.488	12.488
+D+S+H	9.263	9.263
+D+0.750Lr+0.750L+H	11.682	11.682
+D+0.750L+0.750S+H	9.263	9.263
+D+0.60W+H	9.263	9.263
+D+0.750Lr+0.750L+0.450W+H	11.682	11.682
+D+0.750L+0.750S+0.450W+H	9.263	9.263
+0.60D+0.60W+0.60H	5.558	5.558
+D+0.70E+0.60H	9.263	9.263
+D+0.750L+0.750S+0.5250E+H	9.263	9.263
+0.60D+0.70E+H	5.558	5.558
D Only	9.263	9.263
Lr Only	3.225	3.225
H Only		

Masonry Pilaster Design Calculations



Design Criteria

Building Code: TMS 402-2016 (Strength Design)
 Choose Load Combinations Manually: Yes
 Strength Check Load Combinations: ASCE 7-16 Strength Combinations
 Service Check Load Combinations: ASCE 7-16 ASD Combinations
 Apply Sds to Seismic Combinations for Ev: No
 f'm: 1500 psi
 fy: 60000 psi
 Mortar Type: Type M/S - PCL (Portland Cement/Lime)

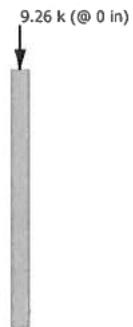
Material (CMU/Clay): Taken from Unit
 Average Unit Weight: 140 lb/ft³
 Secondary Moment Approach: Moment Magnifier
 Include Self-Weight: Yes
 Include Weight In Virtual Eccentricity: No

Checks Summary

<input type="checkbox"/> Ratio	Check	Critical Combination	<input type="checkbox"/> Ratio	Check
Strength Checks				
✓ 0.287	Axial Stress	1.2D + 1.6L		
✓ 0.000	Axial Stress (slender)	1.4D		
✓ 0.107	Axial Force	1.2D + 1.6L		
✓ 0.032	Shear	1.2D + 1.6L		
✓ 0.088	Moment	1.2D + 1.6L		
✓ 0.000	Max Reinforcement	1.4D		
Service Checks				
✓ 0.005	Deflection (TMS)	1D + 0.75L		
✓ 0.000	Deflection (IBC)	1D		

Applied Loads

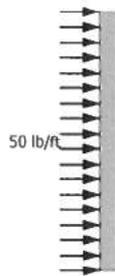
dead [Source: Dead]



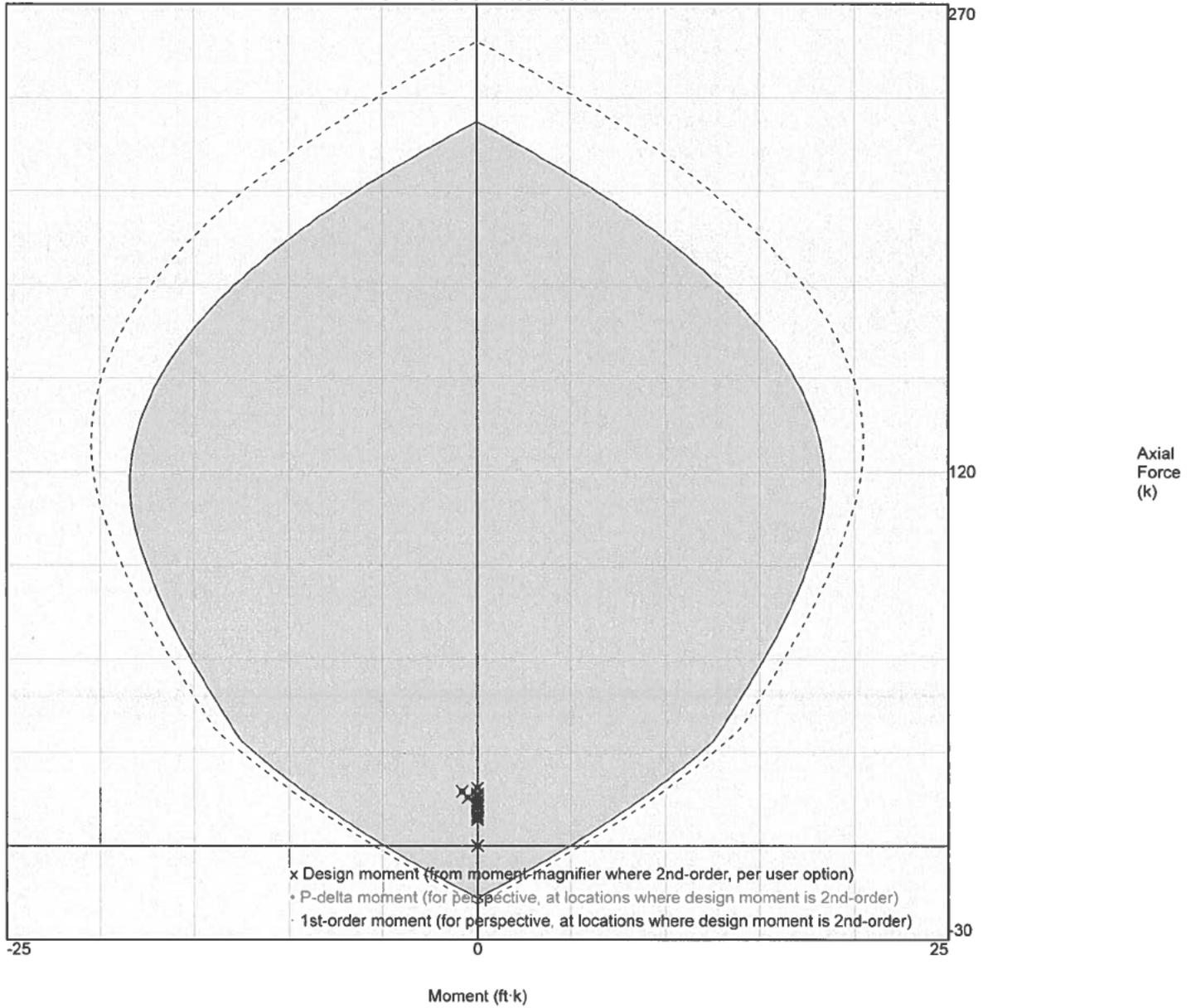
live [Source: Live]



live2 [Source: Live]



Axial-Flexural Interaction



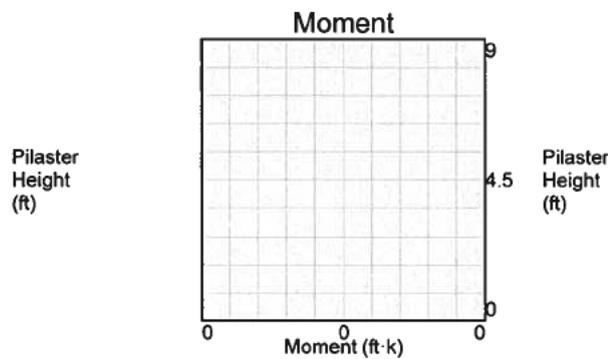
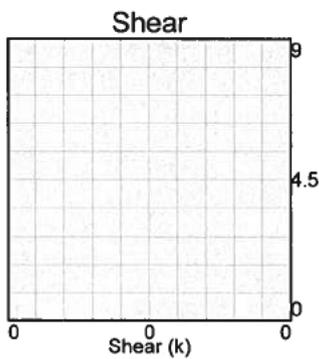
Strength Checks Design Calculations for Combination 1.4D

Load Combination: 1.4D

Pilaster weight: The unfactored weight is 1.88 k, of which 1.35 k is from the wall portion(s) and 0.53 k is from the 'column' portion.

Factored Pilaster weight = 2.63 k (total), 292.54 lb/ft (linear per height).

	Axial Force (k)	Shear (k)	Moment (ft-k)	P-δ Moment (ft-k)	Mag. Moment (ft-k)
9 ft from base	0 k	0 k	0 ft-k	0 ft-k	0 ft-k
	12.96 k	0 k	0 ft-k	N/A	N/A
4.45 ft from base	14.29 k	0 k	0 ft-k	0 ft-k	0 ft-k
0 ft from base	15.6 k	0 k	0 ft-k	N/A	N/A



Secondary Moments

The strength design provisions of TMS 402 require consideration of a secondary moment. Both P-Delta and moment magnifier options are given. The P-delta approach may be used under some conditions and the moment magnifier may be used under all conditions. Both are calculated here for perspective, but per user option the value used for design will be that from the moment magnifier approach.

Cracking Moment

The distance from the neutral axis to the extreme tension fiber 'c' is 3.81 in. This distance is calculated based on the effective cross section of the full wall. Note that because the masonry units used in this wall are not symmetrical, this value will be different when the moment is in the opposite direction such that the tension and compression faces are reversed.

$$S_n = I_n / c_t = (1,041.35 \text{ in}^4) / (3.81 \text{ in}) = 273.14 \text{ in}^3$$

$$f_r = 89.27 \text{ psi}$$

$$M_{cr} = S_n f_r = (273.14 \text{ in}^3)(89.27 \text{ psi}) = 2.03 \text{ ft-k}$$

[NCMA TEK note 14-4B (2008), p. 3]

Secondary Moment Calculation (At Midspan)

Note that the I_n value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \frac{(0 \text{ ft-k})}{1 - \frac{5(14.29 \text{ k})(9 \text{ ft})^2}{48(1,350,000 \text{ psi})(1,041.35 \text{ in}^4)}} = 0 \text{ ft-k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(1,041.35 \text{ in}^4) = 781.01 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi})(781.01 \text{ in}^4))}{(9 \text{ ft})^2} \right) = 892.16 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \left(\frac{1}{1 - \left(\frac{(14.29 \text{ k})}{(892.16 \text{ k})} \right)} \right) = 1.0163$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1.0163)(0 \text{ ft}\cdot\text{k}) = 0 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Secondary Moment Calculation (At Max Moment Location)

Note that the I_x value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \left(\frac{((0 \text{ ft}\cdot\text{k}))}{1 - \left(\frac{(5(0 \text{ k})(9 \text{ ft})^2)}{(48(1,350,000 \text{ psi})(1,041.35 \text{ in}^4))} \right)} \right) = 0 \text{ ft}\cdot\text{k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(1,041.35 \text{ in}^4) = 781.01 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi})(781.01 \text{ in}^4))}{(9 \text{ ft})^2} \right) = 892.16 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \left(\frac{1}{1 - \left(\frac{(0 \text{ k})}{(892.16 \text{ k})} \right)} \right) = 1$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1)(0 \text{ ft}\cdot\text{k}) = 0 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Strength Checks: Combination 1.4D

Design forces for this load combination:

- P_u @ top = 12.96 k
- P_u @ mid-span = 14.29 k
- P_u @ base = 15.6 k
- P_u @ max moment = 0 k
- M_u @ top = 0 ft·k
- M_u @ mid-span = 0 ft·k
- Max M_u = 0 ft·k
- Max V_u = 0 k

Axial Stress Check

$$\sigma_p = P_u / A_g = (15.6 \text{ k}) / (1.49 \text{ ft}^2) = 72.57 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

$$\sigma_{pmax} = 0.2f_m = 0.2(1,500 \text{ psi}) = 300 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

✓ $\sigma_{pmax} \geq \sigma_P$...utilization ratio 0.242

Axial Stress Check with Slender Wall

$$\frac{h}{t} = \frac{(9 \text{ ft})}{(99,999,999 \text{ in})} = 0$$

[TMS 402-16 Section 9.3.5.4.2]

Slenderness ratio does not exceed 30; check does not apply

✓ $\sigma_{pmax} \geq \sigma_P$...utilization ratio 0

Axial Force Check

$$\frac{h}{r} = \frac{(9 \text{ ft})}{(2.2 \text{ in})} = 49.07$$

[TMS 402-16 Section 9.3.4.1.1]

$$P_n = 0.80 \left[0.80f_m (A_n - A_{st}) + f_y A_{st} \right] \left[1 - \left(\frac{h}{140r} \right)^2 \right] = 0.80 \left[(0.80(1,500 \text{ psi}))((1.49 \text{ ft}^2) - (0 \text{ in}^2)) + (60,000 \text{ psi})(0 \text{ in}^2) \right] \left(1 - \left(\frac{(9 \text{ ft})}{(140(2.2 \text{ in}))} \right)^2 \right) = 180.99 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.1, Equation 9-15]

$$\phi P_n = \phi P_n = (0.9)(180.99 \text{ k}) = 162.89 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.4]

$$P_u = 14.29 \text{ k}$$

✓ $\phi P_n \geq P_u$...utilization ratio 0.088

Moment Check @ Axial Load Application Point

The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (12.96 k) and thus $P_n = 14.4 \text{ k}$. The associated M_n (9.14 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 12.96 k, the interaction diagram gives a moment capacity of -8.23 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.19 ft². Depth of compression zone = 0.98 in.

Moment Check @ Mid-span

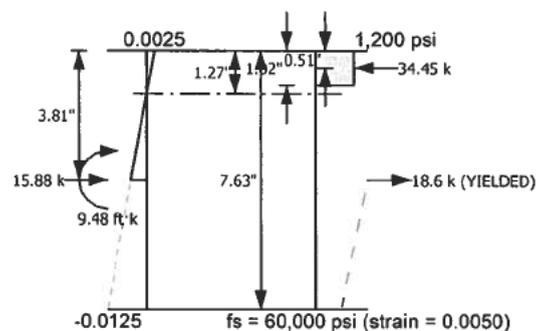
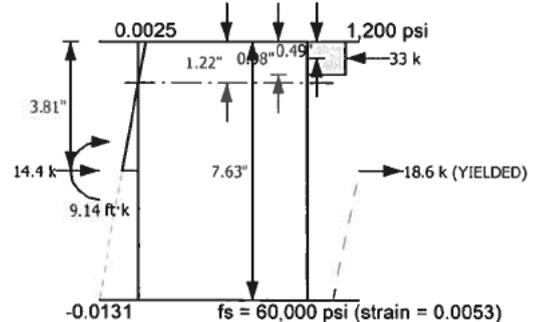
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (14.29 k) and thus $P_n = 15.88 \text{ k}$. The associated M_n (9.48 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 14.29 k, the interaction diagram gives a moment capacity of -8.54 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.2 ft². Depth of compression zone = 1.02 in.



Moment Check @ Max Moment

The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (0 k) and thus $P_n = 0$ k. The associated M_n (5.48 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 0 k, the interaction diagram gives a moment capacity of -4.93 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.11 ft². Depth of compression zone = 0.55 in.

Shear Check

There is no applied shear force in this load combination; check passes.

✓ $\phi V_n \geq V_u$...utilization ratio 0

Maximum Reinforcement Check

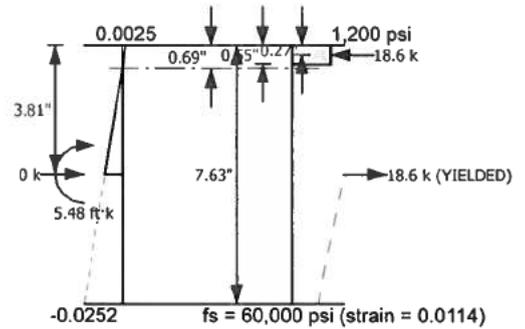
Maximum reinforcement is checked based on the provisions of TMS 402-16 9.3.3.2.

$$\frac{M_u}{V_u d_v} = \frac{(0 \text{ ft}\cdot\text{k})}{((0 \text{ k})(99,999,999 \text{ in}))} = 0$$

[TMS 402-16 Section 9.3.4.1.2]

Because this ratio does not exceed 1.0 and $R \leq 1.5$, there is no limit on flexural tensile reinforcement.

✓ $\epsilon_s \geq \epsilon_{min}$...utilization ratio 0



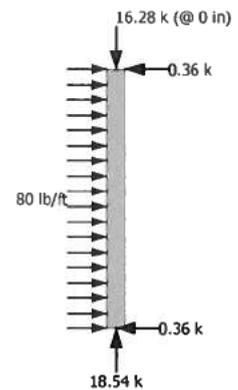
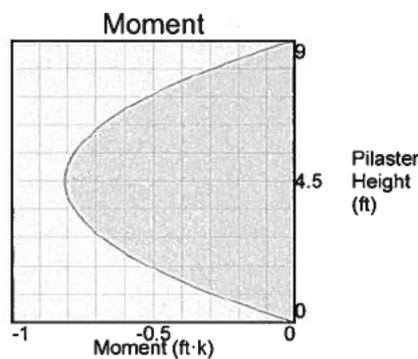
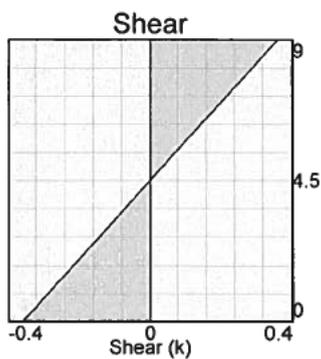
Strength Checks Design Calculations for Combination 1.2D + 1.6L

Load Combination: 1.2D + 1.6L

Pilaster weight: The unfactored weight is 1.88 k, of which 1.35 k is from the wall portion(s) and 0.53 k is from the 'column' portion.

Factored Pilaster weight = 2.26 k (total), 250.75 lb/ft (linear per height).

	Axial Force (k)	Shear (k)	Moment (ft-k)	P-δ Moment (ft-k)	Mag. Moment (ft-k)
9 ft from base	16.28 k	0.36 k	0 ft-k	N/A	N/A
4.45 ft from base	17.42 k	0 k	-0.81 ft-k	-0.82 ft-k	-0.83 ft-k
0 ft from base	18.54 k	-0.36 k	0 ft-k	N/A	N/A



Secondary Moments

The strength design provisions of TMS 402 require consideration of a secondary moment. Both P-Delta and moment magnifier options are given. The P-delta approach may be used under some conditions and the moment magnifier may be used under all conditions. Both are calculated here for perspective, but per user option the value used for design will be that from the moment magnifier approach.

Cracking Moment (At Midspan)

The distance from the neutral axis to the extreme tension fiber 'c' is 3.81 in. This distance is calculated based on the effective cross section of the full wall. Note that because the masonry units used in this wall are not symmetrical, this value will be different when the moment is in the opposite direction such that the tension and compression faces are reversed.

$$S_n = I_r / c_t = (1,041.35 \text{ in}^4) / (3.81 \text{ in}) = 273.14 \text{ in}^3$$

$$f_r = 89.27 \text{ psi}$$

$$M_{cr} = S_n f_r = (273.14 \text{ in}^3) (89.27 \text{ psi}) = 2.03 \text{ ft}\cdot\text{k}$$

[NCMA TEK note 14-4B (2008), p. 3]

Cracking Moment (At Max Moment Location)

The distance from the neutral axis to the extreme tension fiber 'c' is 3.81 in. This distance is calculated based on the effective cross section of the full wall. Note that because the masonry units used in this wall are not symmetrical, this value will be different when the moment is in the opposite direction such that the tension and compression faces are reversed.

$$S_n = I_r / c_t = (1,041.35 \text{ in}^4) / (3.81 \text{ in}) = 273.14 \text{ in}^3$$

$$f_r = 89.27 \text{ psi}$$

$$M_{cr} = S_n f_r = (273.14 \text{ in}^3) (89.27 \text{ psi}) = 2.03 \text{ ft}\cdot\text{k}$$

[NCMA TEK note 14-4B (2008), p. 3]

Secondary Moment Calculation (At Midspan)

Note that the I_r value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{o1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \left(\frac{((-0.81 \text{ ft}\cdot\text{k}))}{1 - \frac{5(17.42 \text{ k})(9 \text{ ft})^2}{48(1,350,000 \text{ psi})(1,041.35 \text{ in}^4)}} \right) = -0.82 \text{ ft}\cdot\text{k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(1,041.35 \text{ in}^4) = 781.01 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi})(781.01 \text{ in}^4))}{(9 \text{ ft})^2} \right) = 892.16 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \left(\frac{1}{1 - \left(\frac{(17.42 \text{ k})}{(892.16 \text{ k})} \right)} \right) = 1.0199$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1.0199)(-0.81 \text{ ft}\cdot\text{k}) = -0.83 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Strength Checks: Combination 1.2D + 1.6L

Design forces for this load combination:

- P_u @ top = 16.28 k
- P_u @ mid-span = 17.42 k
- P_u @ base = 18.54 k
- P_u @ max moment = 18.54 k
- M_u @ top = 0 ft·k
- M_u @ mid-span = -0.81 ft·k
- Max M_u = 0 ft·k
- Max V_u = 0.36 k

Axial Stress Check

$$\sigma_p = P_u / A_g = (18.54 \text{ k}) / (1.49 \text{ ft}^2) = 86.25 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

$$\sigma_{pmax} = 0.2f'_m = 0.2(1,500 \text{ psi}) = 300 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

✓ $\sigma_{pmax} \geq \sigma_p$...utilization ratio 0.287

Axial Stress Check with Slender Wall

$$\frac{h}{t} = \frac{(9 \text{ ft})}{(99,999,999 \text{ in})} = 0$$

[TMS 402-16 Section 9.3.5.4.2]

Slenderness ratio does not exceed 30; check does not apply

✓ $\sigma_{pmax} \geq \sigma_p$...utilization ratio 0

Axial Force Check

$$\frac{h}{r} = \frac{(9 \text{ ft})}{(2.2 \text{ in})} = 49.07$$

[TMS 402-16 Section 9.3.4.1.1]

$$P_n = 0.80 \left[0.80 f_m (A_n - A_{st}) + f_y A_{st} \right] \left[1 - \left(\frac{h}{140r} \right)^2 \right] = 0.80 (0.80 (1,500 \text{ psi}) ((1.49 \text{ ft}^2) - (0 \text{ in}^2)) + (60,000 \text{ psi}) (0 \text{ in}^2)) \left(1 - \left(\frac{(9 \text{ ft})}{(140 (2.2 \text{ in}))} \right)^2 \right) = 180.99 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.1, Equation 9-15]

$$\phi P_n = \phi P_n = (0.9) (180.99 \text{ k}) = 162.89 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.4]

$$P_u = 17.42 \text{ k}$$

✓ $\phi P_n \geq P_u$...utilization ratio 0.107

Moment Check @ Axial Load Application Point

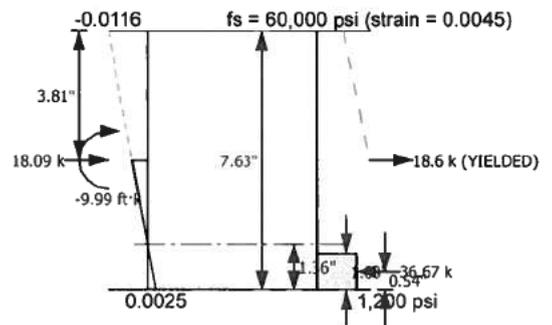
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (16.28 k) and thus $P_n = 18.09 \text{ k}$. The associated M_n (-9.99 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 16.28 k, the interaction diagram gives a moment capacity of 8.99 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.21 ft². Depth of compression zone = 1.08 in.



Moment Check @ Mid-span

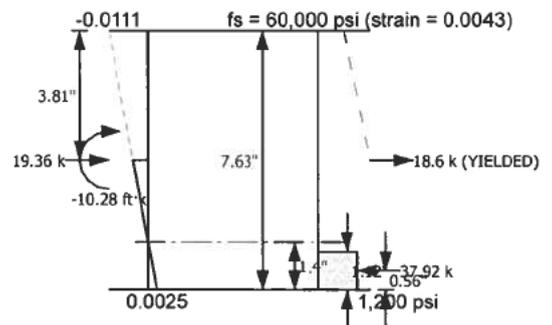
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (17.42 k) and thus $P_n = 19.36 \text{ k}$. The associated M_n (-10.28 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 17.42 k, the interaction diagram gives a moment capacity of 9.25 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0.088



Compression area = 0.22 ft². Depth of compression zone = 1.12 in.



Moment Check @ Max Moment

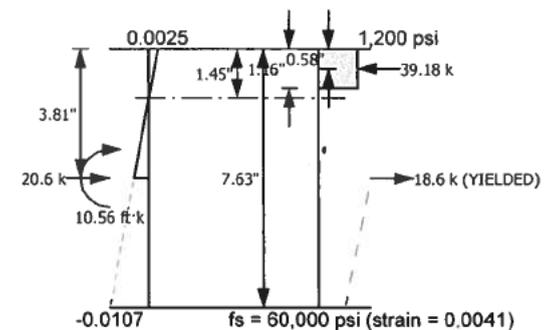
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (18.54 k) and thus $P_n = 20.6 \text{ k}$. The associated M_n (10.56 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 18.54 k, the interaction diagram gives a moment capacity of -9.5 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.23 ft². Depth of compression zone = 1.16 in.



Shear Check

The net shear area is taken as the area bounded by the extreme compression fiber and the reinforcement nearest the opposite face, subject to the effective compression width.

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Pilaster 2

$$A_{nv} = 0.75 \text{ ft}^2$$

Shear capacity due to masonry, conservatively taking $M/Vd = 1.0$:

$$V_{nm} = [4.0 - 1.75(1.0)] A_{nv} \sqrt{f'_m} + 0.25 P_u = (4.0 - 1.75(1.0))(0.75 \text{ ft}^2) \sqrt{(1,500 \text{ psi})} + 0.25(18.54 \text{ k}) = 14 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.2.1, Equation 9-20 with conservative $M/Vd = 1.0$]

No reinforcement for out-of-plane shear:

$$V_{ns} = 0 \text{ k}$$

Nominal shear capacity:

$$V_n = (V_{nm} + V_{ns}) \gamma_g = ((14 \text{ k}) + (0 \text{ k}))(1) = 14 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.2, Equation 9-17]

Limited by:

$$V_{n_limit} = (4 A_{nv} \sqrt{f'_m}) \gamma_g = (4(0.75 \text{ ft}^2) \sqrt{(1,500 \text{ psi})})(1) = 16.65 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.2, Equation 9-19]

...Limit on V_n does not control

$$\phi V_n = \phi V_n = (0.8)(14 \text{ k}) = 11.2 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.5]

✓ $\phi V_n \geq V_u$...utilization ratio 0.032

Maximum Reinforcement Check

Maximum reinforcement is checked based on the provisions of TMS 402-16 9.3.3.2.

$$\frac{M_u}{V_u d_v} = \frac{(0 \text{ ft} \cdot \text{k})}{((0.36 \text{ k})(99,999,999 \text{ in}))} = 0$$

[TMS 402-16 Section 9.3.4.1.2]

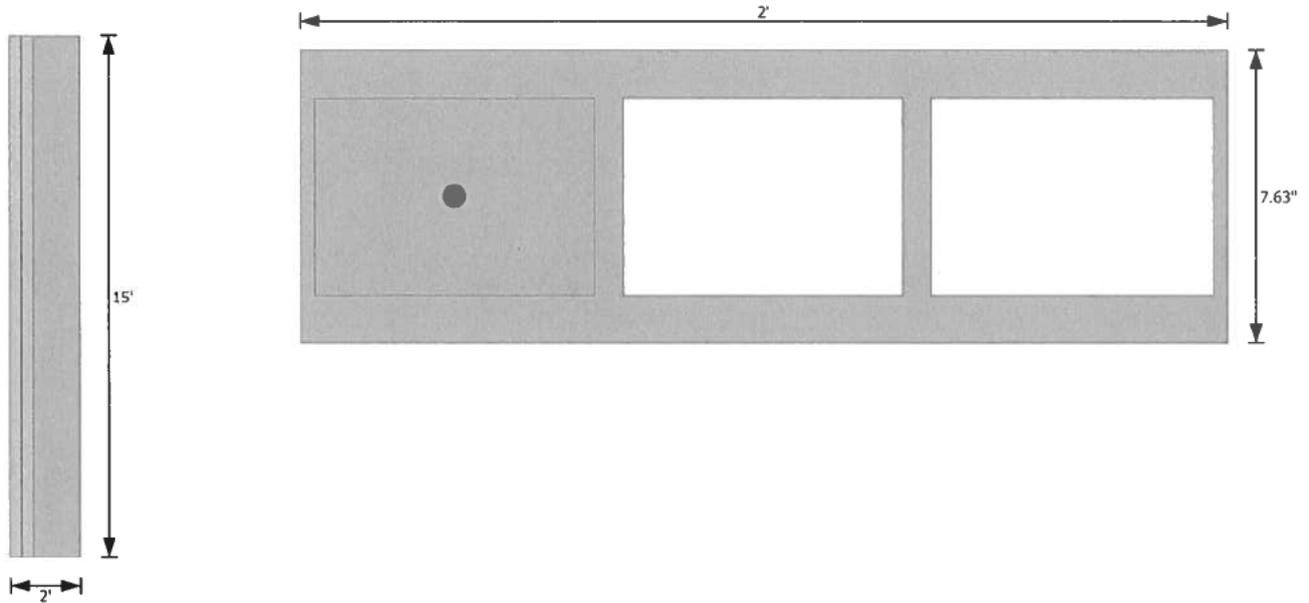
Because this ratio does not exceed 1.0 and $R \leq 1.5$, there is no limit on flexural tensile reinforcement.

✓ $\epsilon_s \geq \epsilon_{min}$...utilization ratio 0

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Pilaster 1

Masonry Pilaster Design Calculations



Design Criteria

Building Code: TMS 402-2016 (Strength Design)
 Choose Load Combinations Manually: Yes
 Strength Check Load Combinations: ASCE 7-16 Strength Combinations
 Service Check Load Combinations: ASCE 7-16 ASD Combinations
 Apply Sds to Seismic Combinations for Ev: No
 f'_m : 1500 psi
 f_y : 60000 psi
 Mortar Type: Type M/S - PCL (Portland Cement/Lime)

Material (CMU/Clay): Taken from Unit
 Average Unit Weight: 140 lb/ft³
 Secondary Moment Approach: Moment Magnifier
 Include Self-Weight: Yes
 Include Weight In Virtual Eccentricity: No

Checks Summary

<input type="checkbox"/> Ratio	Check	Critical Combination	<input type="checkbox"/> Ratio	Check
Strength Checks				
✓ 0.269	Axial Stress	1.2D + 1.6L		
✓ 0.000	Axial Stress (slender)	1.4D		
✓ 0.127	Axial Force	1.2D + 1.6L		
✓ 0.124	Shear	0.9D + 1Wind _{C&C+Wall}		
✓ 0.726	Moment	0.9D + 1Wind _{C&C+Wall}		
✓ 0.000	Max Reinforcement	1.4D		
Service Checks				
✓ 0.081	Deflection (TMS)	1D + 0.6Wind _{C&C+Wall}		
✓ 0.056	Deflection (IBC)	1D + 0.6Wind _{C&C+Wall}		

Applied Loads

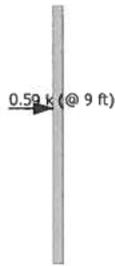
Dead [Source: Dead]



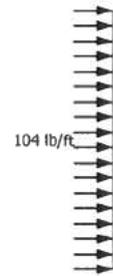
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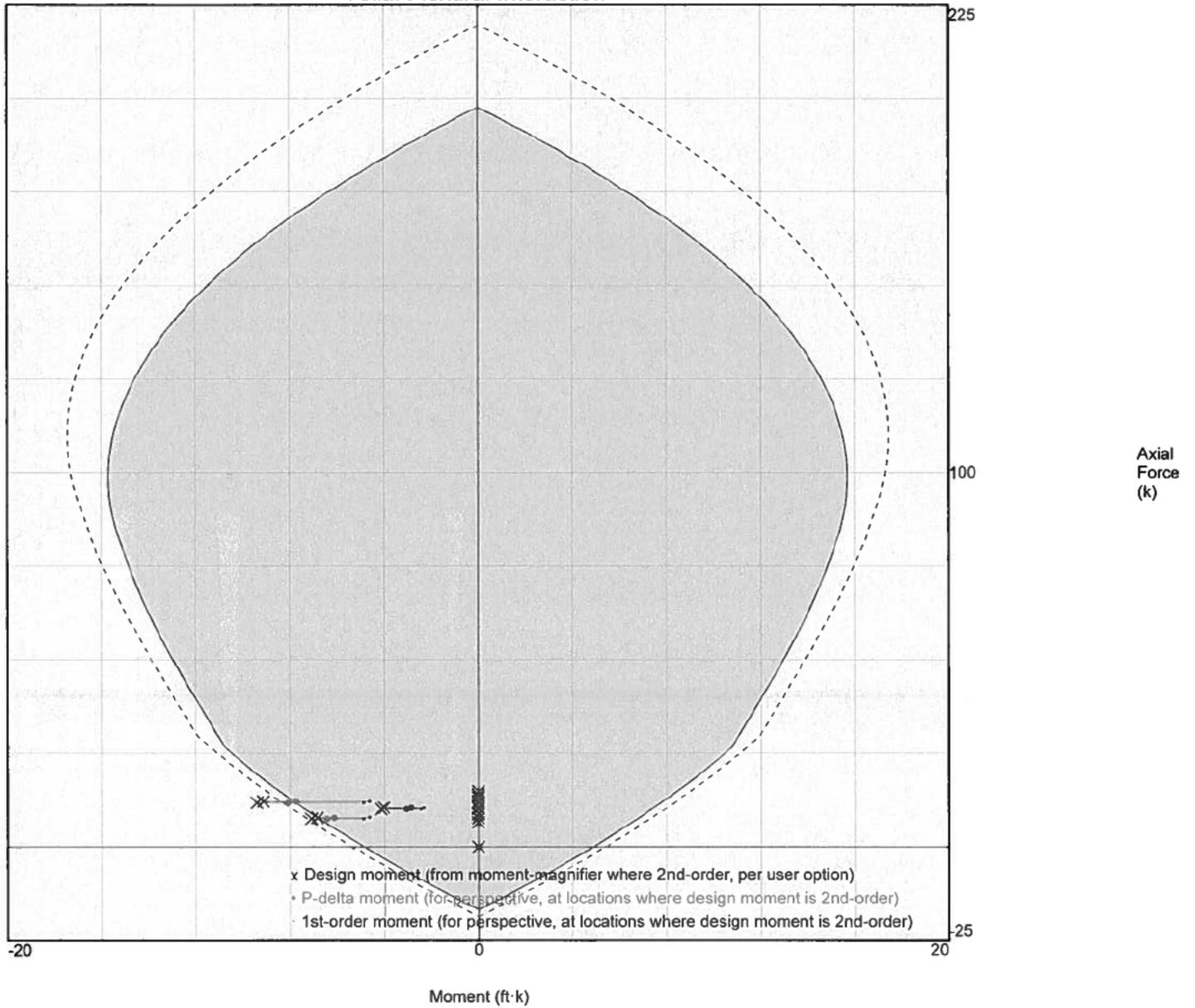
Wind [Source: WindCnCPositiveOnWall]



Wind2 [Source: WindCnCPositiveOnWall]



Axial-Flexural Interaction



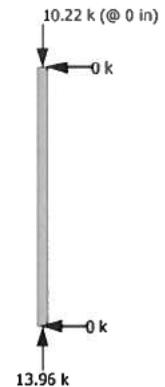
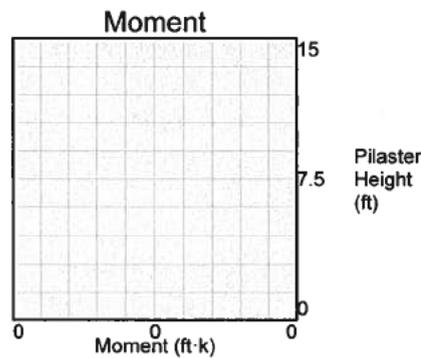
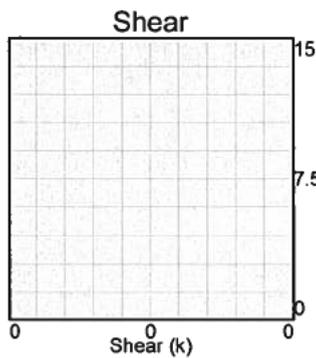
Strength Checks Design Calculations for Combination 1.4D

Load Combination: 1.4D

Pilaster weight: The unfactored weight is 2.67 k, of which 1.78 k is from the wall portion(s) and 0.89 k is from the 'column' portion.

Factored Pilaster weight = 3.74 k (total), 249.08 lb/ft (linear per height).

	Axial Force (k)	Shear (k)	Moment (ft-k)	P-δ Moment (ft-k)	Mag. Moment (ft-k)
15 ft from base	0 k	0 k	0 ft-k	0 ft-k	0 ft-k
	10.22 k	0 k	0 ft-k	N/A	N/A
7.42 ft from base	12.11 k	0 k	0 ft-k	0 ft-k	0 ft-k
0 ft from base	13.96 k	0 k	0 ft-k	N/A	N/A



Secondary Moments

The strength design provisions of TMS 402 require consideration of a secondary moment. Both P-Delta and moment magnifier options are given. The P-delta approach may be used under some conditions and the moment magnifier may be used under all conditions. Both are calculated here for perspective, but per user option the value used for design will be that from the moment magnifier approach.

Cracking Moment

The distance from the neutral axis to the extreme tension fiber 'c' is 3.81 in. This distance is calculated based on the effective cross section of the full wall. Note that because the masonry units used in this wall are not symmetrical, this value will be different when the moment is in the opposite direction such that the tension and compression faces are reversed.

$$S_n = I_n / c_t = (886.64 \text{ in}^4) / (3.81 \text{ in}) = 232.56 \text{ in}^3$$

$$f_r = 89.27 \text{ psi}$$

$$M_{cr} = S_n f_r = (232.56 \text{ in}^3) (89.27 \text{ psi}) = 1.73 \text{ ft-k}$$

[NCMA TEK note 14-4B (2008), p. 3]

Secondary Moment Calculation (At Midspan)

Note that the I_n value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \frac{(0 \text{ ft-k})}{1 - \frac{(5(12.11 \text{ k})(15 \text{ ft})^2)}{(48(1,350,000 \text{ psi})(886.64 \text{ in}^4))}} = 0 \text{ ft-k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(886.64 \text{ in}^4) = 664.98 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi}) (664.98 \text{ in}^4))}{(15 \text{ ft})^2} \right) = 273.46 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \frac{1}{1 - \left(\frac{(12.11 \text{ k})}{(273.46 \text{ k})} \right)} = 1.0463$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1.0463)(0 \text{ ft}\cdot\text{k}) = 0 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Secondary Moment Calculation (At Max Moment Location)

Note that the I_n value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \left(\frac{(0 \text{ ft}\cdot\text{k})}{1 - \frac{(5(0 \text{ k})(15 \text{ ft})^2)}{(48(1,350,000 \text{ psi})(886.64 \text{ in}^4))}} \right) = 0 \text{ ft}\cdot\text{k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(886.64 \text{ in}^4) = 664.98 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi}) (664.98 \text{ in}^4))}{(15 \text{ ft})^2} \right) = 273.46 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \frac{1}{1 - \left(\frac{(0 \text{ k})}{(273.46 \text{ k})} \right)} = 1$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1)(0 \text{ ft}\cdot\text{k}) = 0 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Strength Checks: Combination 1.4D

Design forces for this load combination:

- P_u @ top = 10.22 k
- P_u @ mid-span = 12.11 k
- P_u @ base = 13.96 k
- P_u @ max moment = 0 k
- M_u @ top = 0 ft·k
- M_u @ mid-span = 0 ft·k
- Max M_u = 0 ft·k
- Max V_u = 0 k

Axial Stress Check

$$\alpha_p = P_u / A_g = (13.96 \text{ k}) / (1.27 \text{ ft}^2) = 76.26 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

$$\sigma_{pmax} = 0.2f_m = 0.2(1,500 \text{ psi}) = 300 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

✓ $\sigma_{pmax} \geq \sigma_p$...utilization ratio 0.254

Axial Stress Check with Slender Wall

$$\frac{h}{t} = \frac{(15 \text{ ft})}{(99,999,999 \text{ in})} = 0$$

[TMS 402-16 Section 9.3.5.4.2]

Slenderness ratio does not exceed 30; check does not apply

✓ $\sigma_{pmax} \geq \sigma_p$...utilization ratio 0

Axial Force Check

$$\frac{h}{r} = \frac{(15 \text{ ft})}{(2.2 \text{ in})} = 81.78$$

[TMS 402-16 Section 9.3.4.1.1]

$$P_n = 0.80 \left[0.80P_m (A_n - A_{st}) + f_y A_{st} \right] \left[1 - \left(\frac{h}{140r} \right)^2 \right] = 0.80 (0.80(1,500 \text{ psi})((1.27 \text{ ft}^2) - (0 \text{ in}^2)) + (60,000 \text{ psi})(0 \text{ in}^2)) \left(1 - \left(\frac{(15 \text{ ft})}{(140(2.2 \text{ in}))} \right)^2 \right) = 115.74 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.1, Equation 9-15]

$$\phi P_n = \phi P_n = (0.9)(115.74 \text{ k}) = 104.17 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.4]

$$P_u = 12.11 \text{ k}$$

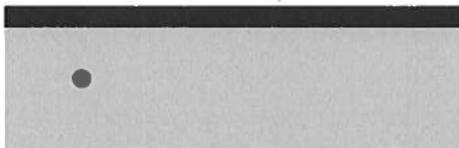
✓ $\phi P_n \geq P_u$...utilization ratio 0.116

Moment Check @ Axial Load Application Point

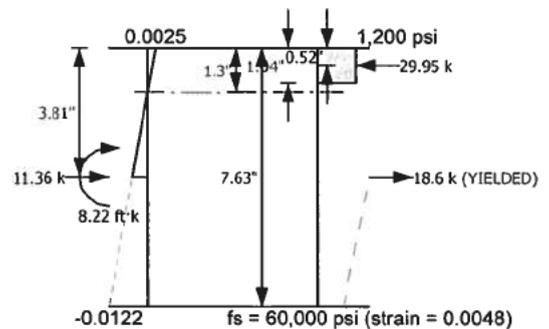
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (10.22 k) and thus $P_n = 11.36 \text{ k}$. The associated M_n (8.22 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 10.22 k, the interaction diagram gives a moment capacity of -7.4 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.17 ft². Depth of compression zone = 1.04 in.



Moment Check @ Mid-span

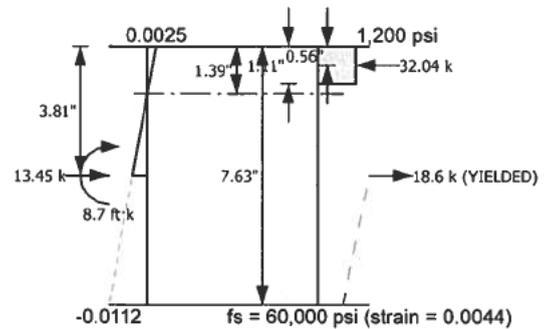
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (12.11 k) and thus $P_n = 13.45$ k. The associated M_n (8.7 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 12.11 k, the interaction diagram gives a moment capacity of -7.83 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.19 ft². Depth of compression zone = 1.11 in.



Moment Check @ Max Moment

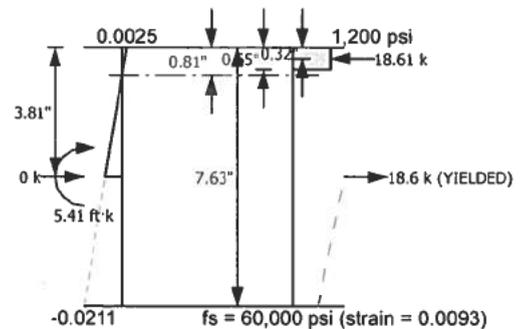
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (0 k) and thus $P_n = 0$ k. The associated M_n (5.41 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 0 k, the interaction diagram gives a moment capacity of -4.87 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.11 ft². Depth of compression zone = 0.65 in.



Shear Check

There is no applied shear force in this load combination; check passes.

✓ $\phi V_n \geq V_u$...utilization ratio 0

Maximum Reinforcement Check

Maximum reinforcement is checked based on the provisions of TMS 402-16 9.3.3.2.

$$\frac{M_u}{V_u d_v} = \frac{(0 \text{ ft}\cdot\text{k})}{((0 \text{ k})(99,999,999 \text{ in}))} = 0$$

[TMS 402-16 Section 9.3.4.1.2]

Because this ratio does not exceed 1.0 and $R \leq 1.5$, there is no limit on flexural tensile reinforcement.

✓ $\epsilon_s \geq \epsilon_{min}$...utilization ratio 0

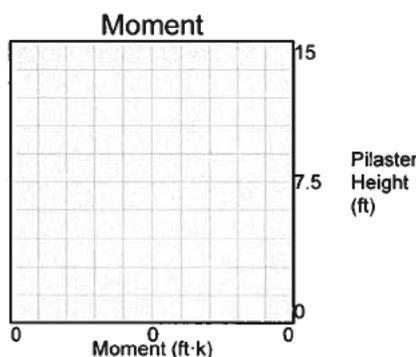
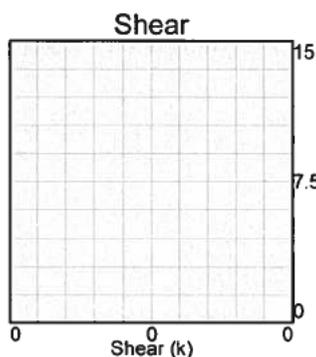
Strength Checks Design Calculations for Combination 1.2D + 1.6L

Load Combination: 1.2D + 1.6L

Pilaster weight: The unfactored weight is 2.67 k, of which 1.78 k is from the wall portion(s) and 0.89 k is from the 'column' portion.

Factored Pilaster weight = 3.2 k (total), 213.5 lb/ft (linear per height).

	Axial Force (k)	Shear (k)	Moment (ft-k)	P-δ Moment (ft-k)	Mag. Moment (ft-k)
15 ft from base	0 k	0 k	0 ft-k	0 ft-k	0 ft-k
	11.58 k	0 k	0 ft-k	N/A	N/A
7.42 ft from base	13.19 k	0 k	0 ft-k	0 ft-k	0 ft-k
0 ft from base	14.78 k	0 k	0 ft-k	N/A	N/A



Secondary Moments

The strength design provisions of TMS 402 require consideration of a secondary moment. Both P-Delta and moment magnifier options are given. The P-delta approach may be used under some conditions and the moment magnifier may be used under all conditions. Both are calculated here for perspective, but per user option the value used for design will be that from the moment magnifier approach.

Cracking Moment

The distance from the neutral axis to the extreme tension fiber 'c' is 3.81 in. This distance is calculated based on the effective cross section of the full wall. Note that because the masonry units used in this wall are not symmetrical, this value will be different when the moment is in the opposite direction such that the tension and compression faces are reversed.

$$S_n = I_n / c_t = (886.64 \text{ in}^4) / (3.81 \text{ in}) = 232.56 \text{ in}^3$$

$$f_r = 89.27 \text{ psi}$$

$$M_{cr} = S_n f_r = (232.56 \text{ in}^3)(89.27 \text{ psi}) = 1.73 \text{ ft-k}$$

[NCMA TEK note 14-4B (2008), p. 3]

Secondary Moment Calculation (At Midspan)

Note that the I_{cr} value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \frac{(0 \text{ ft-k})}{1 - \frac{5(13.19 \text{ k})(15 \text{ ft})^2}{48(1,350,000 \text{ psi})(886.64 \text{ in}^4)}} = 0 \text{ ft-k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(886.64 \text{ in}^4) = 664.98 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi}) (664.98 \text{ in}^4))}{(15 \text{ ft})^2} \right) = 273.46 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \left(\frac{1}{1 - \left(\frac{(13.19 \text{ k})}{(273.46 \text{ k})} \right)} \right) = 1.0507$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1.0507)(0 \text{ ft}\cdot\text{k}) = 0 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Secondary Moment Calculation (At Max Moment Location)

Note that the I_r value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} < M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1}}{1 - \frac{5P_u h^2}{48E_m I_n}} = \left(\frac{((0 \text{ ft}\cdot\text{k}))}{\left(1 - \frac{(5(0 \text{ k})(15 \text{ ft})^2)}{(48(1,350,000 \text{ psi})(886.64 \text{ in}^4))} \right)} \right) = 0 \text{ ft}\cdot\text{k}$$

[Masonry Designers Guide 2016, Eqn 12.4-23 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} < M_{cr}$, $I_{eff} = 0.75I_n = 0.75(886.64 \text{ in}^4) = 664.98 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi}) (664.98 \text{ in}^4))}{(15 \text{ ft})^2} \right) = 273.46 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \left(\frac{1}{1 - \left(\frac{(0 \text{ k})}{(273.46 \text{ k})} \right)} \right) = 1$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1)(0 \text{ ft}\cdot\text{k}) = 0 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Strength Checks: Combination 1.2D + 1.6L

Design forces for this load combination:

- P_u @ top = 11.58 k
- P_u @ mid-span = 13.19 k
- P_u @ base = 14.78 k
- P_u @ max moment = 0 k
- M_u @ top = 0 ft·k
- M_u @ mid-span = 0 ft·k
- Max M_u = 0 ft·k
- Max V_u = 0 k

Axial Stress Check

$$\alpha_p = P_u / A_g = (14.78 \text{ k}) / (1.27 \text{ ft}^2) = 80.76 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

$$\sigma_{pmax} = 0.2f_m = 0.2(1,500 \text{ psi}) = 300 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

✓ $\sigma_{pmax} \geq \sigma_P$...utilization ratio 0.269

Axial Stress Check with Slender Wall

$$\frac{h}{t} = \frac{(15 \text{ ft})}{(99,999,999 \text{ in})} = 0$$

[TMS 402-16 Section 9.3.5.4.2]

Slenderness ratio does not exceed 30; check does not apply

✓ $\sigma_{pmax} \geq \sigma_P$...utilization ratio 0

Axial Force Check

$$\frac{h}{r} = \frac{(15 \text{ ft})}{(2.2 \text{ in})} = 81.78$$

[TMS 402-16 Section 9.3.4.1.1]

$$P_n = 0.80 \left[0.80f_m (A_n - A_{st}) + f_y A_{st} \right] \left[1 - \left(\frac{h}{140r} \right)^2 \right] = 0.80(0.80(1,500 \text{ psi})((1.27 \text{ ft}^2) - (0 \text{ in}^2)) + (60,000 \text{ psi})(0 \text{ in}^2)) \left(1 - \left(\frac{(15 \text{ ft})}{(140(2.2 \text{ in}))} \right)^2 \right) = 115.74 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.1, Equation 9-15]

$$\phi P_n = \phi P_n = (0.9)(115.74 \text{ k}) = 104.17 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.4]

$$P_u = 13.19 \text{ k}$$

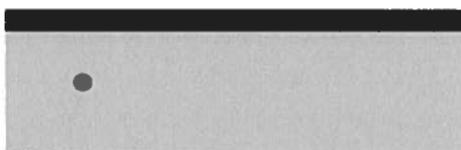
✓ $\phi P_n \geq P_u$...utilization ratio 0.127

Moment Check @ Axial Load Application Point

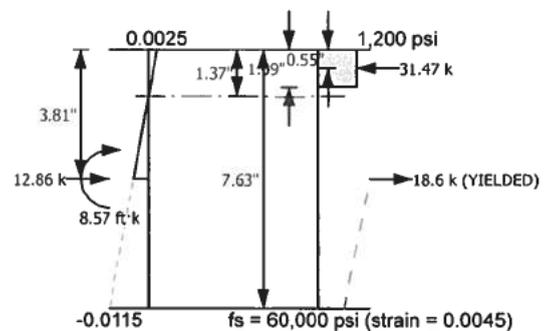
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (11.58 k) and thus $P_n = 12.86 \text{ k}$. The associated M_n (8.57 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 11.58 k, the interaction diagram gives a moment capacity of -7.71 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.18 ft². Depth of compression zone = 1.09 in.



Moment Check @ Mid-span

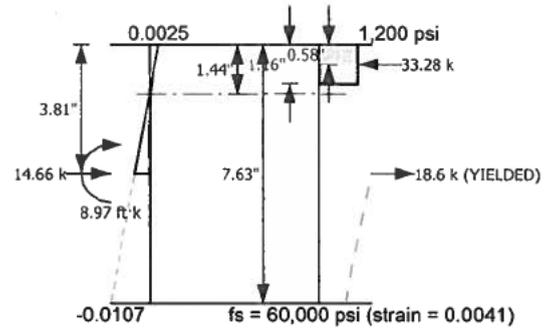
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (13.19 k) and thus $P_n = 14.66$ k. The associated M_n (8.97 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 13.19 k, the interaction diagram gives a moment capacity of -8.07 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.19 ft². Depth of compression zone = 1.16 in.



Moment Check @ Max Moment

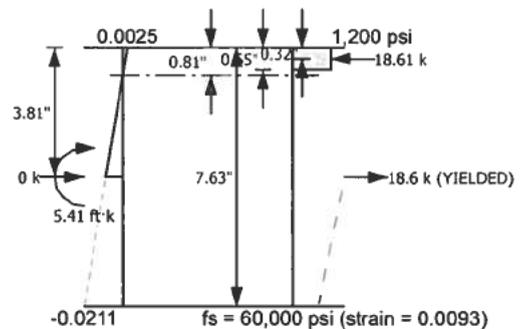
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (0 k) and thus $P_n = 0$ k. The associated M_n (5.41 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 0 k, the interaction diagram gives a moment capacity of -4.87 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.11 ft². Depth of compression zone = 0.65 in.



Shear Check

There is no applied shear force in this load combination; check passes.

✓ $\phi V_n \geq V_u$...utilization ratio 0

Maximum Reinforcement Check

Maximum reinforcement is checked based on the provisions of TMS 402-16 9.3.3.2.

$$\frac{M_u}{V_u d_v} = \frac{(0 \text{ ft}\cdot\text{k})}{((0 \text{ k})(99,999,999 \text{ in}))} = 0$$

[TMS 402-16 Section 9.3.4.1.2]

Because this ratio does not exceed 1.0 and $R \leq 1.5$, there is no limit on flexural tensile reinforcement.

✓ $\epsilon_s \geq \epsilon_{min}$...utilization ratio 0

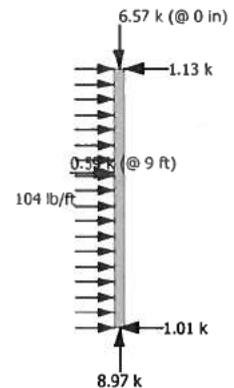
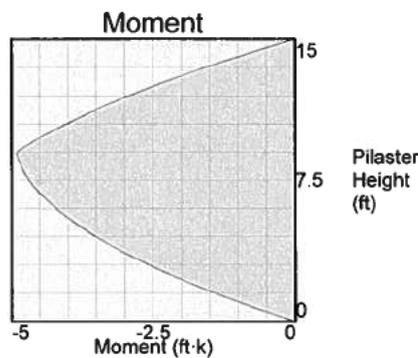
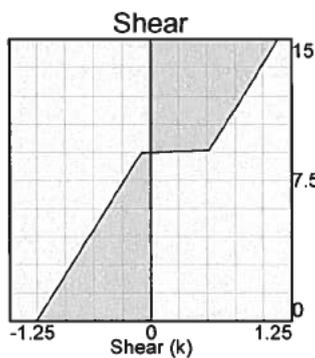
Strength Checks Design Calculations for Combination 0.9D + 1Wind_{C&C+Wall}

Load Combination: 0.9D + 1Wind_{C&C+Wall}

Pilaster weight: The unfactored weight is 2.67 k, of which 1.78 k is from the wall portion(s) and 0.89 k is from the 'column' portion.

Factored Pilaster weight = 2.4 k (total), 160.13 lb/ft (linear per height).

	Axial Force (k)	Shear (k)	Moment (ft-k)	P-δ Moment (ft-k)	Mag. Moment (ft-k)
15 ft from base	6.57 k	1.13 k	0 ft-k	N/A	N/A
8.94 ft from base	7.54 k	-0.08 k	-4.91 ft-k	-6.45 ft-k	-7.11 ft-k
7.42 ft from base	7.78 k	-0.24 k	-4.66 ft-k	-6.15 ft-k	-6.85 ft-k
0 ft from base	8.97 k	-1.01 k	0 ft-k	N/A	N/A



Secondary Moments

The strength design provisions of TMS 402 require consideration of a secondary moment. Both P-Delta and moment magnifier options are given. The P-delta approach may be used under some conditions and the moment magnifier may be used under all conditions. Both are calculated here for perspective, but per user option the value used for design will be that from the moment magnifier approach.

Cracking Moment

The distance from the neutral axis to the extreme tension fiber 'c' is 3.81 in. This distance is calculated based on the effective cross section of the full wall. Note that because the masonry units used in this wall are not symmetrical, this value will be different when the moment is in the opposite direction such that the tension and compression faces are reversed.

$$S_n = I_n / c_t = (886.64 \text{ in}^4) / (3.81 \text{ in}) = 232.56 \text{ in}^3$$

$$f_r = 89.27 \text{ psi}$$

$$M_{cr} = S_n f_r = (232.56 \text{ in}^3)(89.27 \text{ psi}) = 1.73 \text{ ft-k}$$

[NCMA TEK note 14-4B (2008), p. 3]

Secondary Moment Calculation (At Midspan)

Note that the I_{cr} value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} \geq M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1} + \frac{5M_{cr}P_u h^2}{48E_m} \left(\frac{1}{I_n} - \frac{1}{I_{cr}} \right)}{1 - \frac{5P_u h^2}{48E_m I_{cr}}} = \frac{(-4.66 \text{ ft-k}) + \left(\frac{5(-1.73 \text{ ft-k})(7.78 \text{ k})(15 \text{ ft})^2}{48(1,350,000 \text{ psi})} \right) \left(\left(\frac{1}{886.64 \text{ in}^4} \right) - \left(\frac{1}{59.2 \text{ in}^4} \right) \right)}{1 - \left(\frac{5(7.78 \text{ k})(15 \text{ ft})^2}{48(1,350,000 \text{ psi})(59.2 \text{ in}^4)} \right)} = -6.15 \text{ ft-k}$$

[Masonry Designers Guide 2016, Eqn 12.4-25 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} \geq M_{cr}$, $I_{eff} = I_{cr} = 59.2 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi}) (59.2 \text{ in}^4))}{(15 \text{ ft})^2} \right) = 24.35 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \frac{1}{1 - \left(\frac{(7.78 \text{ k})}{(24.35 \text{ k})} \right)} = 1.4699$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1.4699)(-4.66 \text{ ft}\cdot\text{k}) = -6.85 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Secondary Moment Calculation (At Max Moment Location)

Note that the I_c value used here is interpolated from the values in the Section Analysis calculations based on the axial force at this location.

P-Delta

Because $M_{u1} > M_{cr}$, M_u is calculated by:

$$M_u = \frac{M_{u1} + \frac{5M_{cr}P_u h^2}{48E_m} \left(\frac{1}{I_n} - \frac{1}{I_{cr}} \right)}{1 - \frac{5P_u h^2}{48E_m I_{cr}}} = \left((-4.91 \text{ ft}\cdot\text{k}) + \left(\frac{(5(-1.73 \text{ ft}\cdot\text{k})(7.54 \text{ k})(15 \text{ ft})^2)}{(48(1,350,000 \text{ psi}))} \right) \left(\left(\frac{1}{(886.64 \text{ in}^4)} \right) - \left(\frac{1}{(59.22 \text{ in}^4)} \right) \right) \right) = -6.45 \text{ ft}\cdot\text{k}$$

[Masonry Designers Guide 2016, Eqn 12.4-25 (substituting M_{u1} for first order moment expression)]

Moment Magnifier

Because $M_{u1} > M_{cr}$, $I_{eff} = I_{cr} = 59.22 \text{ in}^4$.

$$P_e = \frac{\pi^2 E_m I_{eff}}{h^2} = \left(\frac{((3.1416)^2 (1,350,000 \text{ psi}) (59.22 \text{ in}^4))}{(15 \text{ ft})^2} \right) = 24.35 \text{ k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-29]

$$\psi = \frac{1}{1 - \frac{P_u}{P_e}} = \frac{1}{1 - \left(\frac{(7.54 \text{ k})}{(24.35 \text{ k})} \right)} = 1.4485$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-28]

$$M_u = \psi M_{u0} = (1.4485)(-4.91 \text{ ft}\cdot\text{k}) = -7.11 \text{ ft}\cdot\text{k}$$

[TMS 402-16 Section 9.3.5.4.3, Equation 9-27]

Strength Checks: Combination 0.9D + 1Wind_{C&C+W_{all}}

Design forces for this load combination:

- P_u @ top = 6.57 k
- P_u @ mid-span = 7.78 k
- P_u @ base = 8.97 k
- P_u @ max moment = 7.54 k
- M_u @ top = 0 ft·k
- M_u @ mid-span = -4.66 ft·k
- Max M_u = -4.91 ft·k
- Max V_u = 1.13 k

Axial Stress Check

$$\sigma_p = P_u / A_g = (8.97 \text{ k}) / (1.27 \text{ ft}^2) = 49.03 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

$$\sigma_{pmax} = 0.2f'_m = 0.2(1,500 \text{ psi}) = 300 \text{ psi}$$

[TMS 402-16 Section 9.3.5.4.2]

✓ $\sigma_{pmax} \geq \sigma_p$...utilization ratio 0.163

Axial Stress Check with Slender Wall

$$\frac{h}{t} = \frac{(15 \text{ ft})}{(99,999,999 \text{ in})} = 0$$

[TMS 402-16 Section 9.3.5.4.2]

Slenderness ratio does not exceed 30; check does not apply

✓ $\sigma_{pmax} \geq \sigma_p$...utilization ratio 0

Axial Force Check

$$\frac{h}{r} = \frac{(15 \text{ ft})}{(2.2 \text{ in})} = 81.78$$

[TMS 402-16 Section 9.3.4.1.1]

$$P_n = 0.80 \left[0.80 P_m (A_n - A_{st}) + f_y A_{st} \right] \left[1 - \left(\frac{h}{140r} \right)^2 \right] = 0.80 (0.80 (1,500 \text{ psi}) ((1.27 \text{ ft}^2) - (0 \text{ in}^2)) + (60,000 \text{ psi})(0 \text{ in}^2)) \left(1 - \left(\frac{(15 \text{ ft})}{(140(2.2 \text{ in}))} \right)^2 \right) = 115.74 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.1, Equation 9-15]

$$\phi P_n = \phi P_n = (0.9)(115.74 \text{ k}) = 104.17 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.4]

$$P_u = 7.78 \text{ k}$$

✓ $\phi P_n \geq P_u$...utilization ratio 0.075

Moment Check @ Axial Load Application Point

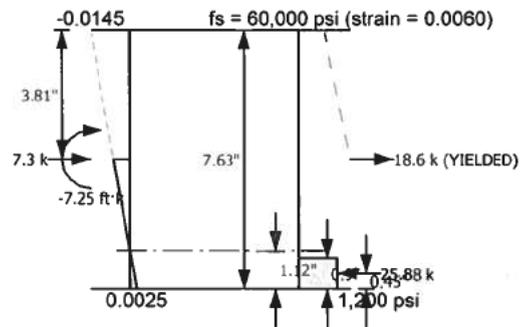
The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (6.57 k) and thus $P_n = 7.3 \text{ k}$. The associated M_n (-7.25 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 6.57 k, the interaction diagram gives a moment capacity of 6.53 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0



Compression area = 0.15 ft². Depth of compression zone = 0.9 in.



Moment Check @ Mid-span

The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (7.78 k) and thus $P_n = 8.65$ k. The associated M_n (-7.58 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 7.78 k, the interaction diagram gives a moment capacity of 6.83 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0.683



Compression area = 0.16 ft². Depth of compression zone = 0.95 in.

Moment Check @ Max Moment

The moment capacity is determined by considering the point on the interaction diagram where ϕP_n is equal to the axial load at this section (7.54 k) and thus $P_n = 8.38$ k. The associated M_n (-7.52 ft·k) is multiplied by ϕ to obtain design moment capacity.

At an axial load of 7.54 k, the interaction diagram gives a moment capacity of 6.76 ft·k.

✓ $\phi M_n \geq M_u$...utilization ratio 0.726



Compression area = 0.16 ft². Depth of compression zone = 0.94 in.

Shear Check

The net shear area is taken as the area bounded by the extreme compression fiber and the reinforcement nearest the opposite face, subject to the effective compression width.

$$A_{nv} = 0.64 \text{ ft}^2$$

Shear capacity due to masonry, conservatively taking $M/Vd = 1.0$:

$$V_{nm} = [4.0 - 1.75(1.0)] A_{nv} \sqrt{f'_m} + 0.25 P_u = (4.0 - 1.75(1.0))(0.64 \text{ ft}^2) \sqrt{(1,500 \text{ psi})} + 0.25(8.97 \text{ k}) = 10.22 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.2.1, Equation 9-20 with conservative $M/Vd = 1.0$]

No reinforcement for out-of-plane shear:

$$V_{ns} = 0 \text{ k}$$

Nominal shear capacity:

$$V_n = (V_{nm} + V_{ns}) \gamma_g = ((10.22 \text{ k}) + (0 \text{ k}))(1) = 10.22 \text{ k}$$

[TMS 402-16 Section 9.3.4.1.2, Equation 9-17]

Limited by:

$$V_{n_limit} = (4 A_{nv} \sqrt{f'_m}) \gamma_g = (4(0.64 \text{ ft}^2) \sqrt{(1,500 \text{ psi})})(1) = 14.18 \text{ k}$$

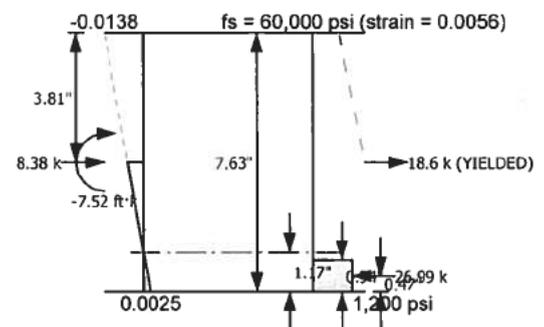
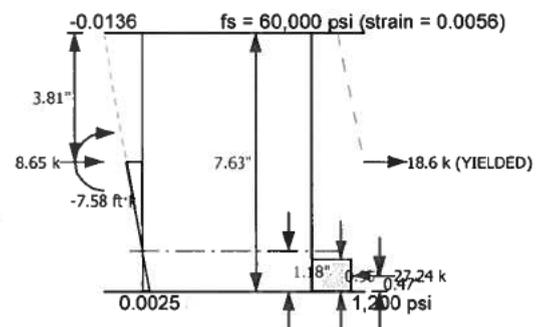
[TMS 402-16 Section 9.3.4.1.2, Equation 9-19]

...Limit on V_n does not control

$$\phi V_n = \phi V_u = (0.8)(10.22 \text{ k}) = 8.17 \text{ k}$$

[TMS 402-16 Section 9.1.3, 9.1.4.5]

✓ $\phi V_n \geq V_u$...utilization ratio 0.124



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Maximum Reinforcement Check

Maximum reinforcement is checked based on the provisions of TMS 402-16 9.3.3.2.

$$\frac{M_u}{V_u d_v} = \frac{(4.91 \text{ ft}\cdot\text{k})}{((0.08 \text{ k})(99,999,999 \text{ in}))} = 0$$

[TMS 402-16 Section 9.3.4.1.2]

Because this ratio does not exceed 1.0 and $R \leq 1.5$, there is no limit on flexural tensile reinforcement.

✓ $\epsilon_s \geq \epsilon_{min}$...utilization ratio 0

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